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NAVAL POSTGRADUATE SCHOOL Monterey, California



THESIS

SEAKEEPING ANALYSIS OF SMALL DISPLACEMENT HIGH-SPEED VESSELS

by

Sarah E. Rollings

March 2003

Thesis Advisor:

Fotis Papoulias

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SEAKEEPING ANALYSIS OF SMALL DISPLACEMENT HIGH-SPEED VESSELS

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Submitted in partial fulfillment of the requirements for the degree of

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ABSTRACT

Today, high speed vessels are the focus of many ship designers in both the military and commercial maritime industries. High speed vessels provide versatility in accomplishing a range of missions. In developing designs for high speed vessels, the engineer must account for the response of the ship in the environment while operating at mission essential speeds. Much of the design and simulations are computer automated and rely on background data that comes from existing ship designs or experimental results. Series 64, which was developed by the David Taylor Model Basin, now known as Naval Surface Warfare Center, Carderock Division, sets the benchmark for resistance data commonly used by naval architects. To expand upon this well known series, this research attempts to develop seakeeping data trends for scaled-up Series 64 models. The scale used is based on a small displacement ship of 2500 tons. The results of the research can then be used by engineers in application to the design of small displacement, high speed ships.

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I would like specifically to thank the following people for making this thesis possible:

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My boyfriend Chris, who understands me at all times and with which I share many experiences, exciting and challenging. For his love and support through the challenging times and for his shared happiness during the exciting times.

My best friend Monica. For her continued caring support and for always being there for me.

I. INTRODUCTION

A. BACKGROUND

As the Navy adjusts to the global climates and the necessary demands, the missions and associated equipment must also transition. The Navy has always produced large war fighting ships as the backbone for naval missions, and history shows the success of utilizing these ships in naval tactics. As the Navy goes into the future, the need for smaller, faster, multi-mission ships grows exponentially. Using smaller ships that travel at higher speeds allows for the necessary versatility needed in a multi-threat theater. This versatility also lends itself to the commercial marine industry. Much of the work in research and development of high speed vessels (HSV's) has taken place in the commercial industry. The trend in the commercial realm has followed the path of increasing transit speeds, but with that, increasing ship size. Finding a meeting ground between the needs of the Navy and an optimal small displacement, high speed platform is a challenge for today's Naval leaders.

B. SCOPE OF THIS WORK

There have been successfully built and tested high speed vessels, promising designs and prototypes, but also limiting factors. Many of these limiting factors come from design constraints, which could be anything from technology to materials. Another design and production factor, which is vital and sometimes becomes the limiting factor in the design process, specifically when establishing the "weighted sum," is the economic considerations. HSV's must combine high speed hull forms, lightweight structures and compact power plants while keeping the ship cost effective. HSV's have seen tremendous success in the commercial realm, particularly with ferries. The mission of ferries lends it to optimize on the current technology available for designing and constructing HSV's. Ferries are usually used for relatively short distances for personnel transport, allowing for lower payloads and the ability to use lighter materials such as aluminum.

Naval architects continue to expand the use of computer aided design in building both commercial and military vessels. These designs rely on data from scaled models, prototypes and full-scale ships. It is possible to build a ship in the computer, but how will it handle at sea? The seakeeping ability of a ship is as important to the completion of a luxury voyage as

it is to the success of a military mission. Seakeeping is the dynamic response of a ship in the water. Knowing the seakeeping abilities of vessels during the early stages of design would enhance the design process, simulations and final product. There are programs that will estimate the seakeeping ability of a ship, but these programs are for specific ship designs. One specific computer program, SHIPMO, takes the ship's characteristics and the wave input to produce data and graphs on the expected wave reactions with the hull for the specified conditions. This program is based on standard strip theory assumptions and is very closely related to the Navy's standard ship motions prediction (SMP) program. This specific computer program and its abilities give rise to the motivation for this thesis.

The focus of this thesis is two-fold:

1. Compile and compare existing HSV designs and information, while extracting useful technology and information for military applications.

2. Correlate seakeeping trends in computer scaled models for small displacement vessels based on changing environmental conditions.

Chapter II will focus on developing a technical tutorial on the state of high speed vessels and associated technology in the commercial and military realm. Chapter III will look at the trends of a seakeeping analysis of six computer-scaled Series 64 models. The scaling will be based on a 2500 ton small displacement ship. Finally, Chapter IV will present the conclusions of this thesis and recommendation for further research.

II. HIGH SPEED VESSELS YESTERDAY AND TODAY

A. DEFINING AND DEVELOPING HIGH SPEED VESSELS

High speed vessels are considered to be vessels that can travel at a sustained speed equal to or greater than 35 knots with bursts of high speeds of 40-60 knots. Two interpretations of high speed bursts are provided by Navy Warfare Development Command, NWDC, and are mission oriented definitions. One is defined as a twenty-one day patrol, averaging 8 knots with 20-minute 55 knot bursts, and the second is a five day mission intense period, averaging 10 knots with 15-minute 55 knot bursts. Speed aids military ships both operationally and tactically. HSV's are also fully accepted and available in the commercial maritime role. High speed ferries are a prime example of this, with some, like the ferry made by the Australian company INCAT, traveling at 45 knot transit speeds.

Factors that can combine to give rise to HSV designs include propulsion, payload, mission, range and building material. As many designers know, some of these factors will be sacrificed to enable the rest to succeed. High speeds means using compact power units and lightweight structural materials while maintaining a low displacement, which gives rise to the necessity of smaller payloads with less fuel capacity. Although this is a typical overview of design priorities, the key design factor could be and many times is the ship's mission. For example, commercial ferries can be made of lightweight aluminum for max speed, but aluminum may not be a feasible material for military use due to the need for high combat survivability.

To start incorporating the design factors into the design process, several comparison tools for vessels have been developed. These comparison tools are non-dimensional relationships that are applied to a given design. Naval architects use four main non-dimensional numbers. They are Transport Efficiency (TE), Lift to Drag ratio (L/D), Transport Factor (TF), [1], and Froude number (Fr). The associated equations are presented on the next page.

$T F = \frac{K * W}{\frac{S H P_{TI}}{W}}$	K: non-dimensionalized constant W: full-load weight
- K * W	V_{K} : design speed V: max speed
$T E = \frac{S H P_{TI}}{S H P_{TI}}$	SHP_{TI} : total installed power for propulsion
$F r = \frac{V}{\sqrt{g L}}$	V: ship's speed L: ship's length g: gravity

TE relates the sum of a vessel's installed propulsion and auxiliary power to the vessel's weight and maximum speed, where TF uses design speed. These two factors are then unique to each ship, but the L/D ratio changes with speed for the individual vessels. The trend of TF for vessels, as developed by Kennell [1], is that high values of TF, TE and L/D are considered to represent good design characteristics which imply increased cargo carrying capacity, increased speed and reduced power. The TF values for the 30-70 knot range fell below 20, with those near 20 being considered optimal designs. The data presented by Kennell indicates that there were not any monohulls in the 50-70 knot range that achieved a TF value near 20. Small displacement monohulls have achieved comparable speeds, but resulted in low TF values of 1-2 because of their limited range and/or cargo capabilities. When looking at designing and building high speed ships, calculating a TF value can help for comparison and also as a benchmark goal when specifically looking at designing small displacement HSV's. In fact, NSWC, Carderock Division developed a list of ships and designs and calculated the TF for each, [2]. Froude number allows for another way to hydrodynamically classify ships. Naval architects use the Froude number when vessels deal with the interaction of the water's free surface and the hull. High speed vessels are typically defined with Fr > 0.4.

A ship design may meet all the specified requirements, but the technology may not be developed to support the design. One area that is expected to favorably impact the development of HSV's is the improvement of propulsion systems. This impact is significant because it is commonly referenced that a ship's power requirement is proportional to the increase in ship's speed cubed, i.e. to double the ship's speed means multiplying the ship's installed power requirements by eight. Table 1 shows a break down of propulsion

technology compared to near-term and far-term technology in relation to payload, speed and range, as presented by Kennell, Lavis and Templeman, [3].

	Power (hp)	Range (nm)	Speed (knots)	Payload (stons)
Today				
LMSR design		12,000	24	19,000
TAKR design		12,000	27	12,000
Near-Term [3]	Gas Turbines	500	40	35,000
	70,000		60	7,500
	Diesel Engines	10,000	40	54,000
	40,000-50,000		60	15,000
Far-Term [3]	Gas Turbines	500	40	12,000
	125,000		60	n.a.
	Diesel Engines*	10,000	40	19,000
	80,000-100,000		60	2,000

*Government funded prediction (commercial evolution prediction: 60-70 khp).

Far and Near Term Propulsion Technology <u>Table 1</u>

Other important propulsion advancements are happening in electric drive, fuel cells, and water jet propulsors. The Navy has funded extensive research in electric drive with Electric Boat, General Dynamics taking the lead on developing the propulsion for the Navy's future destroyer. Fuel cells can be used for ships' power and/or propulsion. The Office of Naval Research (ONR) is planning to have a 625 kW molten carbonate demonstration unit ready by 2004 that is being build by FuelCell Energy. This unit will convert NATO F-76 distillate fuel to 450vac, 3-phase power and be primarily used for ship's installed power, [4]. Siemens-Westinghouse plans to build a 320 kW and 1000 kW commercial solid oxide fuel cell (SOFC) for propulsion use, [4]. Unfortunately, fuel cell technology still would fall under far-term technology for use in the marine environment. Water jet propulsion is also a good alternative to conventional propellers and is actively being pursued. The water jet propulsion system is usually connected to the drive shaft through an internal combustion system still, and can achieve higher overall efficiencies if the engine and jet drive are well matched. Currently, off-the-shelf waterjets are available up to 60,000 SHP which are designed for speeds of 30-60 knots, [5].

Another area that will favorably impact the future of high speed ships is advancements in light-weight hull and machinery material. Using lighter construction materials would decrease the lightship displacement to allow for more available payload or achieving a higher speed at the lighter load for the same power requirements. Aluminum is already an accepted construction material in the commercial world, reducing the weight by a quarter based on traditional steel construction. One of the problems with aluminum is the lack in technology of joining. An option that is seeing success, LASCOR, which is essentially a metal sandwich joined by laser to produce panels that surpassed the equivalent conventional plate beam during mechanical tests (tension, bending, compression). Structures have been successfully installed on the USS Mt. Whitney (LCC-20) and USS Blue Ridge (LCC-19). The future of technology for high speed ships is to find the most advantageous combination of these lightweight materials and propulsion options with an appropriate hull form.

B. SUCCESSFUL HIGH SPEED VESSELS

There are multiple options available for high speed vessels for different purposes. Classifying these based on hull form breaks down to include monohulls, catamarans, trimarans, small waterplane twin hull (SWATH) types, surface effects ships, and hydrofoils. This section will look at these available designs and illustrate the advantages and disadvantages of the high speed potential of the design. In particular, the areas of concern will be speed, displacement, pay load, propulsion and hull form. Some designs also have characteristics that are pertinent to the viability of that design, but may not be common to all designs.

1. Monohulls

Monohulls have long dominated the maritime world from shipping to military combat. Typically, monhulls have lower total resistance in operation, which is a factor of the slenderness, making them the optimal hull form from a purely hydrodynamic resistance roll, [6]. The key to slenderness is vessel length, but the optimal length for decreased resistance is much longer than normally accepted. The problem with slenderness in monohulls is that as it increases; the lateral stability decreases, making them more susceptible to large roll responses. This has given rise to designs that add roll stabilizers in the form of structural attachments, tanks or foils while maintaining the main hull's slenderness. Monohulls also

give rise to a range of propulsion choices from water jets to conventional propellers or electric drive.

The largest monohull to date is the *SS United States*, which is 940 feet, and 45,450 LT with a full load. She has a range of 10,000 nm and a sustaining speed on 35 knots. According to the *High Speed Sealift Technology Workshop* in 1997, several designs were entertained that proposed a 1500 foot ship that could achieve a sustaining speed of 50 knots, [5]. Although this may be interesting, draft and length limitation, not to mention the economic viability may limit this design to paper only. Purely looking at speed, the transatlantic speed record belongs to the 68 meter motor yacht *Destriero* which averaged 53 knots, starting with full load at 42 knots and finishing at 63 knots using three GE LM 1600 engines with three KaMeWa 125 waterjets. Monohulls will always be a consideration, especially in the US Navy, but the future to small displacement high speed ships may not lie with monohulls.

2. Catamarans

Catamarans are fast taking over as the most viable hull form for HSV's. They perform better than monohulls in minimizing wave resistance, but there is the possibility for a high amount of wave interaction between the hulls. Catamarans become stable in the ship's roll response but are more susceptible to pitch and heave responses. Some catamarans have been clocked at speeds over 45 knots. Another advantage to catamarans is the draft. If a monohull and a catamaran of equal displacement were compared, the catamarans will have a lower draft. This is an advantage for missions that require littoral proximity. The catamaran also provides greater beam. INCAT's EV10B achieves speeds of 47 knots when lightly loaded, but only 38 knots fully loaded. It uses four 7080 kW diesel engines with four waterjets and measures 97 meters with only a 3.4 meter draft. Other designs have been proposed with speeds of close to 65 knots, like INCAT's proposed design of a 130 meter catamaran with a 63 knot service speed, [5]. A disadvantage to catamarans is wave interaction between the hulls. Many of the same advantages and disadvantages can be found in trimarans.

3. Trimarans

Trimarans may possibly be able to combine the best of both worlds between monohulls and catamarans. Hydrodynamically the main hull can maintain the optimal slenderness of a monohull while achieving lateral stability with the addition of the side hulls. In fact, trimarans are better than monohulls in minimizing wave resistance by 53% and catamarans by 19%, [6]. Having the additional side hulls also increases the deck space, which could allow for more cargo carrying capabilities or even more armament. The Royal Navy already has a trimaran test ship, *Triton* that they are looking at. This ship is 95 meters with a 3 meter draft. The *Triton* was not built as a HSV and only achieves 20 knots using two 2 MW diesel generators and a single screw. Wave interaction between the hulls must also be considered for trimarans.

4. SWATH (Small Waterplane Area Twin Hulls), Semi-SWATH

SWATH type vessels use the concept of lifting bodies, generating lift from their hydrodynamic shape and from buoyancy. SWATH's have two lifting bodies attached beneath the main hull but other designs such as the Sea Slice have four. The advantage is that these vessels can easily reach high cruising speeds while maintaining low drag resistance. The Slice reduces wave making resistance 35% more than the standard SWATH design according to Lockheed Martin's web information. SWATH hull forms have higher stability over catamarans and trimarans. One disadvantage is the high stress concentration on the hull versus more conventional designs where the stress levels are more evenly distributed. The SWATH hull form can achieve speeds greater than 25 knots but require much more horsepower than other hull forms at those speeds. Lockheed Martin's Slice prototype is 104 feet with a 55 foot beam that can maintain 30 knots in waves up to 12 feet in height. The Slice hull form allows for higher speeds using the same horsepower, which may make it a viable hull form for HSV's. In 2002, the Slice was used by the Navy as a Littoral Combat Ship (LCS) option during the Navy's fleet battle experiment in San Diego.

