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## THESIS

## HOPF BIFURCATION ANALYSIS FOR DEPTH <br> CONTROL OF SUBMERSIBLE VEHICLES

by
Craig A. Bateman
December, 1993

Thesis Advisor:
Fotis A. Papoulias

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| REPORT DOCUMENTATION PAGE |  |  | Form Approved OMB No. 3704 |  |
| :---: | :---: | :---: | :---: | :---: |
| Public reporting burdan for this collection of information is estimated to average 1 bour per response, including the time for reviewing instruction, marching existing deta sources, gatbering and maintaining the data needed, and complecing and reviewing the collection of information. Send comments regarding this burden eatimate or any other aspect of this collection of information, including suggestions for reducing this burden, to Washington beadquarters Services, Directorate for Information Operations and Reports, 1215 Jefferson Davis Highway, Suite 1204, Arlington, VA 22202-4302, and to the Office of Management and Budger, Paperwork Reduction Project (0704-0188) Washingtion DC 20503. |  |  |  |  |
| 1. AGENCY USE ONLY | 2. REPORT DA 30 November |  | 3. REPORT TYPE AND DATES COVERED Master's Thesis |  |
| 4. TITLE AND SUBTITLE HOPF BIFURCATION ANALYSIS FOR DEPTH CONTROL OF SUBMERSIBLE VEHICLES |  |  | 5. FUNDING NUMBERS |  |
| 6. AUTHOR(S) BATEMAN, Craig A. |  |  |  |  |
| 7. PERFORMING ORGANIZATION NAME(S) AND ADDRESS(ES) Naval Postgraduate School Monterey, CA 93943-5000 |  |  | 8. PERFORMING ORGANIZATION REPORT NUMBER |  |
| 9. SPONSORING/MONITORING AGENCY NAME(S) AND ADDRESS(ES) |  |  | 10. SPONSORING/MONTTORING AGENCY REPORT NUMBER |  |
| 11. SUPPLEMENTARY NOTES The views expressed in this thesis are those of the author and do not reflect the official policy or position of the Department of Defense or the U.S. Government. |  |  |  |  |
| 12a. DISTRIBUTION/AVAILABILITY STATEMENT Approved for public release; distribution is unlimited. |  |  | 12b. DISTRIBUTION CODE *A |  |
| 13. <br> ABSTRACT <br> Control of a modern submarine is a multi-dimensional problem coupling initial stability, hydrodynamic and control system response. The loss of stability at moderate to high speeds is examined using a nonlinear Hopf bifurcation analysis. Complete linear state feedback is used for demonstration purposes for depth control at level attitude and for a fixed nomiral speed. Control time constant, nominal and actual speeds, metacentric height, and stern to bow plane deflection ratio are used as the main bifurcation parameters. A complete local bifurcation mapping provides a systematic method for evaluating the bounds of controllability for control system design parameters for a submarine with a given set of hydrodynamic coefficients. The submarine and its potential design modifications are then verified with a nonlinear simulation program. |  |  |  |  |
| 14. SUBJECT TERMS <br> Submarine, bifurcation, nonlinear dynamics, chaos |  |  |  | 15. <br> NUMBER OF <br> PAGES 94 |
|  |  |  |  | 16. PRICE CODE |
| 17. SECURITY CLASSIFICATION OF REPORT Unclassified | 18. SECURITY CLASSIFICATION OF THIS PAGE Unclassified | 19. SECURIT CATION Unclassif | ASSIFI- <br> BSTRACT | 20. LIMITATION OF ABSTRACT UL |

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## HOPF BIFURCATION ANALYSIS FOR DEPTH CONTROL OF SUBMERSIBLE VEHICLES

by

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Submitted in partial fulfillment
of the requirements for the degree of

MASTER'S OF SCIENCE IN MECHANICAL ENGINEERING
from the

NAVAL POSTGRADUATE SCHOOL
December 1993

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Approved by:



#### Abstract

Control of a modern submarine is a multi-dimensional problem coupling initial stability, hydrodynamic and control system response. The loss of stability at moderate to high speeds is examined using a nonlinear Hopf bifurcation analysis. Complete linear state feedback is used for demonstration purposes for depth control at level attitude and for a fixed nominal speed. Control time constant, nominal and actual speeds, metacentric height, and stern to bow plane ratio are used as the main bifurcation parameters. A complete local bifurcation mapping provides a systematic method for evaluating the bounds of controllability for control system design parameters for a submarine with a given set of hydrodynamic coefficients. The submarine and its potential design modifications are then verified with a nonlinear simulation program.



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| Symbol | Definition |
| :--- | :--- |
| A | closed loop dynamics matrix <br> system |
| a for the linearized |  |

## ACINOWLEDGEMENTT

First and foremost, I would like to give my deepest thanks to my wife, Sara, and my children, David and Kristen, for their patience and support during our two and a half year separation. There is no way that I can repay their sacrifice.

Second, my thanks to LT. Mary M. Zurowski, USN for her careful editing and proofreading of this thesis. Hers was a difficult task.

Finally, my highest appreciation to Professor Fotis A. Papoulias, for his expert guidance and excellent instruction during the course of this thesis.

## I. INTRODUCTION

The fundamental goal of submarine control is to reach and maintain ordered depth. Any design that coes not meet this goal, in the face of the inherent complexities, is not overly useful as a practical vessel. Current evaluation schemes involve extensive model testing such as those done on the DARPA SUBOFF Model (DTRC Model 5470) [ref. 8]. This is an expensive and time consuming evaluation method. The goal is to develop an analytic method to determine the stability of a potential design. Much work has been done on depth control and modelling of submarines in the vertical plane [ref's. 1, 2, \& 3] but this work has assumed the existence of a stable platform.

The stability of a design will have a significant impact on its responsiveness. A vehicle with a large margin to instability will not be very responsive. The problem becomes one of determining how close to the margins we can get without a total loss of stability. Nonlinear dynamics and chaos provides us with the tools for solving this problem [ref's 9, \& 10]. By expanding the scope of our research to include nonlinear terms we are able to define the limits of stability and therefore the margins. At the Naval Postgraduate School there has been extensive work on defining the nature of the instabilities that are encountered in the control of
submersibles in path keeping and the dive plane [ref's 4, 5, \& 6].

At low forward velocities a submersible using stern planes for attitude and depth control can experience a loss of stability in the form of stern planes reversal. This is a pitchfork bifurcation that can be predicted [ref. 7] and can be accounted for in the control design. The purpose of this thesis is to develop a program for finding the limits of stability for a submarine at moderate and high speeds. Once these limits are mapped then the nature of the loss of stability must then be determined. For this we use a Hopf bifurcation analysis along with a nonlinear simulation program. Finally, after the limits are determined we are able to define the control system design parameters and evaluate the controllability of the design.

## II. PROBLEM FORMULATION

## A. EOUATIONS OF MOTION

By restricting the submersible's motion to the vertical, or dive plane, the motion can be modeled by the coupled nonlinear equations for pitch and heave. With a body fixed coordinate frame at the vehicles geometric center, we can express Newton's equations of motion as

$$
\begin{align*}
& m\left(\dot{w}-U q-z_{G} q^{2}-x_{G} \dot{q}\right)= z_{q} \dot{q}+Z_{\dot{w}} \dot{w}+Z_{q} U q+Z_{w} U w \\
&-C_{D} \int_{T A I L}^{N O S E} b(x) \frac{(w-x q)^{3}}{|w-x q|} d x  \tag{2.1}\\
&(W-B) \cos \theta+U^{2}\left(z_{\delta_{s}} \delta_{s}+Z_{\delta_{b}} \delta_{b}\right)
\end{align*}
$$

and

$$
\begin{align*}
& I_{y} \dot{q}+m z_{G} w q-m x_{G}(\dot{w}-U q)=M_{q} \dot{q}+M_{\dot{w}} \dot{w}+M_{q} U q+M_{w} U w \\
& +C_{D} \int_{T A I L}^{N O S E} b(x) \frac{(w-x q)^{3}}{|w-x q|} x d x  \tag{2.2}\\
& -\left(x_{G} W-x_{B} B\right) \cos \theta-\left(z_{G} W-z_{B} B\right) \sin \theta \\
& +U^{2}\left(M_{\delta_{s}} \delta_{s}+M_{\delta_{b}} \delta_{b}\right) .
\end{align*}
$$

. or simplicity we will assume that the drag coefficient, $C_{D}$, is constant over the length of the submersible. This does not have a significant effect on the results.

The pitch rate and depth rate for a submersible is given by

$$
\begin{gather*}
\theta=y,  \tag{2.3}\\
\dot{z}=-U \sin \theta+w \cos \theta,
\end{gather*}
$$

respectively. In equations (2.3) and (2.4) $\theta$ is the pitch angle of the vehicle as meas'ured from the horizontal plane. Figure (2.1) shows the geometry and definitions for most of the symbols. We assume that forward velocity, $U$, is maintained constant during maneuvers by an appropriate use of the propulsion system.
B. CONTROL LAW

Equations (2.1) through (2.4) can be written as a set of nonlinear differential equations in the form,

$$
\begin{equation*}
\hat{\theta}=q, \tag{2.5}
\end{equation*}
$$

$$
\begin{gather*}
\dot{w}=a_{11} U w+a_{12} U q+a_{13} z_{G B} \sin \theta+b_{11} U^{2} \delta_{s}+b_{12} U^{2} \delta_{b}  \tag{2.6}\\
+d_{w}(w, q)+c_{1}(w, q),
\end{gather*}
$$

$$
\begin{gather*}
\dot{q}=a_{21} U W+a_{22} U q+a_{23} z_{G B} \sin \theta+b_{21} U^{2} \delta_{s}+b_{22} U^{2} \delta_{b}  \tag{2.7}\\
+d_{q}(w, q)+c_{2}(w, q),
\end{gather*}
$$

$$
\begin{equation*}
\dot{z}=-U \sin \theta+w \cos \theta, \tag{2.8}
\end{equation*}
$$

where,

$$
\begin{align*}
& D_{v}=\left(m-Z_{i}\right)\left(I_{y}-M_{q}\right)-\left(m x_{G}+Z_{q}\right)\left(m x_{G}+M_{\dot{w}}\right), \\
& a_{11} D_{v}=\left(I_{y}-M_{q}\right) Z_{w}+\left(m x_{G}+Z_{q}\right) M_{w} \text {, } \\
& a_{12} D_{v}=\left(I_{y}-M_{q}\right)\left(m+Z_{q}\right)+\left(m x_{G}+Z_{q}\right)\left(M_{q}-m x_{G}\right), \\
& a_{13} D_{v}=-\left(m x_{G}+Z_{q}\right) W \text {, } \\
& b_{11} D_{v}=\left(I_{y}-M_{q}\right) Z_{\delta_{j}}+\left(m x_{G}+Z_{q}\right) M_{\delta_{b}} \text {, } \\
& b_{12} D_{v}=\left(I_{y}-M_{q}\right) Z_{\delta_{b}}+\left(m x_{G}+Z_{\phi}\right) M_{\delta_{b}} \text {, } \\
& a_{21} D_{v}=\left(m-Z_{\dot{w}}\right) M_{w}+\left(m x_{G}+M_{\dot{w}}\right) Z_{w} \text { 。 } \\
& a_{22} D_{v}=\left(m-Z_{\dot{w}}\right)\left(M_{q}-m x_{G}\right)+\left(m x_{G}+M_{\dot{w}}\right)\left(m+Z_{q}\right),  \tag{2.9}\\
& a_{23} D_{v}=-\left(m-Z_{\dot{w}}\right) W \text {, } \\
& b_{21} D_{v}=\left(m-Z_{\dot{w}}\right) M_{\delta_{g}}+\left(m x_{G}+M_{\dot{w}}\right) Z_{\delta_{g}}, \\
& b_{22} D_{v}=\left(m-Z_{i j}\right) M_{\delta_{b}}+\left(m x_{G}+M_{\dot{w}}\right) Z_{\delta_{b}}, \\
& d_{w}(w, q) D_{v}=\left(I_{y}-M_{q}\right) I_{w}+\left(m x_{G}+Z_{q}\right) I_{q} \text {, } \\
& d_{q}(w, q) D_{v}=\left(m-Z_{\dot{w}}\right) I_{q}+\left(m x_{G}+M_{\dot{w}}\right) I_{w} \text {, } \\
& c_{1}(w, q) D_{v}=\left(I_{Y}-M_{q}\right) m z_{G} q^{2}-\left(m x_{G}+Z_{\dot{q}}\right) m z_{G} w q \text {, } \\
& c_{2}(w, q) D_{v}=-\left(m-Z_{\dot{w}}\right) m z_{G} w q+\left(m x_{G}+M_{\dot{w}}\right) m z_{G} q^{2} .
\end{align*}
$$

In equations (2.5) through (2.8), the vehicle is assumed to be neutrally buoyant $(W=B)$, level $\left(X_{G}=X_{B}\right)$, and statically stable $\left(z_{G}>z_{B}\right)$. The terms $I_{w}$ and $I_{q}$ represent the cross flow drag integrals in equations (2.1) and (2.2), and $z_{G B}=z_{G}-z_{B}$ is the metacentric height. We can assume $z_{B}$ to be zero, therefore $z_{G B}=z_{G}$.

The depth control system uses the linearized form of equations (2.5) through (2.8) with the linearization at the
nominal conditions of level attitude, ordered depth and forward speed. This gives the state space form of the depth control equations as

$$
\begin{gather*}
\theta=q,  \tag{2.10}\\
\dot{w}=a_{11} U_{0} w+a_{12} U_{0} q+a_{13} z_{G B} \theta+b_{1} U_{0}^{2} \delta,  \tag{2.11}\\
\dot{q}=a_{21} U_{0} w+a_{22} U_{0} q+a_{23} z_{G B} \theta+b_{2} U_{0}^{2} \delta,  \tag{2.12}\\
\dot{z}=-U_{0} \theta+w, \tag{2.13}
\end{gather*}
$$

where $U_{0}$ is the nominal speed for gain selection, and the control inputs are defined as

$$
\begin{align*}
& \delta_{s}=\delta, \\
& \delta_{b}=\alpha \delta^{\prime},  \tag{2.14}\\
& b_{1}=b_{11}+\alpha b_{12}, \\
& b_{2}=b_{21}+\alpha b_{22},
\end{align*}
$$

where $\alpha$ is the control surface coordination gain. This produces a linear full state feedback control law of

$$
\begin{equation*}
\delta=k_{1} \theta+k_{2} w+k_{3} q+k_{4} z \tag{2.15}
\end{equation*}
$$

where the gains $k_{1}, k_{2}, k_{3}$, and $k_{4}$ are computed to give the closed loop system, equations (2.10) through (2.13), the
desired dynamics. Since the desired characteristic equation has the general form,

$$
\begin{equation*}
\lambda^{4}+\alpha_{3} \lambda^{3}+\alpha_{2} \lambda^{2}+\alpha_{1} \lambda+\alpha_{0}=0, \tag{2.16}
\end{equation*}
$$

the controller gains are computed by equating coefficients of the actual and desired characteristic equations,

$$
\begin{align*}
& b_{1} U_{0}^{2} k_{2}+b_{2} U_{0}^{2} k_{3}=-\alpha_{3}-\left(a_{11}+a_{22}\right) U_{0}, \\
& b_{2} U_{0}^{2} k_{1}+\left(b_{2} a_{12}-b_{1} a_{22}\right) U_{0}^{3} k_{2}+\left(b_{1} a_{21}-b_{2} a_{11}\right) U_{0}^{3} k_{3}+b_{1} U_{0}^{2} k_{4} \\
& =-\alpha_{2}-a_{23} z_{G B}+\left(a_{11} a_{22}-a_{21} a_{12}\right),  \tag{2.17}\\
& \left(b_{2} a_{11}-b_{1} a_{21}\right) U_{0}^{3} k_{1}+\left(b_{1} a_{23}-b_{2} a_{13}\right) z_{G B} U_{0}^{2} k_{2} \\
& \quad+\left(b_{2}+b_{1} a_{22}-b_{2} a_{12}\right) U_{0}^{3} k_{4}=\alpha_{1}+\left(a_{13} a_{21}-a_{23} a_{11}\right) z_{G B} U_{0}, \\
& {\left[\left(b_{1} a_{21}-b_{2} a_{11}\right) U_{0}^{4}+\left(b_{1} a_{23}-b_{2} a_{213}\right) z_{G B} U_{0}^{2}\right] k_{4}=\alpha_{0}}
\end{align*}
$$

[ref. 7]


Figure (2.1) - Geometrical representations of the basic definitions for the equations of motion.
III. STABILITY

## A. BIFURCATION ANALYSIS

In system dynamics, the classical definition of stability states, that the real part of all the eigenvalues of the system must be negative. Therefore, our initial investigations into the stability of the SUBOFF Model was to find those eigenvalues whose real parts cross the imaginary axis. We used the bifurcation analysis program, included as Appendix $A$, to calculate the eigenvalues of the system.