5. Surface Effects Ship (SES)

SES's are interesting because at cruising speeds, the catamaran type main hull completely clears the water and an aircushion, which is typically made of Kevlar, supports the ship. This reduces the drag significantly over that which conventional ships experience. The Navy has been interested in an SES type ship for many years, developing a design for a DDSG (Surface Effect Guided Missile Destroyer) by the late 1970's with a prediction of a speed in excess of 90 knots in average conditions, [7]. This SES would employ either water jet propulsion or supercavitating water screw propulsion. SES's have virtually no frictional resistance and low wave resistance which gives rise to the claim of the ship with the highest TF, [6]. These values are achieved at 50 knots. Also, Bell Aerospace completed a prototype that tested to run at 100 knots in 6 foot seas and became the basis for the Navy's current landing craft air cushion vessel (LCAC). Its propulsion is lift fans, gas turbine engines and supercavitating propellers. A disadvantage of SES's is that the air cushion causes a destabilizing effect on the roll restoring moment due to the water level inside the aircushion being lower than the waterline. SESs' use less power and maintain higher speeds than a catamaran, but the speed loss in waves is more significant than catamarans, [8].

6. Hydrofoils

Hydrofoils are monohulls with structural attachments that behave like aircraft wings to lift the main hull clear of the water. Hydrofoils have large advantages at high speeds but the foil is detrimental to the ship's resistance during low to medium speeds, possibly doubling the ship's drag. Several countries have successfully built hydrofoils, which include the Navy's hydrofoil fleet of PHM's. The *USS Plainview* (AGEH 1) was 212 feet (320 LT) and reached 50 knots. The Canadian Navy's hydrofoil was 150 feet and reached between 50-60 knots. Foilcatamarans have also been developed by both Japan and Norway. Ohkusu pointed out that deck diving in following waves is a danger to catamarans, [8]. Another disadvantage is to take care to avoid cavitations that can quickly destroy the materials and machinery, destroying the lift advantages of the ship. Cavitations can limit speeds to less than 50 knots on certain designs. Lastly, hydrofoils can lose all lift effects on the backside of a wave, causing the elevated main hull to crash to the water, [5].

C. NAVAL OPTIONS AND APPLICATIONS

Many of the designs discussed in the previous section can lend themselves to military applications while many are not as useful. Choosing the priorities in military applications can be subjective. The NSWC, Carderock Division developed a qualitative analysis method that allows weighting factors to be assigned by the user to specific attributes that relate the relative importance to the mission, [3]. Using such a tool as this allows the users, who could be an engineer or politician, to prioritize and easily look and compare different options. One mission that is contributing to extensive design research is the Littoral Combatant Ship (LCS). Small displacement HSV research and development will be useful in the Navy's development of the LCS, which is scheduled to begin construction in the fiscal year 2005. The goal of the LCS is to be a multi-mission capable ship with speed and agility. The possible design limiting, mission critical parameter is that the LCS must sustain operations during periods from non-hostile transit environments to sustained combat. This is design limiting in the sense that the ship cannot necessarily take advantage of all the high speed technology that is available. An example is the usage of aluminum to reduce weight. Aluminum is not as strong as steel, which makes it less likely to survive in a combat environment. The LCS could take advantage of some of the non-traditional hull forms in the preceding section such as a trimaran or Slice hull form.

Other examples of innovative designs have been developed and presented by such groups as the Total Ship Systems Engineering (TSSE) students at the Naval Post Graduate School. This group of students from various curriculums collaborates on a design to meet the mission requirements provided by the Systems Engineering and Analysis students. In the past five years, four of the designs have incorporated multihull configurations. The missions range from an aviation platform to a maritime propositioning ship to a littoral warfare combat system. One design, the Sea Archer, used a hull form related to an SES catamaran with a threshold speed of 40 knots and a surge speed of 60 knots. Many of these designs culminated in the Crossbow project presented in December 2001.

III. SEAKEEPING ANALYSIS

A. DEFINING SEAKEEPING

As previously mentioned seakeeping is the dynamic response of the ship affected by environmental forces, primarily wind and waves. It is often a limiting factor in operability, specifically speed loss. By incorporating seakeeping into the initial ship design, the ship's performance and efficiency can improve. The primary parameters that affect seakeeping are the ship proportions, including waterplane geometry and weight distribution. Secondarily, unique hull characteristics such as transom sterns, bulbous bows or motion damping devices also affect the ship's seakeeping abilities. There are some general seakeeping design guidelines that should be always kept in mind during the design process, [9].

- Longer lengths are better for a ship's seakeeping abilities
- Wave excitation comes in through the waterplane area. Smaller waterplane area ships experience fewer motions, but also have less damping which results in pronounced resonant peaks.
- Wave excitation also comes through pressure, which reduces exponentially with depth.

The primary motions that need controlling are roll in monohulls and heave and pitch in multi-hulls.

The seakeeping response of a ship is a random process and must combine several elements. The key elements are ship characteristics, required functions for mission achievement and a specified sea environment for the given mission, [9]. Table 2 identifies the top level performance requirements for the Navy. The sea states and their conditions mentioned in Table 2 are the seakeeping requirements that the Navy demands of its ship designs. These requirements lead to defined limiting criteria for specific operations and for embarked personnel. Table 3 defines the Navy's current criteria, [9]. As a seakeeping analysis is preformed, the output usually is a speed polar grid, which maps the waves based on ship's heading with increasing speed. These plots and the design criteria, such as what is displayed in Table 3 are then used to develop operating envelopes for a specific ship design.

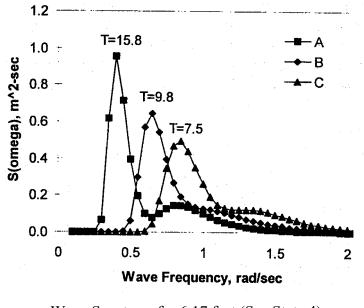
Performance Requirements	Sea State	Environmental Conditions
Operation of embarked	5	Significant wave height 10.2 feet
helicopter.		Wind velocity 20 knots
		Ship takes best heading for helicopter
Replenish and strike-down	5	Significant wave height 10.2 feet
underway.		Wind velocity 20 knots
		Ship takes best heading
Continuous efficient mission	6	Significant wave height 16.9 feet
fulfillment without significant		Wind velocity 30 knots
degradation.		All ship headings
Limited mission operation	7	Significant wave height 30.6 feet
without returning to port for		Wind velocity 44 knots
repairs after seas subsides.		Ship takes best heading
Survivability without serious	8	Significant wave height 51 feet or
damage to mission essential		greater
subsystems.		Wind velocity 63 knots or greater
		Ship takes best heading

Top Level Performance Requirements for the Navy <u>Table 2</u>

Subsystem	Dynamic Response	Limiting Design Criteria
Helicopter	Roll	6.4 deg
_	Pitch	3.0 deg
	Vertical velocities at landing pad	6.5 ft/sec
	Slamming	20 occurrences/hr
	Deck Wetness	30 occurrences/hr
Replenishment	Roll	5.0 deg
	Pitch	3.0 deg
	Slamming	20 occurrences/hr
	Deck Wetness	30 occurrences/hr
Personnel	Roll	10.0 deg
	Pitch	3.0 deg
	Vertical accelerations at Bridge	$12.8 \text{ ft/sec}^2(0.4\text{g})$
	Lateral accelerations at Bridge	$6.4 \text{ ft/sec}^2(0.2g)$
	Slamming	20 occurrences/hr
	Deck Wetness	30 occurrences/hr

Design Limiting Criteria for the Navy <u>Table 3</u>

Further information to incorporate into the design criteria is the location of the ship's general operation. The defined Sea State is not only affected by the significant wave height and wind speed, but also the modal period, which contributes to developing an accepted wave spectrum. The wave spectrum must also be considered when developing a ship's operating envelopes. The wave spectrum is the wave height time history and can be thought of as representing the distribution of energy as a function of wave frequency, [10]. Remember that the plotted wave frequency is representing the modal period for a geographic region. The wave height time history is geographically based, producing a wave spectrum that shows a range of wave energy levels for the same wave height. Figure 1 shows a sample wave spectrum for a wave height of 6.17 feet, which is associated with Sea State 4, [10].



Wave Spectrum for 6.17 feet (Sea State 4) Figure 1

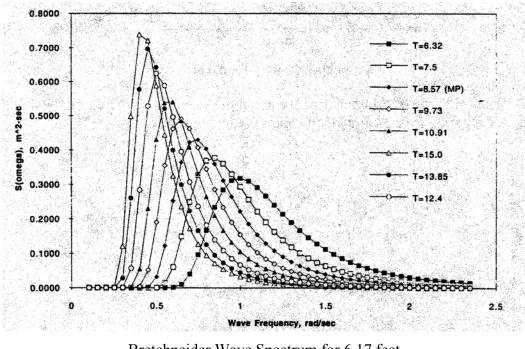
The peaks are associated with the maximum energy and indicate the associated wave frequencies, i.e. modal periods. There are several ways to develop a wave spectrum. The two most common methods are the Pierson - Moskowitz and the Bretschneider spectrums. Pierson – Moskowitz uses wind speed (V) as its primary parameter and the wave spectrum is developed based on the following formula, [11].

$$S(\omega) = \left[\frac{8.1e^{-3}g^2}{\omega^5}\right]^{\left(-0.74\left(\frac{g}{V\omega}\right)^4\right)}; m^2s$$

The spectrum recommended by the International Towing Tank Conference is the Bretschneider spectrum, which is based on the significant wave height $(\xi_{1/3})$ and modal frequency (ω_m) . The associated equations are noted below, [10].

$$S(\omega) = \left(\frac{1.25}{4}\right) \left(\frac{\omega_m^4}{\omega^5}\right) \xi_{1/3}^{2\left(-1.25\left(\frac{\omega_m}{\omega^4}\right)^4\right)}$$

Figure 2 shows the Bretschneider wave spectrum for a 6.17 foot significant wave height (Sea State 4).



Bretchneider Wave Spectrum for 6.17 feet Figure 2

The Bretschneider wave spectrum uses the modal period information from a given locale based on the information provided by the Spectral Ocean Wave Model (SOWM), which is a hind cast mathematical model to generate the climatology database, [10]. Table 4 shows the relationship of modal period to Sea State based on the SOWM database.

Sea State	Significant Wave	Modal Period Range	Sustained Wind Speed
	Height (ft)	(sec)	(kts)
0-1	0 - 0.33	-	0-6
2	0.33 - 1.64	3.3 - 12.8	7-10
3	1.64 - 4.10	5.0 - 14.8	11 – 16
4	4.10 - 8.20	6.1 - 15.2	17-21
5	8.20 - 13.12	8.3 - 15.5	22 - 27
6	13.12 - 19.69	9.8 - 16.2	28-47
7	19.69 - 29.53	11.8 - 18.5	48 - 55
8	29.53 - 45.93	14.2 - 18.6	56 - 63
>8	> 45.93	15.7 - 23.7	>63

* It is customary to use statistics from the North Atlantic when geographic location is not specified for the ship design.

North Atlantic Sea State Table* <u>Table 4</u>

The Navy uses this information as the design criteria for its engineers. The information from Table 4 on the relationship between modal period and Sea State is pertinent to the analysis process used in this project, as well as both the Pierson – Moskowitz and Bretschneider formulations.

B. BACKGROUND AND SET-UP FOR ANALYSIS

In looking at seakeeping trends in various sea states, this project choose specific models to scale for the comparative analysis while using specific software to collect the data. As previously discussed, there are numerous options for HSV's but a good starting point for analysis is the monohull. The results from analyzing monohulls can also be applied to other ship designs. The ship characteristics used in the seakeeping analysis came from the Series 64 models that were designed, constructed and tested at the David Taylor Model Basin, [12]. This series of models was chosen because Series 64 has become a benchmark series for resistance data. The scaled-up ship characteristics were entered into and evaluated with SHIPMO, the software developed by Maritime Research Institute Netherlands (MARIN), [13].

1. Series 64

In ship design, engineers often use data from existing ships as well as from results of experiments. Data from experiments on a methodical series of models can prove very useful

in looking at trends and in developing an optimal ship design. David Taylor Model Basin's Series 64, as presented by Yeh [12], is a key example of this. The motivation for Series 64 came from the lack of existing data at the time for the performance of ships at high speeds, speeds with a speed-length equal to or greater than 2. The engineers were particularly looking for ship resistance data. The parent model was based on its high speed performance (above the speed-length ration of 2.6) during bare-hull resistance tests in comparison to existing models at the Model Basin. Because this was an initial series, only three parameters were selected to develop the models.

The parameters selected were B/H, $\Delta/(0.01L)^3$, and the block coefficient, C_B. Three models for each change in parameter were deemed adequate to determine a trend in the effect of the change, resulting in 27 models. B/H was selected over L/B because naval architects were accustomed to seeing this parameter in contours and 2, 3 and 4 were selected because they covered most ships. Block coefficient was selected because its variance in resistance effects was greater than varying prismatic coefficient, C_p. 0.35, 0.45 and 0.55 also cover the range for block coefficient of most ships. C_p was subsequently set at 0.63 for all models. $\Delta/(0.01L)^3$ was chosen because displacement is an excellent indicator of payload. For most ships the range of this parameter is 15-55 and is coupled with the block coefficient, lower values apply to lower values for block coefficient.

2. SHIPMO

In accordance with MARIN's website description, [13], this software program calculates the motions and behavior of a ship using a program based on 'strip theory'. This program uses characteristic points from the ship's geometric shapes, specifically the body plan view. It uses station characteristics for its calculations, as well as specific ship speed and wave characteristics. The product is based on the slenderness of the ship's hull and on the linearity of the hydrodynamic forces. The program integrates the ship's response at the individual stations over the length of the ship to produce the resultant ship motions.

SHIPMO uses full scale ships, so to collect data for a small displacement ship the selected Series 64 models were scaled to size with a fixed displacement of 2500 tons. 2500 tons was selected because it is comparable to the displacement for a Corvette style small displacement, high speed ship. Six Series 64 models were selected to use with SHIPMO.

These models were selected based on the same parameters as the original series, but specifically using C_B and B/H. Table 4 shows the models chosen, their applicable parametric data and their scaled data.

Series 64 Models			<u>Scaled Characteristics</u> (displacement = 2500 tons)			
Number Cb B/H (disp)/(0.01L)^3			L (ft)	B (ft)	H (ft)	
4787	0.55	2	55	357.00	29.86	14.93
4794	0.55	4	40	357.00	36.01	9.00
4796	0.45	2	45	381.57	31.94	15.96
4802	0.45	4	45	381.57	45.14	11.28
4805	0.35	2	35	414.91	36.54	18.27
4812	0.35	4	25	464.16	46.42	11.60

Series 64 Models and Scaled Ship Characteristics <u>Table 5</u>

The model body plan diagrams are located in Appendix A. The full-scale data from these curves was entered into SHIPMO to obtain the resulting data, which is discussed in further detail in the following section.

3. Bales Estimator

Nathan Bales from David Taylor Naval Ship Research and Development Center developed a comparative parameter for seakeeping, [14]. The seakeeping rank, R, was based on the optimization of 20 destroyer type hulls in long-crested, head seas. The analysis was based on a 4300 ton displacement. A total of six parameters were selected to develop the estimator. The parameters used were the waterplane coefficient forward of amidships (C_{WF}) and aft of amidships (C_{WA}), the draft to length ratio (T/L), the cut-up ratio (c/L), and the vertical prismatic coefficient forward of amidships (C_{VPF}) and aft of amidships (C_{VPA}). Bales went the step further to normalize the values he obtained to fit the scale of 1.0-10.0 with hull receiving a 10.0 to have the best seakeeping ability. The following equation is Bale's seakeeping rank equation.

$$R = 8.422 + 45.104 \ C_{WF} + 10.078 \ C_{WA} - 378.465 \left(\frac{T}{L}\right) + 1.273 \left(\frac{c}{L}\right) - 23.501 \ C_{VPF} - 15.875 \ C_{VPA}$$

This equation is a good tool for comparison of hull types and their performance in the water. To accurately fall on the 1.0-10.0 scale, the hull and its parameters must meet the constraints set forth in [14]. The above equation was used to calculate a seakeeping rank for each of the six Series 64 models to further establish the seakeeping trends of the hulls and will be presented in the discussion of results.