By linearizing the equations of motion, equations (2.1 through 2.4), the state space equations of the dynamic system can be written in the form

$$
\begin{equation*}
\dot{x}=A x+B u \tag{3.1}
\end{equation*}
$$

where

$$
\begin{equation*}
u=-K x \tag{3.2}
\end{equation*}
$$

and $K$ is the matrix of controller gains, as calculated by pole placement in equation (2.17). The eigenvalues of the system are found by solving

$$
\begin{equation*}
\operatorname{det}|A-B K-s I|=0 \tag{3.3}
\end{equation*}
$$

In the bifurcation analysis program a pseudo-root locus method is employed where the time constant, $T_{C}$, is fixed. The constant $T_{c}$ fixes to placement of the system poles at a given nominal forward speed $U_{0}$ and then the model speed, $U$, is varied incrementally with the system eigenvalues calculated at each speed increment. When the real part of an eigenvalue changes sign between the limits of a speed increment a bisection method is employed to find the speed where the real part of the eigenvalue equals 0 .

For each point where the real part of an eigenvalue crosses the imaginary axis the associated $T_{C}$ and $U$ are plotted on a bifurcation map. This map delineates the regions of classical stability (all eigenvalues on the left hand halfplane) from the regions of instability (at least one eigenvalue in the right hand half-plane). A family of bifurcation maps were generated by varying nominal speed, $U_{0}$, initial stability, $\mathrm{Z}_{\mathrm{GB}}$, and control surface gain, $\alpha$.

## B. TYPICAL RESULTS

Figure (3.1) shows a typical bifurcation map with its five distinct regions. Region $I$ is the area of classical stability. In region II there is one real positive eigenvalue which is indicative of a pitchfork bifurcation. Pitchfork bifurcations of this model were previously examined by Reidel [ref. 7]. Regions III, IV, and V have at least one pair of complex conjugate eigenvalues with a positive real part. This
would indicate that there should be an unstable oscillatory behavior for the model.

## C. SIMULATIONS

An extensive set of simulations were run to verify the bifurcation map's prediction of system stability. While the results of simulations showed the demarcation between the stable and unstable regions, the simulations found that the linear bifurcation analysis failed to predict the method of departure from stability. Five points (a through e) as shown on fig. (3.1) were chosen to illustrate the model's behavior in the regions of interest. Table (3.1) lists the eigenvalues found at each of these points and Table (3.2) shows the eigenvalues associated with the exact bifurcation point near points (b through d). Note that the eigenvalues are given in dimensional terms while all other information in the tables are non-dimensionalized.

Point a is in the region of stability and fig (3.2) shows a rapid convergence to nominal stability. Simulations run at points b, $c$, and $d$ are shown on figures (3.3), (3.4), and (3.5). These points are spaced less than 2.2 kts apart but they have a huge difference in their bounded oscillatory behavior. Table (3.1) also lists the measured and theoretical periods for these three points. The theoretical period is computed by dividing $2 \pi$ by the imaginary part of the eigenvalue with the largest real part, and then non-
dimensionalize it by the ratio $U / L$, where $U=$ speed of the vehicle (in feet per second) at each point, and $L=$ vehicle length (13.9792 ft). An interesting point to note is the behavior at point $d$, where the measured period is approximately one half of the theoretical period. The dominant period at 80 nd sec is associated with the creation of the limit cycle at the bifurcation point while the 155.4 nd sec period is part of the chaotic response found at point $d$. Table (3.2) shows the eigenvalues, measured, and theoretical periods at the exact bifurcation points. Finally a simulation run at point is shown in figure (3.6) and demonstrates an unbounded departure from stability.

It is evident that a more detailed analysis had to be performed, and that a Hopf bifurcation analysis should be used.


Figure (3.1) - A typical bifurcation map showing the five distinct regions.


Figure (3.2) - An example of the stable response found in region I. This corresponds to Point a in fig. (3.1).


Figure (3.3) - This figure corresponds to Point bin fig (3.1) and shows the oscillatory behavior in region III.


Figure (3.4) - This simulation corresponds to Point $c$ in fig. (3.1). Note the distinct difference in the nature of oscillations between this point and Point b.


Figure (3.5) . This simulation corresponds to Point d in fig. (3.1). Note the magnitude of the oscillations associated with region IV.


Figure (3.6) - The simulation run at Point $e$ in region $V$ of fig. (3.1) shows the unbounded behavicr found in this region.

| Points | Eigenvalues | $\begin{gathered} \mathrm{U} \\ (\mathrm{fps}) \end{gathered}$ | Measured T (nd sec) | Theoretical $T$ (nd sec) |
| :---: | :---: | :---: | :---: | :---: |
| a | $\begin{aligned} & -0.4581, \quad-0.0001 \\ & -0.0515 \pm 0.4938 \end{aligned}$ | $0.50 U_{0}$ | 4.13 | 5.00 |
| b | $\begin{array}{r} -0.6206,-0.0003 \\ 0.0052 \pm 0.4463 \end{array}$ | $0.62 \mathrm{U}_{0}$ | 5.59 | 5.62 |
| c | $\begin{array}{r} -0.5888,-0.0046 \\ 0.0067 \pm 0.2983 \end{array}$ | $0.94 U_{0}$ | 12.86 | 12.75 |
| d | $\begin{array}{r} -0.3000 \pm 0.2889 \\ 0.0387 \end{array} \pm 0.0268$ | $1.03 U_{0}$ | 80.00 | 155.4 |
| e | $\begin{array}{r} -27.890,-1.7300 \\ 0.7100 \pm 4.1700 \\ \hline \end{array}$ | $1.50 \mathrm{U}_{0}$ | NA | 1.46 |

Table (3.1) - A listing of the points in fig. (3.1) and their associated eigenvalues, and periods.

| Points | Eigenvalues | $U$ <br> $(f p s)$ | Measured $T$ <br> (nd sec) | Theoretical <br> (nd sec) |
| :---: | :--- | :---: | :---: | :---: |
| b | $-0.6070,-0.0002$ <br> $0.0000 \pm 0.4506$ | $0.61 U_{0}$ | 6.11 | 5.57 |
| c | $-0.5710,-0.0054$ <br> $0.0000 \pm 0.2884$ | $0.93 U_{0}$ | 13.13 | 13.18 |
| d | $-0.2675 \pm 0.2063$ <br> $0.0000 \pm 0.0558$ | $1.02 U_{0}$ | 74.93 | 74.67 |

Table (3.2) - Listing of the eigenvalues and periods of the exact bifurcation points associated with the points in Fig. (3.1).
IV. HOPF BIFURCATION

## A. INTIRODOCTION

By definition a Hopf bifurcation occurs when a pair of complex conjugate eigenvalues cross into the right hand halfplane. When this occurs the system will deviate from a steady state solution in an oscillatory manner. This deviation can be either supercritical or subcritical.

For the supercritical case shown in fig. (4.1), stable limit cycles form after straight line stability is lost. Assume that a parameter, $w(t)$, is varying. When $t$ is less than $t_{\text {crit }}$ all of the eigenvalues of the system are located in the left hand half-plane and the system is nominally stable. At $t$ equal to $t_{\text {crit }}$, a complex conjugate pair moves into the right hand half-plane and forms the stable limit cycle. As the distance $w(t)$ increases from $w\left(t_{\text {crit }}\right)$, where the difference $D=w(t)-w\left(t_{\text {crit }}\right)$, the amplitude of the limit cycle will also increase. If $D$ remains small then the system will remain near the nominal steady state solution.

Fig. (4.2) shows the subcritical case with unstable limit cycles being generated prior to the critical point being reached. Thus, as $w(t)$ approaches $w\left(t_{\text {crit }}\right)$ the system could deviate from the nominal steady state solution and converge to a large amplitude limit cycle before the nominal system loses
stability. Hence a random disturbance can cause a nominally stable system, one with all of the eigenvalues in the left hand half-plane, to exhibit oscillatory behavior. Once w(t) equals $w\left(t_{\text {crit }}\right)$ the nominal system becomes unstable and a discontinuous increase in the amplitude of oscillation is seen. This is because there are no nearby stable attractors for the system to converge to.

A system design must make a distinction between these two types of bifurcations because of the disparate nature of the losses in stability. Thus the designer cannot rely on a linear approximation and must use higher order approximations ( $3^{\text {rd }}$ order) of the equations of motion to adequately analyze his/her system.

## B. THIRD ORDER APPROXIMATIONS

The nonlinear equations of motion are expanded using a third order Taylor series approximation near the nominal steady state, $x=[0]$. The control law is then modelled as,

$$
\begin{equation*}
\boldsymbol{\delta}^{\prime}=\boldsymbol{\delta}_{\boldsymbol{s a t}} \tanh \left(\frac{\boldsymbol{\delta}}{\boldsymbol{\delta}_{\boldsymbol{s} a t}}\right) \tag{4.1}
\end{equation*}
$$

where $\delta_{\text {sat }}$ is the saturation angle of the control plane input. Using the same approximation for the control law as the equation of motion, $\delta^{\prime}$ becomes

$$
\begin{equation*}
\delta^{\prime}=8-\frac{1}{3 \delta_{s a t}^{2}} \delta^{3} \tag{4.2}
\end{equation*}
$$

Therefore, the state equations can now be written as

$$
\begin{equation*}
\pm=A x+g(x) \tag{4.3}
\end{equation*}
$$

where

$$
x=\left[\begin{array}{llll}
\theta & w & q & z \tag{4.4}
\end{array}\right]^{T}
$$

and the higher order terms are,

$$
\begin{equation*}
g_{1}=0, \tag{4.5a}
\end{equation*}
$$

$$
\begin{gather*}
g_{2}=b_{1} U^{2} \delta_{3}(\theta, w, q, z)-\frac{1}{6} a_{13} z_{G B} \theta^{3}  \tag{4.5b}\\
\\
+c_{11} q^{2}+c_{12} w q
\end{gather*}
$$

$+c_{21} w q+c_{22} q^{2}$,

$$
\begin{equation*}
g_{4}=-\frac{1}{2} w \theta^{2}+\frac{1}{6} u \theta^{3} . \tag{4.5d}
\end{equation*}
$$

The term $\delta_{3}$ contains the third order terms derives from substituting $\delta$ into $\delta^{\prime}$.

Defining $T$ as the matrix of eigenvectors of $A$ evaluated at the Hopf bifurcation point, then the transformation,

$$
\begin{equation*}
x=T z, \tag{4.6}
\end{equation*}
$$

transforms the system into a canonical form

$$
\begin{equation*}
\dot{z}=T^{-1} A T z+T^{-1} g(T z) . \tag{4.7}
\end{equation*}
$$

At the bifurcation point

$$
T^{-1} A T=\left|\begin{array}{cccc}
0 & -\omega_{0} & 0 & 0  \tag{4.8}\\
\omega_{0} & 0 & 0 & 0 \\
0 & 0 & p & 0 \\
0 & 0 & 0 & 0
\end{array}\right|
$$

with $\omega_{0}>0$ and $p, q<0 . \quad z_{3}$ and $z_{4}$ correspond to $p$ and $q$ and are asymptotically stable. Center manifold theory states that the stable coordinates $z_{3}, z_{4}$ can be expressed as polynomials in the critical coordinates $z_{1}, z_{2}$ and this relationship is at least second order. Therefore, we can write,

$$
\begin{equation*}
z_{3}=\alpha_{1} z_{1}{ }^{2}+\alpha_{2} z_{1} z_{2}+\alpha_{3} z_{2}{ }^{2}, \tag{4.9a}
\end{equation*}
$$

and

$$
\begin{equation*}
z_{4}=\beta_{1} z_{1}^{2}+\beta_{2} z_{1} z_{2}+\beta_{3} z_{2}^{2} . \tag{4.9b}
\end{equation*}
$$

The coefficients $\alpha_{i}$ and $\beta_{i}$ can be computed as follows: We differentiate equations (4.9) with respect to $z$,

$$
\begin{aligned}
& \dot{z}_{3}=2 \alpha_{1} z_{1} \dot{z}_{1}+\alpha_{2}\left(\dot{z}_{1} z_{2}+z_{1} \dot{z}_{2}\right)+2 \alpha_{3} z_{2} \dot{z}_{2}, \\
& \dot{z}_{4}=2 \beta_{1} z_{1} \dot{z}_{1}+\beta_{2}\left(\dot{z}_{1} z_{2}+z_{1} \dot{z}_{2}\right)+2 \beta_{3} z_{2} \dot{z}_{2},
\end{aligned}
$$

and substitute $\dot{z}_{1}=-\omega_{0} z_{2}$, and $\dot{z}_{2}=\omega_{0} z_{1}$. Therefore,

$$
\begin{equation*}
\dot{z}_{3}=\alpha_{2} \omega_{0} z_{1}^{2}+\left(2 \alpha_{3}-2 \alpha_{1}\right) \omega_{0} z_{1} z_{2}-\alpha_{2} \omega_{0} z_{2}^{2}, \tag{4.10a}
\end{equation*}
$$

and

$$
\begin{equation*}
z_{4}=\beta_{2} \omega_{0} z_{1}^{2}+\left(2 \beta_{3}-2 \beta_{1}\right) \omega_{0} z_{1} z_{2}-\beta_{2} \omega_{0} z_{2}^{2} \tag{4.10b}
\end{equation*}
$$

The third and fourth equations of (4.7) are written as,

$$
\left|\begin{array}{l}
\dot{z}_{3}  \tag{4.11}\\
\dot{z}_{4}
\end{array}\right|=\left|\begin{array}{ll}
0 & 0 \\
0 & 9
\end{array}\right|\left|\begin{array}{l}
z_{3} \\
z_{4}
\end{array}\right|+[D]
$$

where,

$$
[D]=T^{-1} g_{2}(T \Sigma),
$$

and $g_{2}(T z)$ contains the second order terms of $g(T z)$. We substitute equations (4.9) into (4.11) and equate coefficients with (4.10). In this way we get a linear system in $\alpha_{i}$, and $\beta_{i}$. From this we can write the two dimensional state space equations as

$$
\begin{gather*}
\dot{z}_{1}=-\omega_{0} z_{2}+r_{11} z_{1}^{3}+r_{12} z_{1}^{2} z_{2}+r_{13} z_{1} z_{2}{ }^{2}+r_{14} z_{2}^{3}  \tag{4.12a}\\
+p_{11} z_{1}{ }^{2}+p_{12} z_{1} z_{2}+p_{13} z_{2}{ }^{2} .
\end{gather*}
$$

and

$$
\begin{align*}
\dot{z}_{2}=\omega_{0} z_{1} & +r_{21} z_{1}^{3}+r_{22} z_{1}^{2} z_{2}+r_{23} z_{1} z_{2}^{2}+r_{24} z_{2}^{3}  \tag{4.12b}\\
& +p_{21} z_{1}^{2}+p_{22} z_{1} z_{2}+p_{23} z_{2}^{2} .
\end{align*}
$$

where the coefficients $r_{i j}$ and $p_{i j}$ are derived from equations (4.8).