C. ANALYSIS

Not only does SHIPMO need user defined ship characteristics, wave data must be defined. This project set the waves as long crested with wavelengths from 20 to 1000 feet in increments of 20 feet and the regular waves have wave amplitudes of 1 foot. This is set for all program runs in SHIPMO. The two parameters that varied between programs runs were the wave angle and ship speed. Each run only changed one of the two parameters at a time. The wave angle was varied from 0 degrees (head seas) to 180 degrees (following seas) in reference to the ship's bow in increments of 15 degrees. The ship's speed was varied from 0 to 100 ft/sec (approximately 0 to 60 knots) in increments of 10 ft/sec. This resulted in a total of 143 programs runs for each model. Appendices B.1 through B.6 includes SHIPMO input file with waves at 0 degrees and the ship traveling at 0 ft/sec. The output file for each program run presented a summary of the input data, computed hydrostatic data and the ship's seakeeping response. Appendix C presents an example output file for Model 4787 with the waves at 0 degrees and the ship traveling at 0 ft/sec. The program also produced a corresponding MATLAB file to be used with further analysis.

The associated MATLAB files were then used with the m-file $v_speed_long.m$, which is located in Appendix D. The MATLAB program prompts the user to specify the ship, the significant wave height in feet and the wave spectrum method to use. Pierson – Moskowitz is indicated with "0" and Bretschneider is indicated by "1". If Bretschneider is chosen, then the user is also prompted to indicate the modal period in seconds. The program then uses all 143 runs for the specified ship to produced two plots of the ship's response in pitch and heave. The roll response was not calculated due to the high dependence on the metacentric height, which is specific to a ship's loading configuration. Contour plots for each ship were plotted using a significant wave height of 10 feet which falls under Sea State 5 and corresponds to the lowest top level performance requirement by the Navy as indicated in Table 2. This is the only wave height looked at due to the fact that $v_speed_long.m$ normalizes the wave height, allowing the contour curves to be an indication of the ship's response regardless of wave height. The contour plots using the Pierson – Moskowitz spectrum are located in Appendix E. The contour plots for the Bretschneider spectrum used the extremis of the modal period range (9.8 and 16.2 seconds) as well as the average modal period (13.0 seconds) corresponding to Sea State 5. These contour plots are located in Appendix F. Due to space limitations, only the two primary responses in heave and pitch are presented. Derived responses such as those shown in Table 3 can be generated from the primary responses. A ship that possesses favorable response characteristics in both heave and pitch, will in general exhibit acceptable seakeeping behavior in all other derived responses.

IV. CONCLUSIONS AND RECOMMENDATIONS

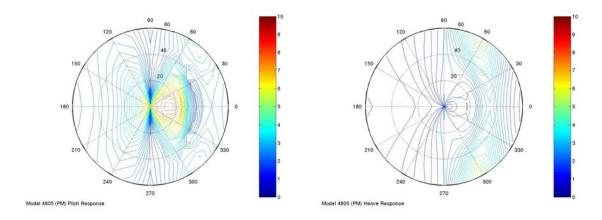
A. TRENDS IN SEAKEEPING

The plots showed that the wave response patterns from the models are the same but with varying levels of intensity. The plots also indicate that the six models fall into two distinct groups based on the pitch and heave response pair. Initial inspection shows the two groupings by response are formed due to the beam to draft ratio (B/T) of the models. Table 6 groups the models by B/T as well as by increasing intensity in the response.

				(disp)/	Ax	S				
Model	Cb	Cx	B/T	(0.01L)^3	(sq. in.)	(sq. ft.)	Срv	R		
Group 1										
4787	0.55	0.873	2	55	43.96	11.388	0.723	3.70		
4796	0.45	0.714	2	45	35.97	10.411	0.591	-0.47		
4805	0.35	0.556	2	35	27.98	9.907	0.460	-0.46		
				Group	2					
4802	0.45	0.714	4	45	35.97	11.109	0.591	6.22		
4794	0.55	0.873	4	40	31.98	10.151	0.723	3.70		
4812	0.35	0.556	4	25	19.99	8.996	0.460	1.68		

Seakeeping Response Groups <u>Table 6</u>

Figure 3 shows samples plots for model 4805 using the Pierson-Moskowitz spectrum that is a good representation of the wave patterns produced by all models.



Contour Plots for Model 4805 Figure 3 As background, the plots illustrate the ship's motion based on the direction of the waves interacting with the ship relative to the bow which is oriented at 0 degrees and the speed of the ship which increases from the center to the outer ring. A scale of 0-10 feet illustrates the intensity of the motion. For example, the ship has a high intensity response in pitch when traveling under 25 knots with head seas versus following seas at any speed.

All models showed the highest pitch response with ship speeds between 0 and 25 knots and a concentration of motions from the bow of the ship to 30 degrees port and starboard. Group 1 had a high response in pitch with all models and modal periods showing an intensity level of 10 feet. Group 2 had a relatively mild response in all models and all modal periods with having an intensity level at or below 7 feet except for the 13-second modal period which reached 10 feet. Model 4794 and 4812 showed an intensity level of 10 feet for the 13-second modal period. In heave, the largest response occurred for speeds over 40 knots with the wave pattern motions concentrated 30 to 60 degrees off the bow for both port and starboard sides of the ship. This time Group 1 had the lower response with a max intensity level of 6 feet except for model 4805 reached a level of 6.5 feet in the 13-second modal period. Group 2 showed a higher response in heave, reaching an intensity level of 10 feet for ship 4812 in the 13-second model period. Overall, Group 1 has a very predictable seakeeping response regardless of modal period. Group 2 has a range of levels in responses. The hull with the least seakeeping abilities comes from Group 2, 4812. The optimum hull for seakeeping abilities also comes from Group 2, model 4802. This model has a low intensity response for both pitch and heave, a highly desirable quality for seakeeping.

Another tool that was looked at to predict the seakeeping abilities of the models was the Bales seakeeping rank, (R). The trend in intensity in each group is dictated by the models' size in terms of length, waterplane area (Ax) and wetted surface area (S). Table 6 also shows the Bales rank trend for the groups. The Bales rank for model 4802 further indicates that its hull has higher seakeeping abilities over the other hull forms. The trend for R decreases as the intensity of the seakeeping response increases. This was expected, but the actual calculated values were not. The table shows several low values and two negative values. Upon further inspection, this is attributed to the fact that the parameters used to calculate R fell outside the constraints documented by Bales. Table 7 quickly shows the key parameters for each model and how each relates to the specified constraints. The other factor is Bales based his rank formula on a displacement of 4300 tons.

Model	4787	4794	4796	4802	4805	4812
u =	0.753	0.843	0.828	0.818	0.862	0.833
L/u =	474.010	423.389	460.575	466.389	481.109	557.060
B/u =	39.647	42.707	38.553	55.174	42.370	55.711
T/u =	19.823	10.674	19.265	13.787	21.185	13.922
Cwf =	0.554	0.554	0.514	0.526	0.468	0.424
Cwa =	1.013	1.013	0.936	0.945	0.868	0.784
T/L =	0.025	0.025	0.042	0.030	0.044	0.025
Cvpf =	0.944	0.944	0.796	0.752	0.691	0.752
Cvpa =	0.580	0.580	0.504	0.479	0.430	0.482
R** =	3.70	3.70	-0.47	6.22	-0.46	1.68

*italics indicate values outside of constraint ranges

**a value of c/L = 0.8 for all models was used

Model Parameters for Bales Seakeeping Rank <u>Table 7</u>

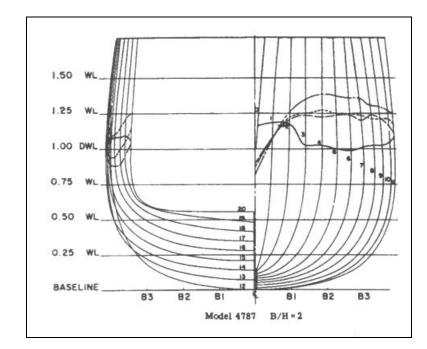
The information from this study can be used in a variety of different ways. It can obviously be used from a purely analytical point by looking for trends in hull forms for seakeeping. Engineers can also use it as a stepping-stone in the pursuit of a hull form with optimal seakeeping abilities. The research and information presented here gives a large enough range of information that a choice of hull form can be made based on desired speed, specifically seakeeping performance at high speeds. The information can be applied to the pursuit of multihull vessels or SWATH type hulls. The information and process produced from the study has use in the immediate future as the Navy develops the design of its Littoral Combat Ship (LCS).

B. RECOMMENDATIONS

As an initial seakeeping study, this research gives an overview and introduction to the trends of seakeeping abilities for monohulls. To further predict the seakeeping trends for hull forms as well as increase the accuracy of information, a continuation of this study looking at all the Series 64 models should be conducted. Doing this would produce a solid base of

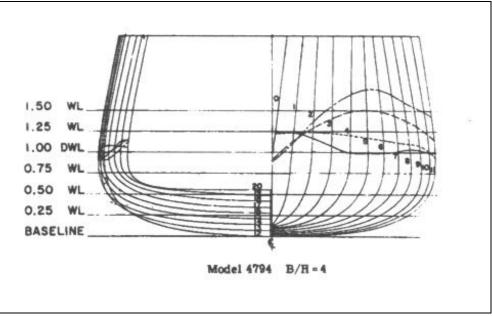
information on how a large variety of accepted hull forms are expected to perform in the seas. Another tool to increase the analysis is SHIPMO. SHIPMO is a versatile program and this study looked at a relatively small range of the possible wave parameters. The engineer could vary the wave amplitude, change the ship's velocity or wave direction as well as look at other modal period ranges. Looking at other values and ranges would only increase the accuracy of information on seakeeping trends. Lastly, doing a similar seakeeping analysis for mulithulls would round out the base of knowledge on seakeeping performance. A full base of information allows engineers to accurately weight the hull performance during the design process when choosing a hull form for a specific mission.

V. APPENDICIES

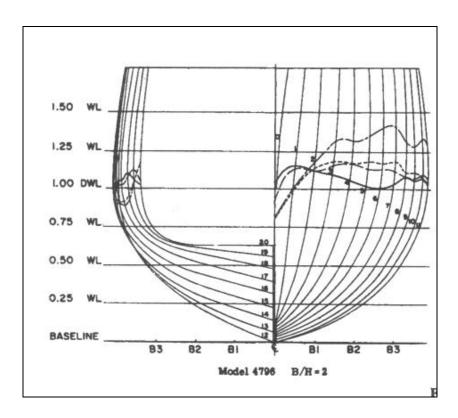


APPENDIX A. MODEL BODY PLANS

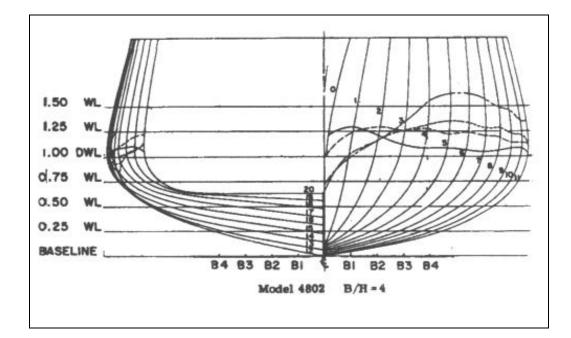
Model 4787



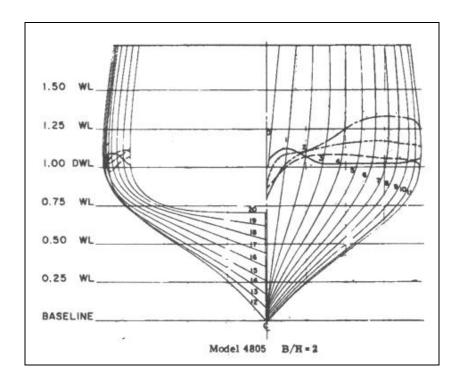
Model 4794



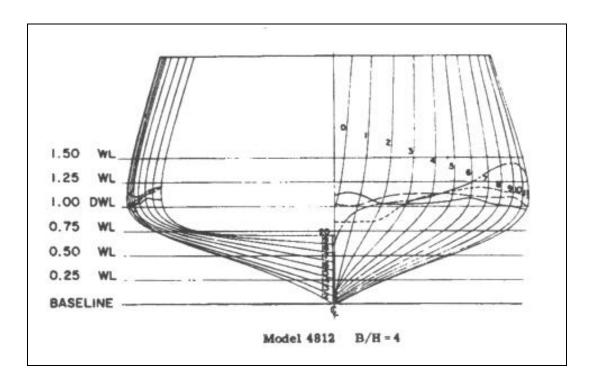
Model 4796



Model 4802



Model 4805



Model 4812

APPENDIX B.1 MODEL 4787 SHIPMO INPUT FILE

Мо

Model													
	0				0	1	0		0	1	0	0 20	0
357.	.0000	1.	.9905	32	.1740		1.26E	-05			0.0	0.0000	
33.	.0000	-26.	.0000	1	.0000								
5	160	.6500	0.	.0000	0								
0.	.0000	-13.	.1250										
	6667												
	. 3333		.7080										
	.6667												
	.8750	0.											
7	142	.8000	0.	.0000	0								
0.	.0000	-13.	.1250										
1.	.0417	-12.	.2913										
1.	6667	-11.	.4580										
2.	.5000	-9.	. 4997										
	9583		.7080										
	4167	-3.	7500										
	.5417												
		.9500		0000	0								
				. 0000	0								
	.0000												
	.4583		.5830										
	.5833												
3.	.0833	-9.	.4997										
4.	.5830	-7.	.7080										
5.	.0000	-9.	.3750										
5.	2083	-3.	.7500										
5.	4166	0.	.0000										
		.1000		.0000	0								
		-13.											
	8750												
	. 3333												
	.5834												
	.6667		.7080										
	.0830		.7500										
	.0830	0.											
		.2500		.0000	0								
0.	.0000	-14.	.7499										
3.	.1250	-13.	.4580										
4.	.5833	-9.	.4997										
5.	9167	-11.	.4580										
7.	.9167	-19.	.5830										
8.	.5417	-3.	.7500										
	.5417	0.	.0000										
8	71	.4000	0.	.0000	0								
0.	.0000												
	1667		.4580										
	.0417		. 4997										
	2917		.4580										
	.7500		. 3750										
	. 3750		.7080										
	.0000		.7500										
	.0283		.0000										
8		.5500		.0000	0								
0.	.0000		.7910										
2.	.9167	-14.	.1666										
5.	.8333	-13.	.1250										
8.	.3330	-11.	.4580										
10.	.0000	-9.	.3750										
	.8330		.7080										
	2500		.7500										
	4583		.0000										
8		.7000		.0000	0								
	.0000		.7910		Ŭ								
	.9167		.1250										
	.4583		.9583										
	.5833		.4580										
⊥⊥.	.2500	-9.	.3750										

0.0000 2.9167 11.4583	-3.7500 0.0000 500 0. -15.0000 -14.7910 -14.7500	0000	0
0 0000	0.0000	0000	0
0.0000 2.9167 6.2500	-15.2080 -15.2080 -15.0000	0000	0
0.0000 5.6250 10.8330	000 0. -15.2080 -15.2080 -12.9166	0000	0
14.1667 15.0000 15.0000	-7.7080 -3.7500 0.0000 500 0. -14.1663 -13.9580	0000	0
15.4167 6 -71.4 0.0000 6.0416 11.8750 14.7916	0.0000 000 0. -13.1250 -12.4996 -10.4163 -7.7080	0000	0
4.7917 10.0000 13.9583 15.2083	-3.7500 0.0000 200 0. -11.8750 -11.8750 -10.6250 -7.7080 -3.7500 0.0000	0000	0
6 -107.1 0.0000	000 0. -11.0413 -10.8330 -9.3750 -7.7080 -3.7500 0.0000	0000	0