These equations are only valid exactly at the Hopf bifurcation point. For speeds $U$ in a region near the bifurcation point these equations become

$$
\begin{equation*}
\dot{z}_{1}=\alpha^{\prime} \epsilon z_{1}-\left(\omega_{0}+\omega^{\prime} \epsilon\right) z_{2}+F_{1}\left(z_{1}, z_{2}\right) \tag{4.13a}
\end{equation*}
$$

and

$$
\begin{equation*}
\dot{z}_{2}=\left(\omega_{0}+\omega^{\prime} \epsilon\right) z_{1}+\alpha^{\prime} \epsilon z_{2}+F_{2}\left(z_{1}, z_{2}\right) \tag{4.13b}
\end{equation*}
$$

where: $\alpha^{\prime}, \omega^{\prime}$ are the derivatives with respect to $U$ of the real and imaginary parts of the critical complex conjugate pair of eigenvalues evaluated at the bifurcation point: $\epsilon$ is the difference in $U$ from the critical value; and, the nonlinear functions $F_{1}$ and $F_{2}$ are

$$
\begin{align*}
F_{1}= & r_{11} z_{1}^{3}+I_{12} z_{1}^{2} z_{2}+I_{13} z_{1} z_{2}^{2}+r_{14} z_{2}^{3}  \tag{4.14a}\\
& +p_{11} z_{1}^{2}+p_{12} z_{1} z_{2}+p_{13} z_{2}^{2} .
\end{align*}
$$

and

$$
\begin{align*}
F_{2}= & r_{21} z_{1}^{3}+r_{22} z_{1}^{2} z_{2}+r_{23} z_{1} z_{2}^{2}+r_{24} z_{2}^{3}  \tag{4.14b}\\
& +p_{21} z_{1}^{2}+p_{22} z_{1} z_{2}+p_{23} z_{2}^{2} .
\end{align*}
$$

By transforming $z_{1}$ and $z_{2}$ to polar coordinates in the form

$$
\begin{equation*}
z_{1}=R \cos \theta \tag{4.15a}
\end{equation*}
$$

$$
\begin{equation*}
z_{2}=R \sin \theta \tag{4.15b}
\end{equation*}
$$

equations (4.13) become

$$
\begin{equation*}
\dot{R}=\alpha^{\prime} \epsilon R+F_{1}(R, \theta) \cos \theta+F_{2}(R, \theta) \sin \theta \tag{4.16a}
\end{equation*}
$$

and

$$
R \Theta=\left(\omega+\omega^{\prime} \epsilon\right) R+F_{2}(R, \theta) \cos \theta-F_{1}(R, \theta) \sin \theta \cdot(4.16 \mathrm{~b})
$$

Equation (4.16a) then yields

$$
\begin{equation*}
\dot{R}=\alpha^{\prime} \epsilon R+P(\Theta) R^{3}+Q(\Theta) R^{2} . \tag{4.17}
\end{equation*}
$$

By averaging equation (4.17) over one cycle we can obtain an equation with constant coefficients. Defining

$$
\begin{equation*}
K=\frac{1}{2 \pi} \int_{0}^{2 \pi} P(\theta) d \theta \tag{4.18}
\end{equation*}
$$

and

$$
\begin{equation*}
L=\frac{1}{2 \pi} \int_{0}^{2 \pi} \rho(\theta) d \theta \tag{4.19}
\end{equation*}
$$

and carrying out the integration we obtain

$$
\begin{equation*}
L=0, \tag{4.20}
\end{equation*}
$$

and

$$
\begin{equation*}
K=\frac{1}{8}\left(3 r_{11}+r_{13}+r_{22}+3 r_{24}\right), \tag{4.21}
\end{equation*}
$$

which reduces equation (4.17) to

$$
\begin{equation*}
\dot{R}=\alpha^{\prime} \epsilon R+K R^{3} . \tag{4.22}
\end{equation*}
$$

The existence and stability of the limit cycle is determined by analyzing the equilibrium points of the averaged equation (4.22), which correspond to periodic solutions in $z_{1}$ $z_{2}$ as seen in the coordinate transformation equations (4.15). From equation (4.22) we can see that two conditions exist,

1) If $\alpha^{\prime}>0$, then:
(a) If $\mathrm{K}>0$ then unstable limit cycles coexist with the stable equilibrium for $\epsilon<0$.
(b) If $\mathrm{K}<0$ then stable limit cycles coexist with the unstable equilibrium for $\epsilon>0$.
2) If $\alpha^{\prime}<0$, then:
(a) If $K>0$ the unstable limit cycles coexist with the stable equilibrium for $\epsilon>0$.
(b) If $\mathrm{K}<0$ then stable limit cycles coexist with the unstable equilibrium for $\epsilon<0$.

From this criteria, by computing the nonlinear coefficient K we can use it to distinguish the two different types of Hopf bifurcations:

- Supercritical if $K<0$;
- Subcritical if $\mathrm{K}>0$.
[ref. 6]


## C. RESULTS

A typical bifurcation map of stability for the SUBOFF Model is shown in fig. (4.3). This map is characterized by the Pitchfork bifurcation curve (P) and the three Hopf bifurcation curves ( $\mathrm{H} 1, \mathrm{H} 2$, and H 3 ). The nature of curve P was previously analyzed and those results (Riedel, 1993) are reconfirmed in this study.

Curve H1 is characterized by a weak supercritical branch $(a \rightarrow b)$ at low nominal speeds, $U_{0}$. As $U_{0}$ increases this branch develops a weak to moderate subcritical behavior with $K$ between 0 and $10^{2}$. The second branch of $\mathrm{H} 1(\mathrm{~b} \rightarrow \mathrm{c})$ has a
consistent moderate subcritical behavior with $K$ on the order of $10^{2}$.

Cusp (C) marks the intersection of curve H2 with curve H3. The cusp is highly dependent on both $U_{0}$ and initial stability $\mathrm{Z}_{\mathrm{GB}}$.

As $Z_{G B}$, for a given $U_{0}$, increases curve $H 2(d \rightarrow e)$ shifts from a very weak subcritical nature with $K$ between $+10^{-2}$ and 1 to a very weak supercritical nature with $K$ between -1 and -10-2. With a lower $U_{0}$ and/or higher $Z_{G B}$, point $e$ moves down in the time constant and may not intersect curve H3.

Curve H3 ( $f \rightarrow g$ ) is a strong supercritical bifurcation with $K$ values between $-10^{4}$ and $-10^{6}$. The position of $H 3$ is independent of $\mathrm{U}_{0}$, initial stability, and control surface coordination gain.

Finally, curve $H 4$ is a strong subcritical branch with $K$ having values between $10^{3}$ and $10^{6}$. Because of this highly subcritical behavior, $H 4$ can dominate and obscure the stable region at speeds greater than $U / U_{0}=1$.

Figure (4.4) plots the $K$ values for a representative bifurcation map shown in fig (3.1). Note the predicted superand subcritical branches associated with fig (4.3). Point (a) is inside the stable region $I$ and the numerical simulations seen in fig. (3.2) converge to zero. Point (b) is located in the unstable region, immediately after a supercritical bifurcation. As a result, small amplitude limit cycle oscillations have developed. The same is true as we move
towards point (c) although we expect a family of unstable limit cycles around this point as a result of the subcritical bifurcation. This point is further explored in the next section. As we approach point (d) a family of stable limit cycles is generated but its behavior is influenced by the previously developed unstable limit cycles. The real part of the critical pair of eigenvalues is becoming positive and relatively large, see fig. (3.1), which means that the amplitudes of these stable limit cycles are expected to be significantly higher, a result which is confirmed by fig. (3.5). The imaginary parts of the critical pair of eigenvalues are also changing very fast in this region. This means that the period of these limit cycles must be computed based on the value of the imaginary part right at the bifurcation point, rather than its value at the specific parameter point. This was observed previously in tables (3.1) and (3.2). Point (e) is in the strongly subcritical region $V$, thus we see the rapid divergence from stability as seen in fig. (3.6).

## D. SImULATIONS

The response of the submersible was simulated using an Adams-Bashforth integration scheme in Fortran program roding and is included in Appendix C. The contro. law (2.15) and gains (2.17) discussed in Chapters II and III were used. Nondimensional ships speed and control system time constant,
nominal speed, initial stability, and control surface coordination gain were the input values to tre system. A nominal $0.1 \mathrm{ft} / \mathrm{sec}$ vertical speed was used as the external initial disturbance.

The simulations were used to compare the Hopf bifurcation data in two ways:

1) By confirming sub-/ supercritical behavior predicted by the K factor, and;
2) By comparing the predicted to simulated period of oscillations.

In fig. (4.5) we have plotted the stable equilibria and the stable and unstable limit cycles from our example first used in fig. (3.1). Figure (4.5) clearly shows the predicted sub- and supercritical behavior that the $K$ values predicted. The important features to note in fig. (4.5) are:

1. Unstable limit cycles found with the subcritical Hopf bifurcation;
2. The divergence of the amplitudes as the velocity moves away from the bifurcation point velocity, and;
3. The rapid divergence of the right most bifurcation and its quick and abrupt termination.

To see why the abrupt termination occurs we must look at the root locus plot as shown in fig. (4.6). The parameter in this the third bifurcation terminates when there is a break in point and the complex conjugate pair, which had given the oscillatory response, becomes two positive real eigenvalues.

These two eigenvalues now make the system completely unstable. The splitting of this complex conjugate pair into two real and positive eigenvalues can be better seen in fig (4.7).

SLPERCRITICAL HOPF BIFLRCATION


Figure (4.1) - An example of a supercritical Hopf bifurcation where the system has a nominal steady state solution until point $C$, which occurs at $t_{\text {crit }}$. After $t_{\text {crit }}$ the system becomes unstable but forms a stable limit cycle.

## SUBCRITICAL HOPF BIFLRCATION



Figure (4.2) - This figure shows a subcritical Hopf bifurcation where the system loses stability prior to reaching point $C$, which again occurs at $t_{\text {crit }}$.

A TYPICAL BIFURCATION MAP


Figure (4.3) - An annotated bifurcation map for the SUBOFF Model.


Figure (4.4) - A plot of the K values associated with fig. (3.1).


Figure (4.5) - This shows the amplitude response for a speed range encompassing the Hopf bifurcation points that are shown in fig. (3.1).


Figure (4.6) - The root locus plot of the eigenvalues associated with fig. (4.5).


Figure (4.7) - A plot of the real parts of the eigenvalues plotted in fig. (4.6).

## V. APPLICATIONS

## A. CONTROL SYSTEA PARANGTERS

From the typical bifurcation maps we can see that a region of stability is created between the pitchfork and Hopf bifurcation boundaries. For the control system designer the limits or parameters must be defined prior to starting the design. By maximizing the region of stability we can give the designer the most leeway in his/her work. There are three parameters that we can use to change the bifurcation maps, nominal speed, initial stability, and the control surface gain coefficient.

First we will look at changing the nominal speed. In fig. (5.1) we have plotted three curves for nominal speeds of 3.0 , 9.0, and 15.0 fps . We can see that although the pitchfork line moves to the left in dimensional speeds this line remains nearly constant with a dimensional stern planes reversal occurring at 1.2 kts . The high speed Hopf boundaries $\left(U / U_{0}\right.$ > 1.0) move apart as the nominal speed increases. The effectiveness of increasing $U_{0}$ is limited in the upper branch by the fixed position of the $H 3$ line with the maximum practical $T_{c}$ achieved at $a U_{0}=9.0$ fps. In the lower arm there is no increase in the stability area after $U_{0}=9.0 \mathrm{fps}$ therefore any increase in $U_{0}$ is pointless. For the low speed

Hopf curves ( $U / U_{0}<1.0$ ) we quickly lose our margin of stability as $U_{0}$ increases thus necessitating further changes to regain the lost area of stability.

The next parameter change we examined was varying the initial stability. Figure (5.2) shows the effect of increasing $Z_{g}$ from 0.1 to 0.4 ft . The subcritical H 4 branch remains constant while the upper high speed Hopf branch moves down effectively decreasing the area of stability. The low speed Hopf curves move up to increase the low speed area of stability. We can see that the additional loss in area is by the movement of the pitchfork line to the right. At a $\mathbf{z}_{g}=$ 0.4 ft stern planes reversal occurs at a dimensional speed of 2.4 kts which is well within the currently accepted range of 1.0 to 3.0 kts for modern submarines. Therefore in this model we would want to balance the initial stability to maximize the low and high speed areas.

An increase in the control surface gain coefficient, $\alpha$, is shown in fig. (5.3). Note that the low and high speed Hopf curves all move up in time constant. While the low speed Hopf curves give a large increase in stability, the high speed curves move up proportionally and there is no increase in stability area. This does allow the designer to shift the range of stable time constants without a loss of high speed stability. The pitchfork line will move to the left until it equals zero when $\alpha=1.0$.

In order to examine what happens at the extremes of the design options we can look at the low nominal speed ( 3 fps) bifurcation maps. Figure (5.4) shows the typical bifurcation map such as the one we have previously discussed. As the metacentric height is increased there are significant changes in the nature of the bifurcation curves. In fig. (5.5) we see that the pitchfork bifurcation line has moved significantly to the right and has intersected the low speed Hopf bifurcation curve. This intersection along with the merger of the H 2 and H4 curves have combined to reduce the region of stability to a negligible portion of the map.

A further increase in the metacentric height as shown in fig. (5.6) demonstrates a dramatic change in the nature of the stability of the model. The low speed region has two hyperbolic-like Hopf bifurcation curves (the upper curve occurs well above the region of interest) bounding the lower and upper limits of stability. For the speeds $U / U_{0}>1.0$, the pitchfork bifurcation line now intersects the H 2 curve and has changed from a supercritical to a subcritical pitchfork.

Figure (5.7) is an example of the effect just described and how it can occur at higher nominal speeds. This shows that although initial stability is necessary for overall stability, if the metacentric height becomes too large it can have an adverse affect on the performance of the submarine.

## B. SUBMARINE DESIGN EVALOATION

For all of the simulations up to this point the moment and inertia coefficients of the control surfaces have been . $5 x$ of those listed in the SUBOFF Model report. By using the reduce effect control surface input we have been able to show stable simulations in all of the mapped stable regions of the bifurcation maps.

Linear bifurcation methods fail to predict the change in system response for changes in control surface coefficients. The bifurcation maps for . $1 x$ to $1.0 x$ the control surface coefficients are exactly the same, in other words the bifurcation points are independent of the size and effectiveness of the control surfaces. Therefore we must examine the $K$ values in order to predict the response of the model.

Figure (5.8) shows the change in stability for the model with and increase in the control surface coefficients. The area of lost stability is indicated by the shaded portion of the map. This loss of stability is caused by the shift of the H2 curve from weak to moderately supercritical to a strongly subcritical curve. With two strongly subcritical curves in the high speed region $\left(U / U_{0}>1.0\right)$ the possibility of subcritical capture is greatly increased. This effect is confirmed by running an extensive set of simulations and mapping the change in stable to unstable response. We must note that this instability occurs in a region that has four
eigenvalues with negative real parts where linear control system design would not predict an instability.

With this method a design can be tested and then modified using the design goals as determined by the nonlinear response in the simulations. This can change a completely unstable model such as the SUBOFF Model into one where a large margin of stability and control system latitude is design into it.


Figure (5.1) - The effects of changing the nominal speed, $U_{0}$, on the bifurcation maps.


Figure (5.2) - The effects of changing initial stability, $Z_{g}$, on the bifurcation maps.


Figure (5.3) - The effects of changing the control surface gain coefficient, $\alpha$, on the bifurcation maps.


Figure (5.4) - A low nominal speed bifurcation map.


Figure (5.5) - This demonstrates the effects of an increase in the metacentric height for low nominal speeds.


Figure (5.6) - As the metacentric height increases it has an adverse effect on the stability of the submarine.


Figure (5.7) - An example of increasing the metacentric height for a higher nominal speed model.


Figure (5.8) . The loss of stability due to using full moment and inertia coefficients vice the same coefficients $\times 0.5$.