	-9.9996 -9.7850 -8.5413 -6.2500 -3.7500					
	0.0000					
	8000 0.0000					
0.0000	-8.9580 -8.5413					
4.7917 9.3750	-7.7080					
	-5.8333					
	-3.7500					
	0.0000					
6 -160.	6500 0.0000					
0.0000	-7.9163					
4.7917	-7.4997					
9.3750	-6.6666					
12.0830	-5.4166					
13.1250	-3.7500					
13.7500	0.0000					
6 -178.	5000 0.0000					
0.0000	-6.6666					
4.7917						
9.3750	-6.6666					
11.4583	-5.8333					
12.9167	-3.7500					
12.9167	0.0000					
-2.0	0.0000					
9999.0	0.0	0.0				
1.0000		.0000	20.0000	0.0000	0.0000	0.0000
0.0000	000.0000 00	.0000				

APPENDIX B.2

MODEL 4794 SHIPMO INPUT FILE

Model 4794

Model	4794												
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357.	.0000	1	.9905	32.	1740		1.2	6E-05			0.0	0.0000	
33.	.0000	-26	.0000	1.	0000								
5	160	.6500	0	.0000	0								
0.	.0000	-7	.8572										
0.	.7143	-б	.9148										
1.	.6667		.0476										
	.1429		.3810										
	.1429		.0000										
			0.	0000	0								
	.0000		.8572	.0000	0								
	.9048		.9148										
	.5714		.0476										
			.3810										
	.2857												
	.5238		.0000		0								
5			0	.0000	0								
	.0000		.0953										
	.5714		.9148										
	.4714		.0476										
6.	.4286		.3810										
6.	.6667	0	.0000										
6	107	.1000	0	.0000	0								
0.	.0000	-8	.3334										
2.	.1429	- 8	.0953										
5.	.9524	-б	.9148										
7.	.6190	-4	.0476										
	.5714		.3810										
	.8095		.0000										
6			0.0000	0000	0								
	.0000		.3334		0								
	. 3333		.0953										
	.1429		.9148										
	. 5238		.0476										
	.4762		.3810										
	.7143		.0000		0								
6			0	.0000	0								
	.0000		.5715										
	.2381		.0953										
	.8095		.9148										
	.4286		.0476										
	.3810		.3810										
12.	.3810		.0000										
7	53	.5500	0	.0000	0								
	.0000	-8	.5715										
3.	.8095	-8	.5715										
7.	.1429	-8	.0953										
10.	.2381	-б	.9148										
13.	.0952	-4	.0476										
14.	.0476	-2	.3810										
14.	.0476	0	.0000										
7	35	.7000	0	.0000	0								
0.	.0000	-8	.5715										
5.	.0000		.5715										
9.	.0476		.0953										
	.9048		.9148										
	.7619		.0476										
	4762		.3810										
	.2381		.0000										
		.8500		.0000	0								
	.0000		.0477		5								
	.9048		.5715										
	. 4762		.0953										
	. 3333		.0953										
	.9524		.0476										
10.	.6667	-2	.3810										

$\begin{array}{cccccc} 16.4286 & 0.0000 \\ 7 & 0.0000 & 0 \\ 0.0000 & -9.0477 \\ 6.9048 & -8.8096 \\ 11.4286 & -8.0953 \\ 14.2857 & -6.9148 \\ 16.9048 & -4.0476 \end{array}$.0000	0
$\begin{array}{rrrr} 17.6190 & -2.3810 \\ 17.3810 & 0.0000 \\ 7 & -17.8500 & 0 \\ 0.0000 & -9.0477 \\ 6.9048 & -9.0477 \\ 12.3810 & -8.0953 \end{array}$.0000	0
$\begin{array}{rrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrr$.0000	0
$\begin{array}{rrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrr$.0000	0
$\begin{array}{cccccccccccccccccccccccccccccccccccc$.0000	0
$\begin{array}{cccc} 0.0000 & -8.0953 \\ 7.8571 & -7.6191 \\ 13.0952 & -6.9148 \\ 18.0952 & -4.0476 \\ 19.2857 & -2.3810 \\ 19.2857 & 0.0000 \\ 6 & -89.2500 & 0 \end{array}$.0000	0
$\begin{array}{cccc} 0.0000 & -7.8572 \\ 9.5238 & -6.9148 \\ 14.2857 & -5.4762 \\ 17.1429 & -4.0476 \\ 18.8095 & -2.3810 \\ 18.8095 & 0.0000 \\ 6 & -107.1000 & 0 \end{array}$	0000	
$\begin{array}{rrrr} 0.0000 & -6.1905 \\ 6.1905 & -5.9524 \\ 11.4286 & -5.4762 \\ 15.9524 & -4.0476 \\ 18.5714 & -2.3810 \\ 18.8095 & 0.0000 \end{array}$		
$\begin{array}{ccccc} 6 & -124.9500 \\ 0.0000 & -5.4762 \\ 6.1905 & -5.4762 \\ 11.4286 & -4.7619 \\ 14.2857 & -4.0476 \\ 17.6190 & -2.3810 \\ 18.8095 & 0.0000 \end{array}$	0.	0000
$\begin{array}{cccccc} 6 & -142.8000 & 0 \\ 0.0000 & -4.7619 \\ 6.1905 & -4.7619 \\ 11.4286 & -4.0476 \\ 15.2381 & -3.8096 \\ 17.1492 & -2.3810 \\ 17.6190 & 0.0000 \end{array}$.0000	
	.0000	

0.0000 6.1905 11.4286 14.7619 16.1905	-4.0476 -4.0476 -3.3333 -3.5715 -2.3810					
16.6667	0.0000					
6 -178.	5000 0	.0000				
0.0000	-3.5714					
6.1905	-3.5714					
11.4286	-3.3333					
14.7619	-3.5715					
15.7143	-2.3810					
15.7143	0.0000					
-2.0	0.0000					
9999.0	0.0	0.0				
1.0000	20.0000	1000.0000	20.0000	0.0000	0.0000	0.0000
0.0000	000.0000	00.0000				

APPENDIX B.3 MODEL 4796 SHIPMO INPUT FILE

Model 4796

Model	4796												
0	0	0	0	0	0	1	0	1	0	1	0	0 20	0
381	.5714	1	.9905	32.	1740		1.26E	-05			0.0	0.0000	
33	.0000		.0000		0000								
			0.		0								
				.0000	0								
	.0000		.1666										
	.5556		.2222										
1	.1111	-8	.0556										
1	.6667	-3	.8889										
1	.9444	0	.0000										
5	145	.3608	0.	.0000	0								
	.0000		.1666										
	.1111		.2222										
	.5000		.0556										
	.3333		.8889										
3	.8889	0	.0000										
6	127	.1907	0.	.0000	0								
0	.0000	-14	.4444										
1	.9444	-12	.2222										
	.0556		.0000										
	.8889		.0556										
	.0000		.8889										
	.8333		.0000										
			0.	.0000	0								
0	.0000	-14	.7222										
2	.7778	-12	.2222										
4	.1667	-10	.0000										
5	.2778	-8	.0556										
	.6667		.8889										
	.7778		.0000										
6		.8505		0000	0								
				.0000	0								
	.0000		.2777										
	.8889		.2222										
	.5556		.0000										
6	.9444	-8	.0556										
8	.6111	-3	.8889										
9	.4444	0	.0000										
7	72	.6804	0.	.0000	0								
	.0000		.5555										
	.6667		.4444										
	.4444		. 2222										
	.6667		.0000										
	.0556		.0556										
	.0000		.8889										
11	.1111		.0000										
7	54	.5103	0.	.0000	0								
0	.0000	-15	.5555										
2	.5000	-14	.4444										
	.5556		.2222										
		-10	.0000										
	.4444		.0556										
	. 3889		.8889										
	.2222		.0000		0								
7		.3402		.0000	0								
	.0000		.8333										
3	.3333	-14	.4444										
6	.6667	-12	.2222										
8	.8889	-10	.0000										
	.5556		.0556										
	.5000		.8889										
	.3333		.0000										
13		.1701		.0000	0								
					U								
	.0000		.1111										
	.4444		.4444										
	.7778		.2222										
10	.0000	-10	.0000										

13.6111	-8.0556 -3.8889 0.0000 0000 0 -16.1111 -14.4444 -12.2222 -10.0000 -8.0556 -3.8889 0.0000	.0000	0
12.2222 14.4444 15.2778 8 -18. 0.0000 3.0556 5.8333 9.1667	-8.0556 -3.8889 0.0000 1701 0 -16.1111 -15.5555 -14.4444 -12.2222	.0000	0
11.3889 13.0556 15.0000 15.8333 7 -36. 0.0000 8.6111	1/01 0 -16.1111 -15.5555 -14.4444 -12.2222 -10.0000 -8.0556 -3.8889 0.0000 3402 0 -16.1111 -12.2222 -10.0000	.0000	0
7 -54. 0.0000 7 7776	-16.1111 -12.2222 -10.0000 -8.0556 -3.8889 -2.2222 0.0000 5103 0 -15.2777 -12.2222	.0000	0
15.8333 15.8333 16.3889 16.9444 7 -72. 0.0000	-10.0000 -8.0556 -3.8889 -2.2222 0.0000 6804 0 -14.1666 -12.2222 -10.0000	.0000	0
12.7778 15.8333 16.3889 16.9444 7 -90. 0.0000	-8.0556 -3.8889 -2.2222 0.0000 8505 0 -12.5000	.0000	0
11.6667 13.8889 15.8333 16.1111 16.6667 6 -109. 0.0000 10.2778		.0000	
12.7778 15.2778 15.8333 16.3889 6 -127. 0.0000 10.5556	-9.4444 -3.8889 -2.2222 0.0000 1907 -9.7223 -6.9445	0.	0000
0.0000	-3.8889 -2.2222 0.0000	.0000	

$\begin{array}{c} 13.8889\\ 14.7222\\ 15.5556\\ 6&-163.\\ 0.0000\\ 10.0000\\ 11.9444\\ 13.3333\\ 14.4444\\ 14.7222\\ 6&-181.\\ 0.0000\\ 10.0000\\ 11.3889\\ 12.7778\\ 13.3333\\ 13.8889\\ 05.0\\ \end{array}$	$\begin{array}{c} -6.9445 \\ -5.5556 \\ -5.2778 \\ -3.8889 \\ -2.2222 \\ 0.0000 \\ 7010 \\ 0 \\ -6.9445 \\ -5.5556 \\ -5.2778 \\ -3.8889 \\ -2.2222 \\ 0.0000 \\ 0.0000 \\ 0.0000 \\ \end{array}$					
9999.0 1.0000 0.0000	0.0	0.0 1000.0000 00.0000	20.0000	0.0000	0.0000	0.0000

APPENDIX B.4

MODEL 4802 SHIPMO INPUT FILE

Model 4802

Model	4802												
	0		0	0	0	1	0		0	1	0	0 20	0
	.5714		.9905		1740		1.26E	-05			0.0	0.0000	
33.	.0000	-26	.0000	1.	0000								
5	163		0.	.0000	0								
	.0000		.3182										
	.6818		.1818										
1.	.3636	-5	.4545										
2.	.2727	-2	.7273										
2.	.7273	0	.0000										
5	145	.3608	0.	.0000	0								
0.	.0000	-9	.5454										
1.	.5909	-8	.1818										
3.	.4091	-5	.4545										
4.	.7727	-2	.7273										
5.	.6818	0	.0000										
5	127	.1907	0.	.0000	0								
0.	.0000	-9	.5454										
2.	.7273	-8	.1818										
5.	.4545	-5	.4545										
7.	.2727	-2	.7273										
8.	.6364	0	.0000										
5	109	.0206	0.	.0000	0								
0.	.0000	-9	.7727										
3.	.6364	-8	.1818										
7.	.7273	-5	.4545										
9.	.7727	-2	.7273										
10.	.9091	0	.0000										
5	90	.8505	0.	.0000	0								
0.	.0000	-9	.7727										
5.	.2273	-8	.1818										
9.	.5455	-5	.4545										
12.	.2727	-2	.7273										
13.	.6364	0	.0000										
5	72	.6804	0.	.0000	0								
0.	.0000		.2273										
б.	.5909	-8	.1818										
11.	.3636		.4545										
14.	.3182	-2	.7273										
15.	.9091	0	.0000										
б	54	.5103	0.	.0000	0								
0.	.0000		.2273										
4.	.0909	-9	.5454										
8.	.6364	-8	.1818										
13.	.4091	-5	.4545										
19.	.3636	-2	.7273										
17.	.7273	0	.0000										
6	36	.3402	0.	.0000	0								
	.0000		.6818										
5.	.2273	-9	.5454										
9.	.5455	-8	.1818										
15.	.0000	-5	.4545										
17.	.9545	-2	.7273										
19.	.5455		.0000										
6	18	.1701	0.	.0000	0								
	.0000		.9091										
	.5909		.5454										
	.9091		.1818										
	.5909		.4545										
	.5455		.7273										
	.9091		.0000										
6			0.	.0000	0								
	.0000		.9091										
	.7273		.5454										
	.2727		.1818										
17.	.7273	-5	.4545										

20.6818 -2.7273 22.0455 0.0000 6 -18.1701 0 0.0000 -10.9091 8.6364 -9.5454 13.1818 -8.1818 19.0909 -5.4545	.0000 0	
$\begin{array}{cccccccccccccccccccccccccccccccccccc$.0000 0	
22.9545 -2.7273 23.8636 0.0000 5 -54.5103 0 0.0000 -10.2273 11.1364 -8.1818 19.0909 -5.4545 22.9545 -2.7273	.0000 0	
23.8636 0.0000 5 -72.6804 0 0.0000 -9.3182 7.2727 -8.1818	.0000 0	
$\begin{array}{rrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrr$.0000 0	
17.2727 -5.4545 19.0909 -4.3182 22.5000 -2.7273 23.6364 0.0000 5 -109 0206	0000	
0.0000 -7.5000 14.5455 -5.4545 19.3182 -4.3182 21.8182 -2.7273 23.1818 0.0000		
$\begin{array}{cccc} 6 & -127.1907 \\ 0.0000 & -6.5909 \\ 9.7727 & -5.4545 \\ 16.8182 & -4.3182 \\ 20.9091 & -2.7273 \\ 22.2727 & -1.3636 \\ 22.7273 & 0.0000 \end{array}$	0.0000	
$\begin{array}{rrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrr$.0000	
22.0455 0.0000	.0000	
21.1364 0.0000	.0000	
20.0000 0.0000 -2.0 0.0000 9999.0 0.0	0.0	

1.0000	20.0000	1000.0000	20.0000	0.0000	0.0000	0.0000
0.0000	000.0000	00.0000				

APPENDIX B.5 MODEL 4805 SHIPMO INPUT FILE

				0 01111 112	0 11 (1	• -		
0 0 0 0 414.9133 1.9905	0		1	0 1	0	1	0	0 20 0.0000
				1.26E-05			0.0	0.0000
33.0000 -26.0000								
5 177.8202 0	.0000	0						
0.0000 -15.5263								
0.5263 -13.9474								
1.0526 -9.4737								
1.5789 -4.7368								
2.1053 0.0000								
5 158.0624 0	.0000	0						
0.0000 -16.0527								
0.7895 -13.9474								
1.8421 -9.4737								
3.1579 -4.7368								
4.4737 0.0000								
5 138.3046 0	.0000	0						
0.0000 -16.0527								
1.0526 -13.9474								
2.8947 -9.4737								
5.0000 -4.7368								
6.5789 0.0000								
5 118.5468 0	.0000	0						
0.0000 -16.3158								
1.5789 -13.9474								
4.2105 -9.4737								
6.8421 -4.7368								
8.6842 0.0000								
5 98.7890 0	.0000	0						
0.0000 -17.1053								
2.1053 -13.9474								
5.2632 -9.4737								
8.4211 -4.7368								
10.5263 0.0000								
6 79.0312 0	.0000	0						
0.0000 -17.1053								
2.6316 -13.9474								
6.5789 -9.4737								
10.0000 -4.7368								
11.5789 -2.3684 12.3684 0.0000								
6 59.2734 0	.0000	0						
0.0000 -17.1053	.0000	0						
3.1579 -13.9474								
7.3684 -9.4737								
11.5789 -4.7368								
13.1579 -2.3684 13.9474 0.0000								
6 39.5156 0		0						
0.0000 -17.3685								
3.6842 -13.9474								
8.4211 -9.4737								
12.8947 -4.7368								
14.4737 -2.3684								
15.2632 0.0000								
6 19.7578 0	.0000	0						
0.0000 -18.1579								
4.2105 -13.9474								
9.4737 -9.4737								
14.2105 -4.7368								
15.7895 -2.3684								
16.5789 0.0000								
6 0.0000 0	.0000	0						
0.0000 -18.9474								
4.7368 -13.9474								
10.5263 -9.4737								
15.2632 -4.7368								
16.8421 -2.3684								

18 2604						
17.3684						
6 -19.75		000 0				
	18.9474					
	13.9474					
	-9.4737					
16.0526	-4.7368 -2.3684					
17.8947	-2.3684					
18.1579	0.0000					
6 -39.51	56 0.0	0000 0				
0.0000 -	18.9474					
3.9474 -	13.9474					
10.7895						
	-4.7368					
	-2.3684					
	0.0000					
6 -59.27		000 0				
0.0000 -		000 0				
	13.9474					
	-9.4737					
	-4.7368					
	-2.3684					
	0.0000					
6 -79.03	12 0.0	000 0				
0.0000 -	15.2632					
1.8421 -	13.9474					
9.7368	-9.4737					
16.8421	-4.7368					
18.4211	-2.3684					
19.4737	0.0000					
5 -98.78		000 0				
0.0000 -						
	-9.4737					
	-4.7368					
	-2.3684					
	0.0000					
5 -118.54		000				
0.0000 -						
	-9.4737					
	-4.7368					
17.8947	-2.3684					
18.9474	0.0000					
5 -138.30	46	0.0000				
0.0000 -	10.5263					
3.1579	-9.4737					
14.4737	-4.7368					
	-2.3684					
	0.0000					
4 -158.06		000				
	-8.9473					
	-4.7368					
	-2.3684					
17.6316						
	0.0000	000				
5 -177.82		000				
	-5.5263					
	-4.7368					
	-3.4211					
	-2.3684					
16.8421	0.0000					
5 -197.57		000				
0.0000	-7.3684					
11.8421	-4.7368					
14.2105	-3.6842					
15.2632	-2.3684					
16.3158	0.0000					
-2.0	0.0000					
9999.0	0.0	0.0				
1.0000	20.0000 1		20.0000	0.0000	0.0000	0.0000
	00.0000	00.0000	20.0000	0.0000	0.0000	0.0000
0.0000 0		00.0000				

APPENDIX B.6 MODEL 4812 SHIPMO INPUT FILE

Model 48

Model	4812												
	0	0	0	0	0	1	0	1	0	1	0	0 20	0
464	.1589	1.	.9905		.1740		1.26E-				0.0	0.0000	
	.0000		.0000		.0000								
5	198	.9252	0	.0000	0								
0	.0000	-10.	.0000										
0	.6667	-8.	.6667										
1	.3333	-б.	.0000										
2	.0000	-3.	.0000										
2	.6667	0.	.0000										
5		.8224		.0000	0								
	.0000	-10.											
	.0000		.6667										
	.3333		.0000										
	.0000		.0000										
	.6667	0.		0000	0								
	154			.0000	0								
	.0000		.0000										
	.6667		.6667										
	.6667		.0000										
	.6667		.0000										
5		.6168		0000	0								
	.0000	-10.			0								
	.6667		.6667										
	.3333		.0000										
8	.6667	-3.	.0000										
11	.3333	0.	.0000										
5	110	.5140	0	.0000	0								
0	.0000	-10.	.3334										
2	.3333	-8.	.6667										
	.6667		.0000										
	.6667		.0000										
	.6667		.0000										
5		.4112		.0000	0								
	.0000	-10.											
	.0000		.6667										
	.3333		.0000										
	.0000	-3.											
6		.3084		0000	0								
	.0000		.0000		0								
	.0000		.6667										
	.6667		.0000										
14	.6667	-3.	.0000										
17	.0000	-1.	.3333										
18	.0000	0.	.0000										
б		.2056		.0000	0								
0	.0000	-11.	.0000										
	.3333		.6667										
	.0000		.0000										
	.3333		.0000										
	.6667		.3333										
19	.6667	.1028	.0000	.0000	0								
	.0000		.6667	.0000	0								
	.3333		.6667										
	.3333		.0000										
	.0000		.0000										
	.0000		. 3333										
	.0000		.0000										
6	0	.0000	0	.0000	0								
0	.0000	-11.	.6667										
	.0000		.6667										
	.3333		.0000										
	.6667		.0000										
21	.3333	-1.	.3333										

22.6667 0.0000 6 -22.1028 0.0000 0 0.0000 -11.6667 6.0000 -8.6667 13.6667 -6.0000 20.3333 -3.0000 22.3333 -1.3333 23.3333 0.0000 6 -44.2056 0.0000 0 0.0000 -11.6667 5.6667 -8.6667 13.6667-6.000021.6667-3.000024.3333-1.3333 25.0000 0.0000 6 -66.3084 0.0000 0 0.0000 -10.6667 4.6667 -8.6667 13.6667 -6.0000 -3.0000 -1.3333 21.6667 24.3333 25.0000 0.0000 6 -88.4112 0.0000 0 0.0000 -9.3334 3.0000 -8.6667 12.0000 -6.0000 -3.0000 -1.3333 21.6667 24.0000 24.6667 0.0000 5 -110.5140 0.0000 0 0.0000 -8.6667 L0.3333 -6.0000 10.3333 20.6667 -3.0000 -1.3333 23.3333 24.6667 5 -132.6168 0.0000 0.0000 -8.0000 7.6667 -6.0000 19.6667 -3.0000 23.3333 -1.3333 24.0000 0.0000 5 -154.7196 0.0000 0.0000 -6.6667 3.6667 -6.0000 18.6667 -3.0000 22.6667 -1.3333 23.3333 0.0000 4 -176.8224 0.0000 0.0000 -5.6667 18.0000 -3.0000 21.6667 -1.3333 22.6667 0.0000 4 -198.9252 0.0000 0.0000 -4.6667 -3.0000 -1.3333 17.3333 20.6667 0.0000 22.0000 4 -221.0280 0.0000 0.0000 -3.6667 L6.6667 -3.0000 16.6667 20.0000 -1.3333 0.0000 20.6667 0.0000 -2.0 9999.0 0.0 0.0 1.0000 20.0000 1000.0000 20.0000 0.0000 0.0000 0.0000 0.0000 000.0000 00.0000

APPENDIX C

Model 4787

SAMPLE SHIPMO OUTPUT FILE FOR MODEL 4787

 OOPTION CONTROL TAGS
 A
 B
 C
 D
 E
 F
 G
 H
 I
 J
 K
 L
 M
 N
 NS
 NPAC

 0
 0
 0
 0
 0
 1
 0
 1
 0
 20
 0
 20
 0

0***BASIC INPUT DATA***

INFINITE DEPTH ****** * * * * * * * 0 LENGTH= 357.000 DENSITY= 1.9905 GAMMA= 64.0423 GNU= 0.126000E-04 GRAVITY= 32.1740 OBILGE KEEL DATA: XFOR= 33.0000 XAFT= -26.0000 LENGTH= 59.0000 WIDTH= 1.0000 OSTA NO XAXIS 1/2BEAM DRAFT AREA AREA COEF ZBAR BILGE R AREA COEF2 160.6500 1.8750 13.1250 33.7668 0.6861 -5.1895 0.0000 0.6861 1 3.541713.125072.39095.416625.4163160.3840 0.7787 -5.5049 0.0000 0.5825 -9.3555 0.0000 2 142.8000 0.7787 0.5825 3 124,9500 107.1000 7.0830 13.7500 164.0813 0.8424 -6.0174 0.0000 0.8424
 89.2500
 8.5417
 19.5830
 226.2390
 0.6763
 -6.8682
 0.0000
 0.6763

 71.4000
 10.0283
 14.1666
 232.6713
 0.8189
 -6.0524
 0.0000
 0.8189
 5 6 0.8261 53.5500 11.4583 14.7910 280.0273 -6.3950 0.0000 7 0.8261 8 35.7000 12.5000 14.7910 295.4970 0.7991 -6.0616 0.0000 0.7991 17.8500 17.8500 13.3330 15.0000 369.4937 0.0000 14.1667 15.2080 386.9037 -7.0913 0.0000 -7.0487 0.0000 9 0.9238 0.9238 0.8979 0.8979 10 -17.8500 14.5833 15.2080 391.7454 11 0.8832 -6.9214 0.0000 0.8832 -35.7000 15.0000 15.2080 399.6873 0.8760 -6.9166 0.0000 0.8760 12 13 -53,5500 15.4167 14.1663 382.0205 0.8746 -6.3868 0.0000 0.8746 -71.4000 15.4167 13.1250 348.5149 0.8612 -5.8033 0.0000 14 0.8612 15 -89.5200 15.4167 11.8750 318.6611 0.8703 -5.3806 0.0000 0.8703 0.8567 16 -107.1000 15.0000 17 -124.9500 14.5830 11.0413 283.7609 9.9996 246.7169 0.0000 0.0000 -4.9000 0.8567 0.8459 -4.4404 0.8459 18 -142.8000 14.1670 8.9580 212.9712 0.8391 -3.9494 0.0000 0.8391 19-160.650013.75007.9163183.41390.842520-178.500012.91676.6666165.01610.9582 0.8425 -3.4852 0.0000 -3.2250 0.0000 0.9582 OTHE FOLLOWING STATIONS AND OFFSETS ARE USED FOR TWO-DIMENSIONAL CALCULATIONS STA NO= 1 XAXIS= 160.6500 NT= 5 ILID= 0 0.0000 -13.1250 0.6667 -11.4580 1.3333 -7.7080 1.6667 -3.7500 1.8750 0.0000 STA NO= 2 XAXIS= 142.8000 NT= 7 ILID= 0 0.0000 -13.1250 1.0417 -12.2913 1.6667 -11.4580 2.5000 -9.4997 2.9583 -7.7080 3.4167 -3.7500 0.0000 3.5417 STA NO= 3 XAXIS= 124.9500 NT= 8 ILID= 0 0.0000 -25.4163 1.4583 -24.5830 2.5833 -11.4580 3.0833 -9.4997 -7.7080 4.5830 -9.3750 5.0000 5.2083 -3.7500 5.4166 0.0000 STA NO= 4 XAXIS= 107.1000 NT= 7 ILID= 0 0.0000 -13.7500 1.8750 -13.4580 3.3333 -12.7080 4.5834 -11.4580 6.6667 -7.7080 -3.7500 7.0830 7.0830 STA NO= 5 XAXIS= 89.2500 NT= 7 ILID= 0 0.0000 -14.7499 3.1250 -13.4580 4.5833 -9.4997 5.9167 -11.4580

7.9167 -19.5830 8.5417 -3.7500 8.5417 0.0000 STA NO= 6 XAXIS= 0.0000 -14.1666 4.1667 -13.4580 6.0417 -9.4997 7.2917 -11.4580	71.4000	NT=	8	ILID=	0
8.7500 -9.3750 9.3750 -7.7080 10.0000 -3.7500 10.0283 0.0000 STA NO= 7 XAXIS= 0.0000 -14.7910 2.9167 -14.1666 5.8333 -13.1250 8.3330 -11.4580	53.5500	NT=	8	ILID=	0
10.0000 -9.3750 10.8330 -7.7080 11.2500 -3.7500 11.4583 0.0000 STA NO= 8 XAXIS= 0.0000 -14.7910 2.9167 -13.1250 5.4583 -11.9583 9.5833 -11.4580	35.7000	NT=	8	ILID=	0
11.2500 -9.3750 12.0830 -7.7080 12.5000 -3.7500 12.5000 0.0000 STA NO= 9 XAXIS= 0.0000 -15.0000 2.9167 -14.7910 11.4583 -14.7500 10.8333 -11.4580	17.8500	NT=	8	ILID=	0
12.0830 -9.3750 12.9167 -7.7080 13.5416 -3.7500 13.3330 0.0000 STA NO= 10 XAXIS= 0.0000 -15.2080 2.9167 -15.2080 6.2500 -15.0000 10.0000 -14.5830	0.0000	NT=	9	ILID=	0
10.4167 -13.0410 12.9167 -9.3750 13.7500 -7.7080 14.1667 -3.7500 14.1667 0.0000 STA NO= 11 XAXIS= 0.0000 -15.2080 2.9167 -15.2080 6.2500 -15.0000	-17.8500	NT=	8	ILID=	0
10.4167 -13.0410 13.5417 -9.3750 14.1667 -7.7080 14.5833 -3.7500 14.5833 0.0000 STA NO= 12 XAXIS= 0.0000 -15.2080 5.6250 -15.2080 10.8330 -12.9166	-35.7000	NT=	7	ILID=	0
12.5000 -11.4580 14.1667 -7.7080 15.0000 -3.7500 15.0000 0.0000 STA NO= 13 XAXIS= 0.0000 -14.1663 6.0416 -13.9580	-53.5500	NT=	7	ILID=	0

11.8750 -11.4580 13.5416 -9.5830 14.7916 -7.7080 15.4167 -3.7500 15.4167 0.0000 STA NO= 14 XAXIS= -71.4000 NT= 6 ILID= 0 0.0000 -13.1250 6.0416 -12.4996 11.8750 -10.4163 14.7916 -7.7080 15.4167 -3.7500 15.4167 0.0000 STA NO= 15 XAXIS= -89.5200 NT= 6 ILID= 0 0.0000 -11.8750 4.7917 -11.8750 10.0000 -10.6250 13.9583 -7.7080 15.2083 -3.7500 15.4167 0.0000 STA NO= 16 XAXIS= -107.1000 NT= 6 ILID= 0 0.0000 -11.0413 4.7917 -10.8330 10.0000 -9.3750 12.9167 -7.7080 15.0000 -3.7500 15.0000 0.0000 STA NO= 17 XAXIS= -124.9500 NT= 6 ILID= 0 0.0000 -9.9996 4.7917 -9.7850 9.7916 -8.5413 12.9167 -6.2500 14.1667 -3.7500 14.5830 0.0000 STA NO= 18 XAXIS= -142.8000 NT= 6 ILID= 0 0.0000 -8.9580 4.7917 -8.5413 9.3750 -7.7080 12.5000 -5.8333 13.5417 -3.7500 14.1670 0.0000 STA NO= 19 XAXIS= -160.6500 NT= 6 ILID= 0 0.0000 -7.9163 4.7917 -7.4997 9.3750 -6.6666 12.0830 -5.4166 13.1250 -3.7500 13.7500 0.0000 STA NO= 20 XAXIS= -178.5000 NT= 6 ILID= 0 0.0000 -6.6666 4.7917 -6.6666 9.3750 -6.6666 11.4583 -5.8333 -3.7500 12.9167 12.9167 0.0000 OCOMPARISON OF INPUT AND COMPUTED DATA INPUT DISPL= 2579.7871 XCG= -21.9409 RELATIVE TO ORIGIN AT MIDSHIP HYDROSTSTIC DISPL= 2579.7871 XCG= -21.9409 RELATIVE TO ORIGIN AT MIDSHIP OBLOCK COEFFICIENT= 0.5866 OSTABILITY PARAMETERS LCF = -35.3216 ZCG = -2.0000 ZCB = -6.0841 WATERPLANE AREA = 8096.8198 BML = 659.4050 GML = 655.3209 BMT = 5.4405 GMT = 1.3565 ORADII OF GYRATION KYY = 89.2500 KXX = 9.9167 PRODUCT OF INERTIA(146) = 0.788154E+07 1Model 4787