## VI. CONCLUSIONS AND RECONAENDATIONS

## a. COAKCLOSIONS

The application of $a$ Hopf bifurcation analysis to a submarine design can be an effective tool in the design evaluation and modification phase. These methods when paired with programs that generate hydrodynamic coefficients for a submarine will save time, effort, and money by reducing the amount of model testing necessary to validate a design. An effective set of control system design parameters will be generated in the process that will be optimal for the final design of a submersible.

Secondly, this system will give the limits of the range of metacentric heights that will maintain stability for the full range of speeds of the design. As was shown changes in the metacentric height can have a dramatic affect on stability.

Finally, an evaluation of the need for bow planes or forward control surfaces are actually to maintain control of the submarine. If the forward planes can be eliminated a potential source of noise wolld also be eliminated along with simplifying the design of the control system.

## B. pecommendations

First, modify the programs to evaluate the performance of the submarine for the effects of external forces such as, wave effects, currents, and free surface effects.

Second, perform a systematic study of the bifurcations at conditions other than nominal in trim conditions, such as out of neutral buoyancy (heavy/light), or out-of-trim fore and aft conditions.

## APPENDIX A - BIFURCATION NNALYSIS PROGRAM

```
C PROGRAM BIFURI.FOR
C BIFURCATION ANALYSIS FOR THE DARPA SUBOFF MODEL
C
C
    PARAMETERS ARE: TC VS. U
    IMPLICIT DOUBLE PRECISION (A-H,O-Z)
    DOUBLE PRECISION K1,K2,K3,K4,L,MQDOT,
    &
                                MWDOT , MQ, MW, MDS , MDB , MD ,
    & MASS,IY,WF,pi
        DIMENSION A(4,4),FV1 (4),IV1(4), ZZZ (4,4),WR(4),WI (4)
C
    OPEN (11,FILE='BIF1.RES',STATUS='NEW')
    OPEN (12,FILE='BIF2.RES',STATUS='NEW')
    OPEN (13,FILE='BIF3.RES',STATUS='NEW')
C
    WEIGHT=1556.2363
    BUO =1556.2363
    L =13.9792
    IY =561.32
    G =32.2
    MASS =WEIGHT/G
    RHO =1.94
    XB =0.0
    ZB}=0.
    pi =3.14159
C WRITE (*,*) 'ENTER MIN, MAX, AND # OF INCREMENTS IN TIME
                                    CONSTANT'
C READ (*,*) TCMIN,TCMAX,ITC
C WRITE (*,*) 'ENTER MIN, MAX, AND # OF INCREMENTS IN
                VEHICLE SPEED'
C READ (*,*) UMIN,UMAX,IU
C WRITE (*,*) 'ENTER NOMINAL SPEED'
C
C
    READ (*,*) U0
    WRITE (*,*) 'ENTER XG AND ZG'
    READ (*,*) XG,ZG
    TCMIN = 0.1
    TCMAX =9.0
    ITC = 200
    IU = 200
    UO = 12.0
    XG =0.0
    WRITE (*,*) 'ENTER MIN, MAX, OF VEHICLE SPEED'
    READ (*,*) UMIN,UMAX
    WRITE (*,*) 'ENTER ZG'
    READ (*,*) ZG
```

C
ZGB=2G-ZB
TCMIN=TCMIN*L/U0
TCMAX $=$ TCMAX*L/UO
UMIN =UMIN*UO
UMAX =UMAX*UO
C

> ZQDOT=-6.3300E-04*0.5*RHO*L**4
> ZWDOT $=-1.4529 \mathrm{E}-02 * 0.5 * R H O *$ L** 3
> $\mathrm{ZQ}=7.5450 \mathrm{E}-03 * 0.5 * \mathrm{RHO} * \mathrm{~L} * * 3$
> ZW $=-1.3910 \mathrm{E}-02 * 0.5 *$ RHO $\mathrm{ZH}_{\mathrm{L}} * 2$
> ZDS $=0.5 *(-5.6030 \mathrm{E}-03 * 0.5 * R H O * L * * 2)$
> ZDB $=0.5 *(-5.6030 \mathrm{E}-03 * 0.5 * \mathrm{RHO} * \mathrm{~L} * * 2)$
> MQDOT $=-8.8000 \mathrm{E}-04 * 0.5 * \mathrm{RHO} * \mathrm{~L} * * 5$
> MNDOT $=-5.6100 \mathrm{E}-04 * 0.5 *$ RHO $=\mathrm{L} * * 4$
> MQ $=-3.7020 \mathrm{E}-03 * 0.5 * \mathrm{RHO} * \mathrm{~L} * * 4$
> MW $=1.0324 \mathrm{E}-02 * 0.5 *$ RHO $=\mathrm{L} * * 3$
> MDS $=0.5 *(-2.4090 \mathrm{E}-03 * 0.5 * \mathrm{RHO} \mathrm{L} * * 3)$
> MDB $=0.5 *(2.4090 \mathrm{E}-03 * 0.5 * \mathrm{RHO}$ (L**3)
$A L P H A=-0.0$
ZD=ZDS+ALPHA* ZDB
$M D=M D S+A L P H A * M D B$
C
DV $\quad=($ MASS -ZWDOT$) \star($ IY - MQDOT $)$
\& $\quad$ - (MASS*XG+ZQDOT) * (MASS*XG+MWDOT)
A11DV $=(I Y-M Q D O T) * Z W+(M A S S * X G+Z Q D O T) ~ * M W$
A12DV $=(I Y-M Q D O T) *($ MASS $+Z Q)+($ MASS *XG $+Z Q D O T) *($ MQ-MASS*XG)
A13DV $=-($ MASS*XG + ZQDOT $)$ *WEIGHT
B1DV $=(I Y-M Q D O T) * Z D+(M A S S * X G+Z Q D O T) * M D$
A2 IDV = (MASS - ZWDOT) *MW+ (MASS*XG+MWDOT) * ZW

\& $\quad+($ MASS *XG + MNDOT) * (MASS + ZQ)
A2 3DV $=$ - (MASS - ZWDOT) *WEIGHT
B2DV $=($ MASS - ZWDOT $) * M D+(M A S S * X G+M W D O T) * Z D$
A11=A11DV/DV
A12=A12DV/DV
A13 =A13DV/DV
A21=A21DV/DV
A22 =A22DV/DV
A23=A23DV/DV
B1 =B1DV /DV
$B 2=B 2 D V / D V$
C
EPS $=1 . D-5$
ILMAX $=1500$