CONDITIONAL INPUT DATA

WAVE AMPLITUDE= 1.0000 INITIAL WAVELENGTH= 20.0000 FINAL WAVELENGTH= 1000.0000 DELTA WAVELENGTH= 20.0000 INITIAL VEL= 0.0000 FINAL VEL= 0.0000 DELTA VEL= 0.0000 INITIAL WAVE HEADING ANGLE= 0.0000 FINAL WAVE HEADING ANGLE= 0.0000 DELTA WAVE HEADING ANGLE= 0.0000 ***UNITS OF OUTPUT*** NONDIMENSIONAL OUTPUT OUTPUT IS DIVIDED BY: LINEAR MOTIONS - WAVE AMPLITUDE (WA) ROTATIONAL MOTIONS - WAVE SLOPE (WA*WAVEN) VELOCITIES - WA*SQRT(GRAV/BPL) ACCELERATIONS - WA*GRAV/BPL SHEAR - WA*GAMMA*BEAM*BPL BENDING MOMENTS - WA*GAMMA*BEAM*BPL**2 1Model 4787 0 SPEED = 0.0000WAVE ANGLE = 0.00 DEG. VERTICAL PLANE RESPONSES (NON-DIMENSIONAL) 0 WAVE ENCOUNTER WAVE WAVE/SHIP SURGE HEAVE РІТСН FREQUENCIES LENGTH LENGTH AMPL. PHASE AMPL. PHASE AMPL. PHASE 20.000 0.0560 0.0021 148.8 0.0002 -113.2 0.0000 -118.9 3.17927 3.17927 2.24808 40.000 0.1120 0.0088 -9.6 0.0023 -62.3 0.0003 83.7 2.24808 1.83555 60.000 0.1681 0.0135 173.2 0.0069 -133.3 0.0013 1.83555 124.9 80.000 0.2241 0.0146 -81.0 0.0358 -26.5 0.0089 -136.2 1.58964 1.58964 1.42181 1.42181 100.000 0.2801 0.0272 118.4 0.0805 109.7 0.0087 121.3 1.29793 120.000 1.20165 140.000 0.3361 0.0181 -15.6 0.1635 -132.6 0.0711 0.3922 0.0317 -82.3 0.0938 -53.3 0.0349 1.29793 47.5 -53.3 0.0349 1.20165 -5.5 0.4482 0.0222 -152.2 0.0455 140.5 0.0837 1.12404 1.12404 160.000 -67.2 1.05976 180.000 0.5042 0.0379 143.0 0.1320 160.9 0.1040 -82.4 1.05976 118.1 0.1608 94.2 0.1464 1.00537 1.00537 200.000 0.5602 0.0484 169.3 0.0895 -101.8 0.95859 0.6162 0.0424 174.7 0.0707 -140.6 0.95859 220.000 0.91778 0.91778 240.000 0.6723 0.0325 46.9 0.1049 -179.8 0.0901 171.4 260.000 0.88177 0.88177 146.6 -46.0 0.2046 0.84970 0.84970 280.000 134.8 0.82088 0.82088 300.000 0.8403 0.1336 -44.4 0.0847 -14.3 0.2667 127.9 0.79482 320.000 0.8964 0.1809 -50.8 0.1487 -9.0 0.3256 123.4 0.79482 -55.1 0.2099 -58.3 0.2672 340.000 0.77109 0.77109 0.9524 0.2276 -6.7 0.3803 120.1 0.74936 1.0084 0.2725 -5.4 0.4305 360.000 117.5 0.74936 0.72937 0.72937 380.000 1.0644 0.3151 -60.9 0.3203 -4.5 0.4761 115.5 1.1204 0.3550 1.1765 0.3922 -62.9 0.3692 -64.6 0.4139 0.71091 400.000 -3.8 0.5174 113.8 0.71091 -3.3 0.5548 0.69377 0.69377 420.000 112.3 0.67782 0.67782 440.000 1.2325 0.4268 -66.1 0.4548 -2.9 0.5885 111.0 0.66292 1.2885 0.4587 -67.3 0.4921 0.66292 460.000 -2.6 0.6191 109.9 0.64897 0.64897 480.000 1.3445 0.4882 -68.5 0.5261 -2.3 0.6467 109.0 0.63585 1.4006 0.5155 -69.4 0.5572 -2.1 0.6718 0.63585 500.000 108.1 0.62351 520.000 1.4566 0.5407 -70.3 0.5856 -1.9 0.6945 0.62351 107.3 1.5126 0.5640 1.5686 0.5855 -71.1 0.6115 -71.8 0.6352 0.61185 0.61185 540.000 -1.8 0.7152 106.6 -1.6 0.7341 0.60083 0.60083 560.000 105.9 1.6247 0.6054 -72.4 0.6570 0.59038 0.59038 580.000 -1.5 0.7513 105.3 0.58045 600.000 1.6807 0.6239 -73.0 0.6770 0.58045 -1.4 0.7670 104.7 -73.6 0.6953 -1.3 0.7814 0.57101 0.57101 620.000 1.7367 0.6410 104.2 0.56202 -74.1 0.7123 -1.2 0.7946 1.7927 0.6569 640.000 103.7 0.56202 0.55344 660.000 1.8487 0.6717 -74.5 0.7279 -1.2 0.8068 0.55344 103.3 1.9048 0.6855 0.54524 680.000 -74.9 0.7423 -1.1 0.8180 102.9 0.54524 700.000 1.9608 0.6984 -75.3 0.7556 -1.0 0.8284 0.53739 0.53739 102.5 -75.7 0.7680 0.52988 0.52988 720.000 2.0168 0.7104 -1.0 0.8380 102.1 0.52267 0.52267 740.000 2.0728 0.7216 -76.0 0.7795 -0.9 0.8468 101.8 0.51575 0.51575 760.000 2.1289 0.7321 -76.4 0.7902 -0.9 0.8551 101.4 780.000 -76.6 0.8001 -0.8 0.8627 0.50909 0.50909 2.1849 0.7419 101.1 0.50269 800.000 2.2409 0.7512 -76.9 0.8094 -0.8 0.8698 0.50269 100.8 820.000 2.2969 0.7598 -77.2 0.8180 0.49652 0.49652 -0.8 0.8764 100.5

0.49057 0.48483 0.47929 0.47394 0.46876 0.46374 0.45889 0.45418	0.49057 0.48483 0.47929 0.47394 0.46876 0.46374 0.45889 0.45418	840.000 860.000 880.000 900.000 920.000 940.000 960.000 980.000	2.3529 2.4090 2.4650 2.5210 2.5770 2.6331 2.6891 2.7451	0.7680 0.7757 0.7830 0.7899 0.7964 0.8025 0.8083 0.8138	-77.4 -77.9 -78.1 -78.3 -78.4 -78.6 -78.8	0.8261 0.8337 0.8408 0.8474 0.8537 0.8596 0.8651 0.8703	-0.6 -0.5	0.8826 0.8884 0.8938 0.8989 0.9036 0.9081 0.9123 0.9162	100.3 100.0 99.8 99.6 99.3 99.1 98.9 98.7
0.45418 0.44962	0.45418 0.44962	980.000 1000.000	2.7451 2.8011	0.8138 0.8191	-78.8 -78.9	0.8703 0.8753	-0.5 -0.5	0.9162 0.9200	98.7 98.5

APPENDIX D

MATLAB CODE FOR V_SPEED_LONG.M

```
% Vertical Plane
% Dimensional version (U.S. units)
% Contour plots (heading/wave height)
% Fully developed Pierson-Moskowitz spectrum - Long crested seas
ship=input('Enter ship number = ');
i_Sea=input('Enter 0 for Pierson-Moskowitz spectrum or 1 otherwise = ');
HS =input('Significant Wave Height (feet) = ');
if i_Sea == 0
   omega_m=0.4*sqrt(32.2/HS);
else
    T_m=input('Enter modal period (sec) = ');
    omega_m=2*pi/T_m;
end
warning off
÷
lambda_min =20;
                                                           % Min wave length (ft)
lambda_max =1000;
                                                           % Max wave length (ft)
delta_lambda=20;
                                                           % Wave length increment (ft)
rho
           =1.9905i
                                                           % Water density
           =1;
                                                           % Regular wave height
zeta
L
            =357;
                                                           % Reference length for
nondimensionalization
                                                           % Ship model
           =num2str(ship);
shipname
beta_incr
           =15;
                                                            % Increment in sea direction (deg)
                                                            % Vector of wavelengths
lambda
           = lambda_min:delta_lambda:lambda_max;
wavenumber = 2.0*pi./lambda;
                                                            % Wave number
            = 32.2;
g
omega
           = sqrt(wavenumber*g);
                                                            % Wave frequency
xarm
           = -0.25*L;
                                                             % Point for motion calculation
           = 2.0*pi./omega;
period
filesize
          = size(lambda);
         = 0;
V_min
V_max
           = 100;
delta V
          = 10;
ò
% Set up file reading format.
°
iSpeed=0;
for V=V_min:delta_V:V_max,
                                % Loop on speed
    iSpeed=iSpeed+1;
    V_string
              =num2str(V);
    ibeta=0;
    for beta=0:beta_incr:360, % Loop on sea direction
        if beta>180
           beta=360-beta;
        end
        beta_string=num2str(beta);
        if beta < 100
            beta_string=strcat('0',num2str(beta));
        end
        if beta < 10
           beta_string=strcat('00',num2str(beta));
        end
        ibeta=ibeta+1;
        trigg = 27;
        f3loc = 26; f5loc=27;
        if beta==0
            trigg = 27;
            f3loc = 26; f5loc=27;
        elseif beta==180
        trigg = 27;
        f3loc = 26; f5loc=27;
        end
        % Load ship data file msvhV_beta.txt
```

```
load_filename=strcat(shipname,'_',V_string,'_',beta_string,'.txt');
filename=load(load_filename);
% GENERAL DATA
÷
          = omega-wavenumber*V*cos(beta*pi/180);
                                                            % Frequency of encounter
omegae
          = 2.0*pi./omegae;
periode
omegae
          = omegae';
lambda_size= trigg*filesize(2);
% VERTICAL PLANE RESPONSE CALCULATIONS
ŝ
% Set mass matrix elements
M33=filename(3:trigg:lambda_size,3);
M35=filename(3:trigg:lambda_size,5);
M53=filename(5:trigg:lambda_size,3);
M55=filename(5:trigg:lambda_size,5);
°
% Added mass terms
A33=filename(9:trigg:lambda_size,3);
A35=filename(9:trigg:lambda_size,5);
A53=filename(11:trigg:lambda_size,3);
A55=filename(11:trigg:lambda_size,5);
% Damping terms
B33=filename(15:trigg:lambda_size,3);
B35=filename(15:trigg:lambda_size,5);
B53=filename(17:trigg:lambda_size,3);
B55=filename(17:trigg:lambda_size,5);
÷
% Hydrostatic terms
C33=filename(21:trigg:lambda_size,3);
C35=filename(21:trigg:lambda_size,5);
C53=filename(23:trigg:lambda_size,3);
C55=filename(23:trigg:lambda_size,5);
% Total exciting forces
F3_t_amp=filename(f3loc:trigg:lambda_size,5);
F5_t_amp=filename(f5loc:trigg:lambda_size,5);
F3_t_pha=filename(f3loc:trigg:lambda_size,6);
F5_t_pha=filename(f5loc:trigg:lambda_size,6);
        =F3_t_amp.*exp(i*F3_t_pha.*pi/180.0);
F3 t
F5_t
        =F5_t_amp.*exp(i*F5_t_pha.*pi/180.0);
2
% Froude/Krylov exciting forces
ò
F3_f_amp=filename(f3loc:trigg:lambda_size,1);
F5_f_amp=filename(f5loc:trigg:lambda_size,1);
F3_f_pha=filename(f3loc:trigg:lambda_size,2);
F5_f_pha=filename(f5loc:trigg:lambda_size,2);
F3_f
       =F3_f_amp.*exp(i*F3_f_pha.*pi/180.0);
F5_f
        =F5_f_amp.*exp(i*F5_f_pha.*pi/180.0);
2
% Diffraction exciting forces
ŝ
F3_d_amp=filename(f3loc:trigg:lambda_size,3);
F5_d_amp=filename(f5loc:trigg:lambda_size,3);
F3_d_pha=filename(f3loc:trigg:lambda_size,4);
F5_d_pha=filename(f5loc:trigg:lambda_size,4);
F3_d=F3_d_amp.*exp(i*F3_d_pha.*pi/180.0);
F5_d=F5_d_amp.*exp(i*F5_d_pha.*pi/180.0);
% Calculate Ship Motions
ŝ
```

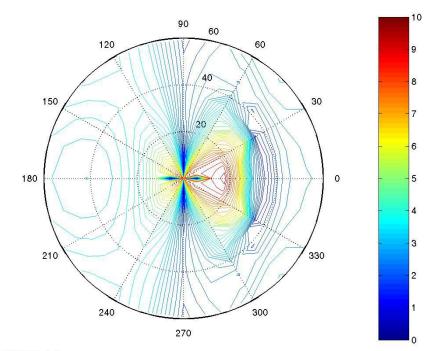
```
A33bar=-(omegae.^2).*(M33+A33)+i*omegae.*B33+C33;
        A35bar=-(omegae.^2).*(M35+A35)+i*omegae.*B35+C35;
        A53bar=-(omegae.^2).*(M53+A53)+i*omegae.*B53+C53;
        A55bar=-(omegae.^2).*(M55+A55)+i*omegae.*B55+C55;
        °
        eta3=(A55bar.*F3_t-A35bar.*F5_t)./(A33bar.*A55bar-A35bar.*A53bar);
        eta5=(A53bar.*F3_t-A33bar.*F5_t)./(A53bar.*A35bar-A33bar.*A55bar);
        xi=eta3-eta5*xarm;
        ŝ
        % Random wave calculations
        ÷
        A=(1.25/4)*(omega_m^4)*(HS^2);
        B=1.25*omega_m^4;
              =(A./omega.^5).*exp(-B./omega.^4);
        S
        Se
              =S./abs((1-(2.0/g)*omega*V*cos(beta*pi/180)));
                                                                       % Convert S(w) to S(we)
        °
        % Define response spectra
        8
        Sxi =((abs(xi)).^2).*Se';
        Seta3=((abs(eta3)).^2).*Se';
        Seta5=((abs(eta5)).^2).*Se';
        ÷
        % Initializations
        2
        Sxi_i=0;
        Seta3_i=0;
        Seta5_i=0;
        2
        % Integral S(w)*|RAO|^2
        ò
        for I=2:1:filesize(2),
            Sxi_i = Sxi_i + 0.5*(Sxi(I) + Sxi(I-1)) * abs((omegae(I-1)-omegae(I)));
            Seta3_i= Seta3_i+ 0.5*(Seta3(I)+ Seta3(I-1))* abs((omegae(I-1)-omegae(I)));
            Seta5_i= Seta5_i+ 0.5*(Seta5(I)+ Seta5(I-1))* abs((omegae(I-1)-omegae(I)));
        end
        ÷
        % RMS values
        °
        RMS_xi(ibeta,iSpeed) = sqrt(Sxi_i);
        RMS_eta3(ibeta,iSpeed) = sqrt(Seta3_i);
        RMS_eta5(ibeta,iSpeed) = sqrt(Seta5_i);
        ŝ
    end
% Contour plots
conv=1.6878;
Vp_min=V_min/conv;
Vp max=V max/conv;
delta_Vp=delta_V/conv;
figure(1)
[th,r]=meshgrid((0:beta_incr:360)*pi/180,Vp_min:delta_Vp:Vp_max);
[X,Y]=pol2cart(th,r);
h=polar(th,r);delete(h);
hold on
contour(X',Y',RMS_eta5/HS,c_p),caxis([0 10]),colorbar
figure(2)
[th,r]=meshgrid((0:beta_incr:360)*pi/180,Vp_min:delta_Vp:Vp_max);
[X,Y]=pol2cart(th,r);
h=polar(th,r);delete(h);
hold on
c_p=[0:0.2:10];
contour(X',Y',RMS_eta3/HS,c_p),caxis([0 10]),colorbar
```

end ÷

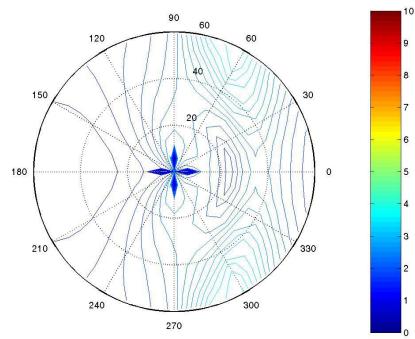
÷

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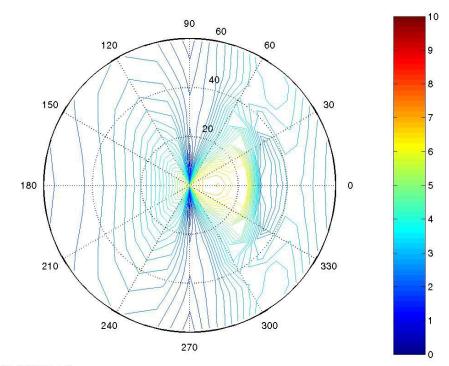
PIERSON – MOSKOWITZ CONTOUR PLOTS



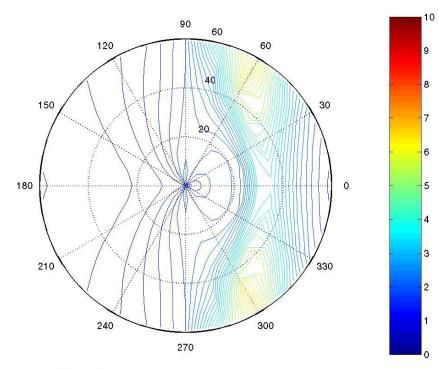
Model 4787 (PM) Pitch Response



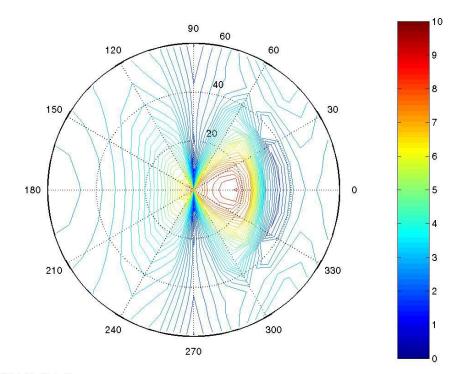
Model 4787 (PM) Heave Response



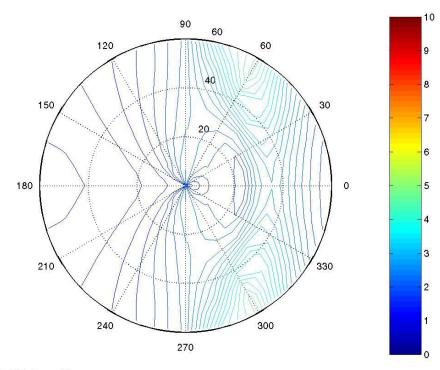
Model 4794 (PM) Pitch Response



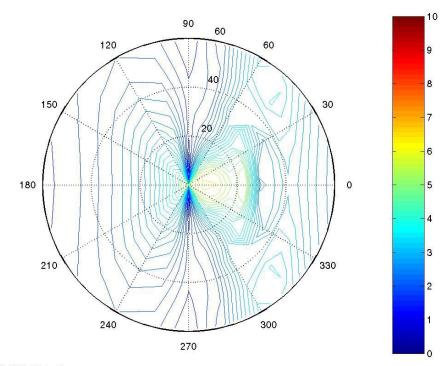
Model 4794 (PM) Heave Response



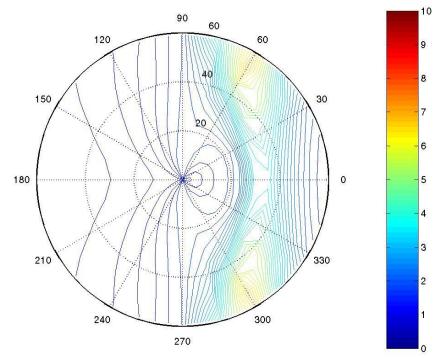
Model 4796 (PM) Pitch Response



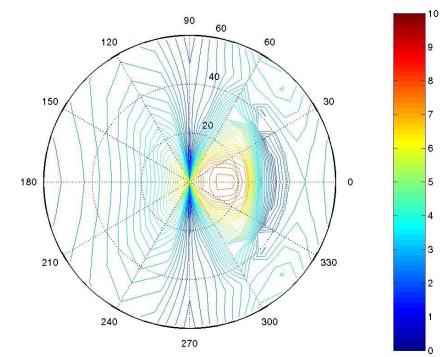
Model 4796 (PM) Heave Response



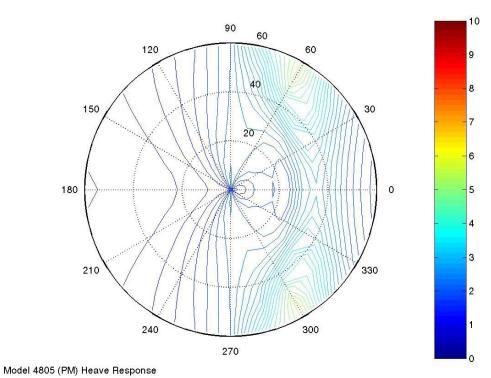
Model 4802 (PM) Pitch Response



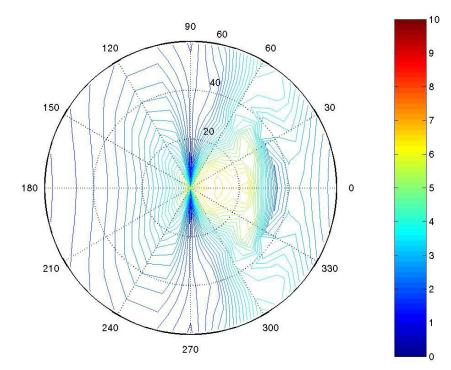
Model 4802 (PM) Heave Response



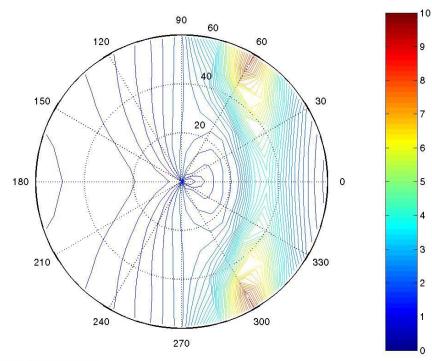
Model 4805 (PM) Pitch Response



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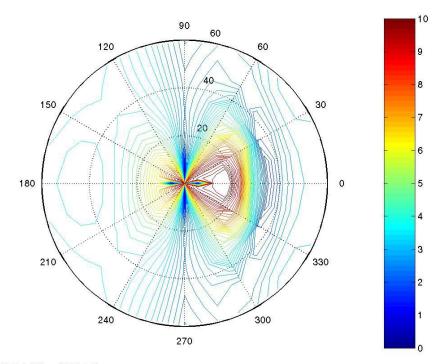


Model 4812 (PM) Pitch Response

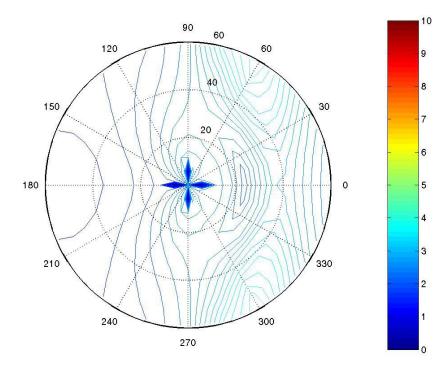


Model 4812 (PM) Heave Response

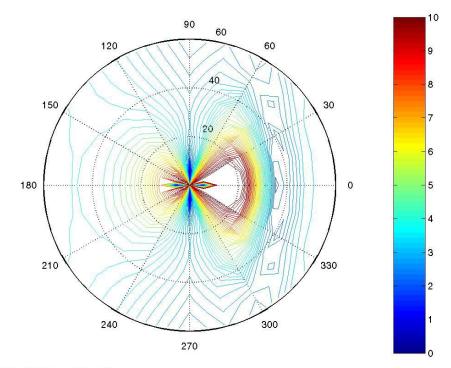
BRETSCHNEIDER CONTOUR PLOTS



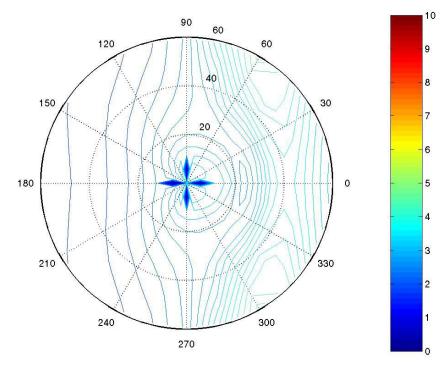
Model 4787 (B: 9.8 sec) Pitch Response



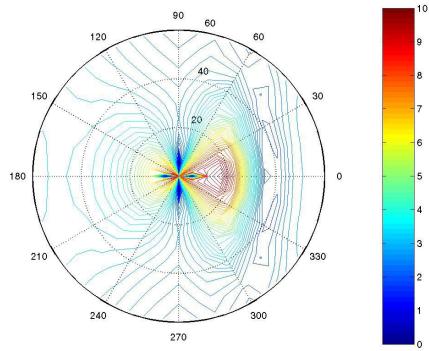
Model 4787 (B: 9.8 sec) Heave Response



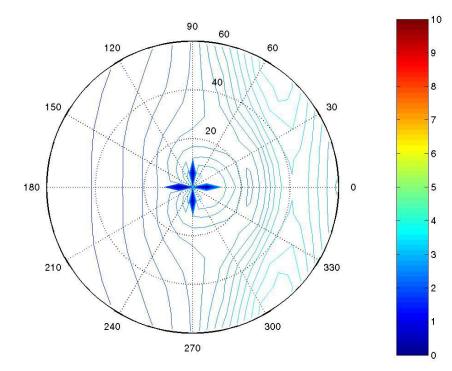
Model 4787 (B: 13.0 sec) Pitch Response



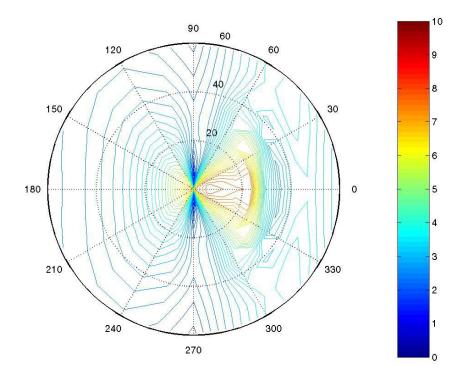
Model 4787 (B: 13.0 sec) Heave Response



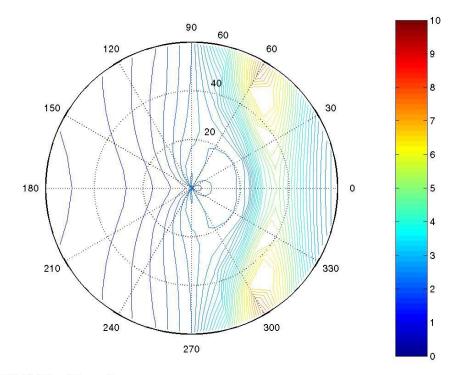
Model 4787 (B: 16.2 sec) Pitch Response



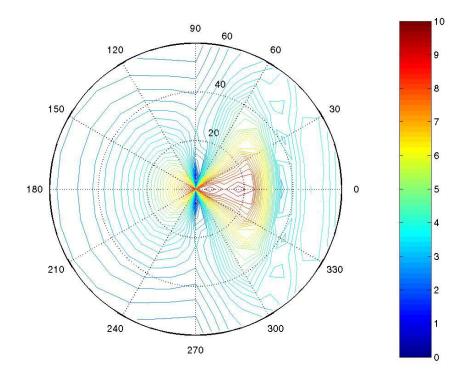
Model 4787 (B: 16.2 sec) Heave Response



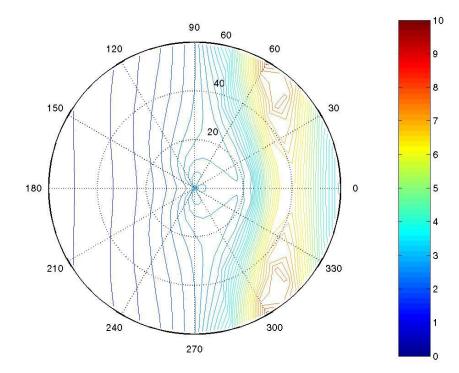
Model 4794 (B: 9.8 sec) Pitch Response



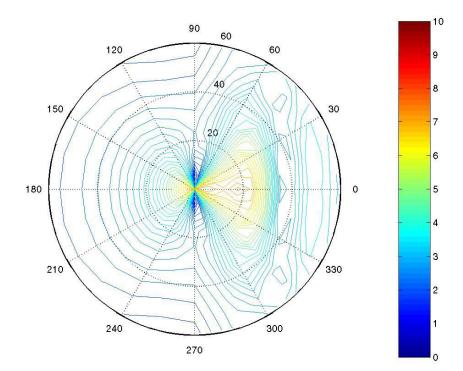
Model 4794 (B: 9.8 sec) Heave Response



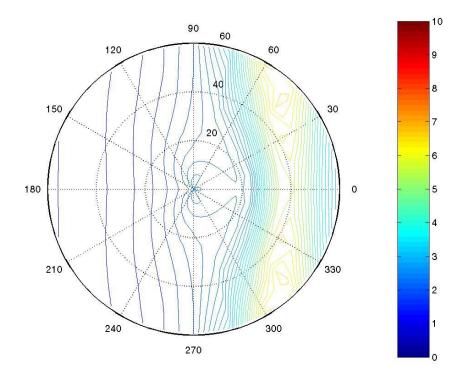
Model 4794 (B: 13.0 sec) Pitch Response



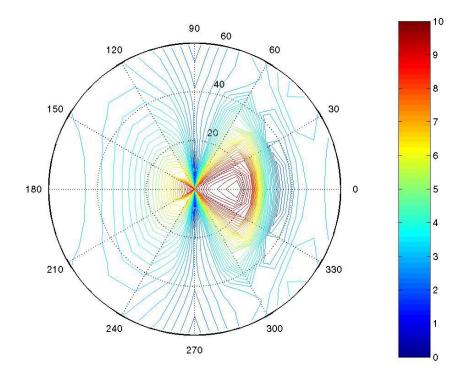
Model 4794 (B: 13.0 sec) Heave Response



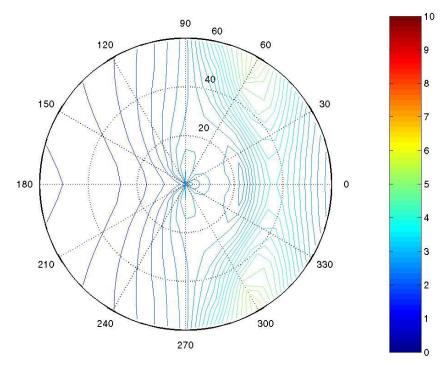
Model 4794 (B: 16.2 sec) Pitch Response



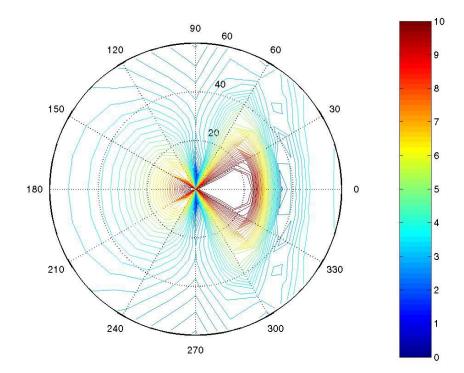
Model 4794 (B: 16.2 sec) Heave Response



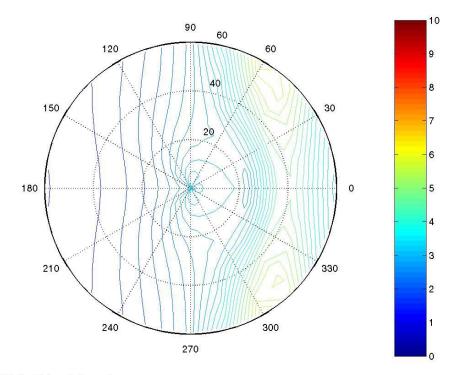
Model 4796 (B: 9.8 sec) Pitch Response



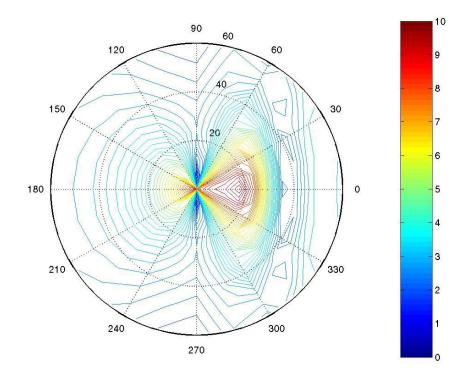
Model 4796 (B: 9.8 sec) Heave Response



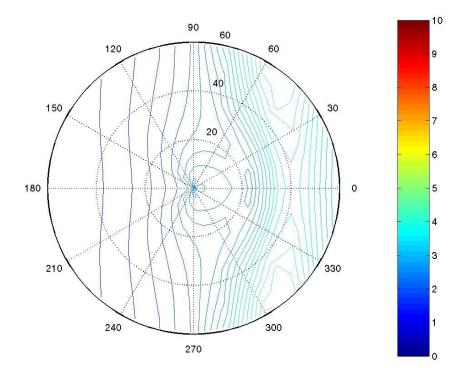
Model 4796 (B: 13.0 sec) Pitch Response



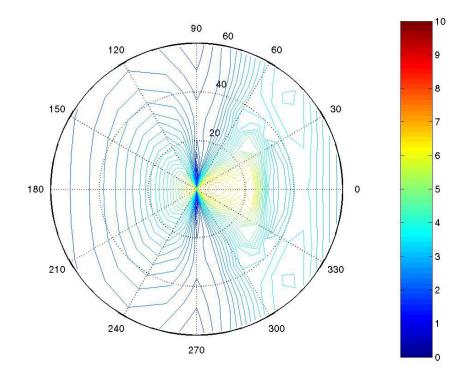
Model 4796 (B: 13.0 sec) Heave Response



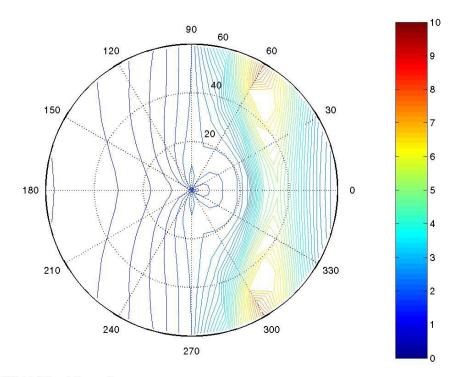
Model 4796 (B: 16.2 sec) Pitch Response



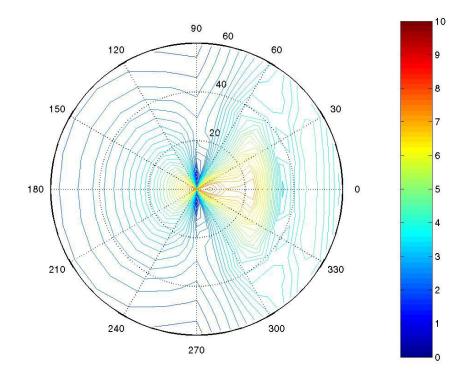
Model 4796 (B: 16.2 sec) Heave Response



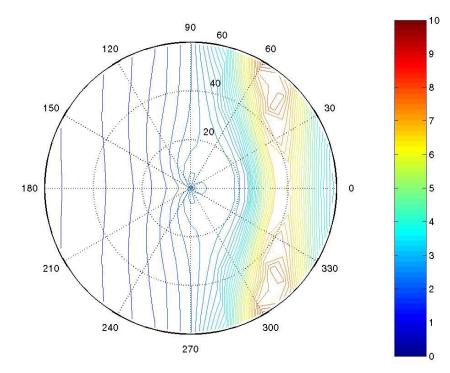
Model 4802 (B: 9.8 sec) Pitch Response



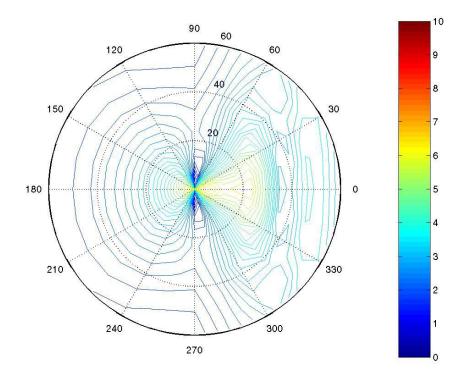
Model 4802 (B: 9.8 sec) Heave Response



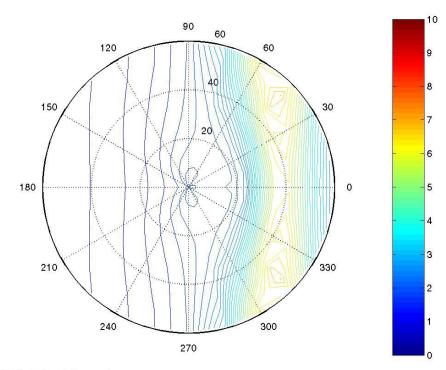
Model 4802 (B: 13 sec) Pitch Response



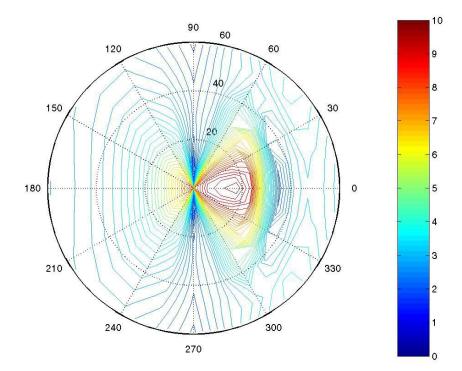
Model 4802 (B: 13.0 sec) Heave Response



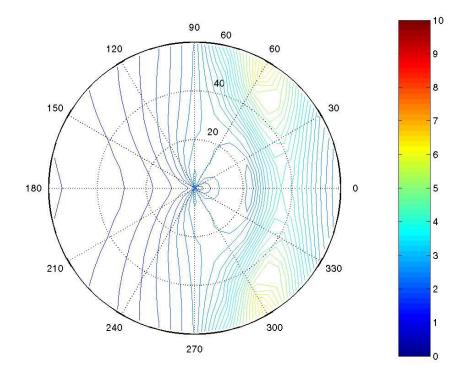
Model 4802 (B: 16.2 sec) Pitch Response



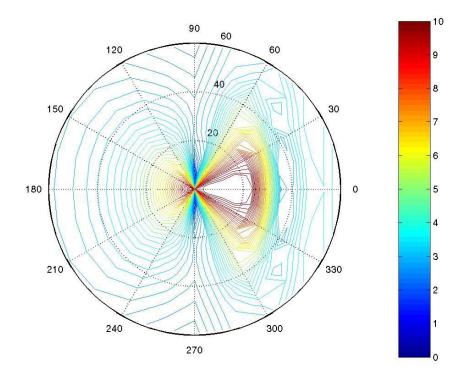
Model 4802 (B: 16.2 sec) Heave Response



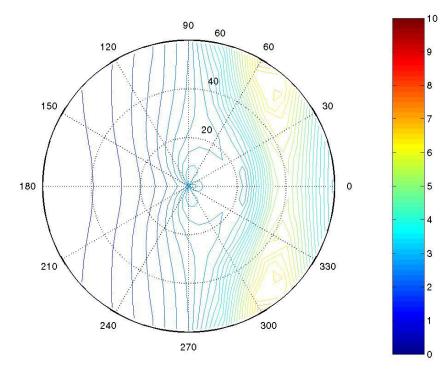
Model 4805 (B: 9.8 sec) Pitch Response



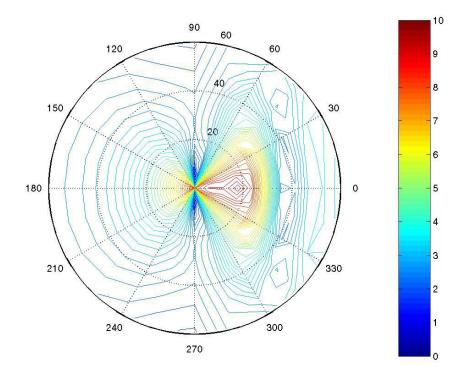
Model 4805 (B: 9.8 sec) Heave Response



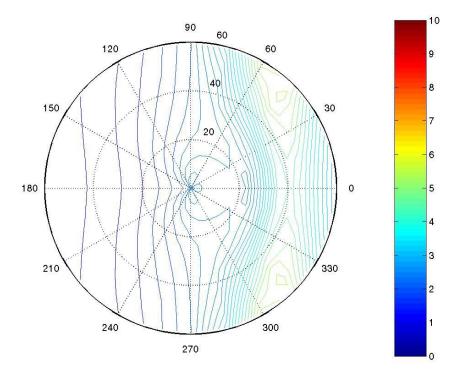
Model 4805 (B: 13.0 sec) Pitch Response



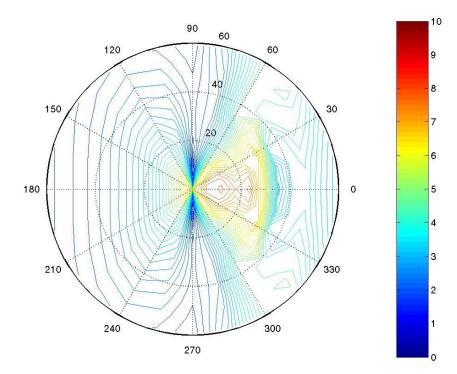
Model 4805 (B: 13.0 sec) Heave Response



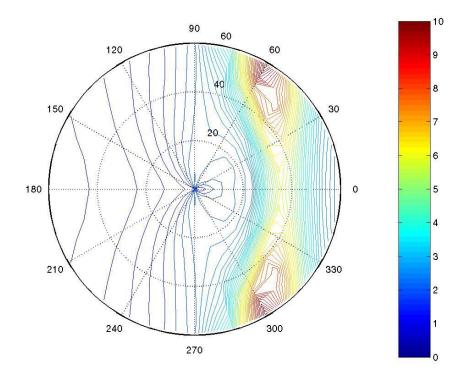
Model 4805 (B: 16.2 sec) Pitch Response



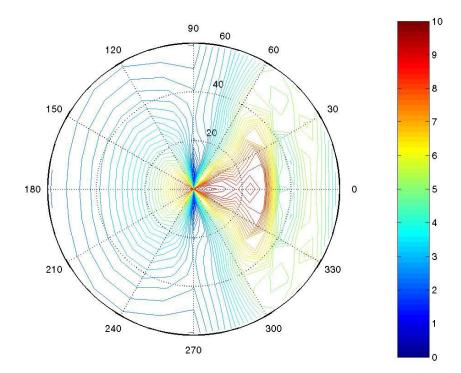
Model 4805 (B: 16.2 sec) Heave Response



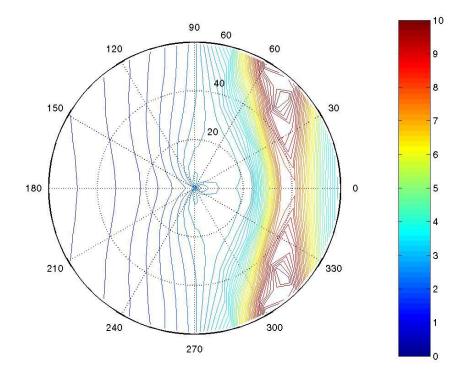
Model 4812 (B: 9.8 sec) Pitch Response



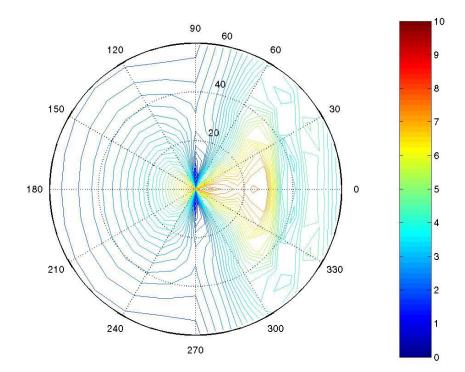
Model 4812 (B: 9.8 sec) Heave Response



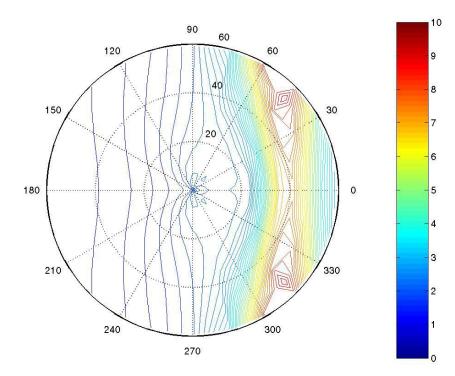
Model 4812 (B: 13.0 sec) Pitch Response



Model 4812 (B: 13.0 sec) Heave Response



Model 4812 (B: 16.2 sec) Pitch Response



Model 4812 (B: 16.2 sec) Heave Response

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