WRITE (*, 2001) I, ITC
TC=TCMIN+(I-1)*(TCMAX-TCMIN) /(ITC-1)
POLE $=1.0 / \mathrm{TC}$
ALPHA3 $=4.0 *$ POLE
ALPHA2 $=6.0 *$ POLE $* * 2$
ALPHA1=4.0*POLE**3
ALPHAO $=$ POLE $* * 4$
K4 =ALPHAO / ( $\mathrm{B} 1 * \mathrm{~A} 21-\mathrm{B} 2 * A 11)$ *UO** 4
22M $+(B 1 * A 23-B 2 * A 13) * Z G B * U O * * 2)$
$\mathrm{A} 2 \mathrm{M}=\mathrm{BI}$ *UO**2
A3M=B2*U0**2
AOM $=-(\mathrm{A} 11+\mathrm{A} 22) * \cup O-\mathrm{ALPHA} 3$
$\mathrm{B} 1 \mathrm{M}=\mathrm{B} 2 * \mathrm{UO} * * 2$
$\mathrm{B} 2 \mathrm{M}=(\mathrm{B} 2 * \mathrm{~A} 12-\mathrm{B} 1 * \mathrm{~A} 22) * \mathrm{U} 0 * * 3$
$B 3 M=(B 1 * A 21-B 2 * A 11) * U 0 * * 3$
$\mathrm{BOM}=(\mathrm{A} 11 * \mathrm{~A} 22-\mathrm{A} 21 * \mathrm{~A} 12) * \mathrm{UO} * * 2-\mathrm{A} 23 * \mathrm{ZGB}-\mathrm{ALPHA2}-\mathrm{B} 1 * \mathrm{U} 0 * \mathrm{U} 0 * \mathrm{~K} 4$
$\mathrm{C} 1 \mathrm{M}=(\mathrm{B} 2 * \mathrm{~A} 11-\mathrm{B} 1 * \mathrm{~A} 21) * \mathrm{U} 0 * * 3$
$\mathrm{C} 2 \mathrm{M}=(\mathrm{B} 1 * \mathrm{~A} 23-\mathrm{B} 2 * \mathrm{~A} 13) * \mathrm{ZGB} * \mathrm{U} 0 * * 2$
$\mathrm{COM}=(\mathrm{A} 13 * A 21-\mathrm{A} 23 * \mathrm{~A} 11) * Z \mathrm{~GB} * \mathrm{UO}+\mathrm{ALPHA} 1$
$-(\mathrm{B} 2+\mathrm{B} 1 * \mathrm{~A} 22-\mathrm{B} 2 * \mathrm{~A} 12) * \mathrm{~K} 4$ * $\mathrm{U} 0 * * 3$
$\mathrm{K} 2=\mathrm{C} 1 \mathrm{M} * \mathrm{BOM}^{2} \mathrm{~A} 3 \mathrm{M}-\mathrm{B} 1 \mathrm{M} * \mathrm{COM} * A 3 \mathrm{M}-\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 3 \mathrm{M} *$ AOM
$K 2=K 2 /(C 1 M * B 2 M * A 3 M-B 1 M * C 2 M * A 3 M-C 1 M * B 3 M * A 2 M)$
$\mathrm{K} 1=(\mathrm{COM}-\mathrm{C} 2 \mathrm{M} * \mathrm{~K} 2) / \mathrm{C} 1 \mathrm{M}$
$\mathrm{K} 3=(\mathrm{A} 0 \mathrm{M}-\mathrm{A} 2 \mathrm{M} * \mathrm{~K} 2) / \mathrm{A} 3 \mathrm{M}$
DO $2 J=1, I U$
$U=$ UMIN $+(J-1) *($ UMAX -UMIN $) /(I U-1)$
$A(1,1)=0.0 D 0$
$A(1,2)=0.0 D 0$
$A(1,3)=1$. ODO
$A(1,4)=0.0 D 0$
$A(2,1)=A 13 * Z G B+B 1 * U * U * K 1$
$A(2,2)=A 11 * U \quad+B 1 * U * U * K 2$
$A(2,3)=A 12 * U \quad+B 1 * U * U * K 3$
$A(2,4)=\quad B 1 * U * U * K 4$
$A(3,1)=A 23 * Z G B+B 2 * U * U * K 1$
$A(3,2)=A 21 * U+B 2 * U * U * K 2$
$A(3,3)=A 22 * U+B 2 * U * U * K 3$
$A(3,4)=$
B2*U*U*K4
$A(4,1)=-U$
$A(4,2)=1$. 0D0
$A(4,3)=0.0 D 0$
$A(4,4)=0.0 D 0$
CALL RG(4,4, A, WR,WI, 0, ZZZ, IV1, FV1, IERR)
CALL DSTABL (DEOS,WR,WI, FREQ)

C

|  | IF (J.GT.1) GO TO 10 |
| :---: | :---: |
|  | DEOSOO=DEOS |
|  | UOO = U |
|  | LL=0 |
|  | GO TO 2 |
| 10 | DEOSNN=DEOS |
|  | UNN $=\mathrm{U}$ |
|  | PR=DEOSNN*DEOSOO |
|  | IF (PR.GT.O.DO) GO TO 3 |
|  | LL=LL+1 |
|  | IF (LL.GT.3) STOP 1000 |
|  | IL=0 |
|  | UO=UOO |
|  | UN=UNN |
|  | DEOSO=DEOSOO |
|  | DEOSN=DEOSNN |
| 6 | UL=UO |
|  | UR $=$ UN |
|  | DEOSL=DEOSO |
|  | DEOSR=DEOSN |
|  | $\mathrm{U}=$ (UL+UR) / $2 . \mathrm{DO}$ |
|  | $\mathrm{A}(1,1)=0.0 \mathrm{DO}$ |
|  | $A(1,2)=0.000$ |
|  | $A(1,3)=1.000$ |
|  | $A(1,4)=0.000$ |
|  | $A(2,1)=A 13 * Z G B+B 1 * U * U * K 1$ |
|  | $A(2,2)=A 11 * U+B 1 * U * U * K 2$ |
|  | $A(2,3)=A 12 * U \quad+B 1 * U * U * K 3$ |
|  | $A(2,4)=\quad B 1 * U * U * K 4$ |
|  | $A(3,1)=A 23 * Z G B+B 2 * U * U * K 1$ |
|  | $A(3,2)=A 21 * U \quad+B 2 * U * U *$ K2 |
|  | $A(3,3)=A 22 * U \quad+B 2 * U * U * K 3$ |
|  | $A(3,4)=\quad B 2 * U * U * K 4$ |
|  | $A(4,1)=-\mathrm{U}$ |
|  | $A(4,2)=1.000$ |
|  | $A(4,3)=0.0 \mathrm{DO}$ |
|  | $A(4,4)=0.000$ |
|  | CALL RG(4, 4, A, WR, WI, 0, ZZZ, IV1, FV1, IERR) |
|  | CALL DSTABL (DEOS, WR, WI, FREQ) |
|  | DEOSM=DEOS |
|  | UM=U |
|  | PRL=DEOSL*DEOSM |
|  | PRR=DEOSR*DEOSM |
|  | IF (PRL.GT.O.DO) GO TO 5 |
|  | UO=UL |
|  | UN=UM |
|  | DEOSO=DEOSL |
|  | DEOSN=DEOSM |

```
            IL=IL+1
            IF (IL.GT.ILMAX) STOP 3100
            DIF=DABS (UL-UM)
            IF (DIF.GT.EPS) GO TO 6
            U=UM
            GO TO 4
            5 IF (PRR.GT.O.DO) STOP 3200
            UO=UM
            UN=UR
            DEOSO=DEOSM
            DEOSN=DEOSR
            IL=IL+1
                IF (IL.GT.ILMAX) STOP 3100
                DIF=DABS (UM-UR)
                    IF (DIF.GT.EPS) GO TO 6
                U=UM
                LLL=10+LL
                WRITE (LLL, 2002) U/UO,TC*UO/L
                    U UOO=UNN
                DEOSOO=DEOSNN
                    CONTINUE
    1 CONTINUE
C
    2001 TORMAT (2I5)
    2002 FORMAT (4(3X,F10.6))
        END
C
        SUBROUTINE DSTABL (DEOS,WR,WI,OMEGA)
        IMPLICIT DOUBLE PRECISION (A-H,O-Z)
        DIMENSION WR(4),WI(4)
        DEOS=-1. OD +20
        DO 1 I=1,4
        IF (WR(I).LT.DEOS) GO TO 1
        DEOS=WR(I)
        IJ=I
        1 CONTINUE
        OMEGA=WI (IJ)
        OMEGA=DABS (OMEGA)
        RETURN
        END
```


## APPENDIX B - HOPF BIFURCATION PROGRAK

PROGRAM HOPFIA.FOR EVALUATION OF HOPF BIFURCATION FORMULAE USING THE SUBOFF SUBMARINE MODEL

IMPLICIT DOUBLE PRECISION (A-H,O-Z) DOUBLE PRECISION L,IY,MASS,MQDOT,MWDOT,

1
2
3
DOUBLE PRECISION M11,M12,M13,M14,M21,M22,M23,M24, 1 M31,M32,M33,M34,M41,M42,M43,M44, $2 \quad \mathrm{~N} 11, \mathrm{~N} 12, \mathrm{~N} 13, \mathrm{~N} 14, \mathrm{~N} 21, \mathrm{~N} 22, \mathrm{~N} 23, \mathrm{~N} 24$, 3 N31,N32,N33,N34,N41,N42,N43,N44, 4 L21,L22,L23,L24,L31,L32,L33,L34, 5 L41,L42,L43,L44,L25,L26,L27,L35, 6 L36, L37, L21A, L22A, L23A, L24A, L31A, 7 L32A, L33A, L34A

DIMENSION $A(4,4), T(4,4), \operatorname{TINV}(4,4), \operatorname{FV1}(4), \operatorname{IV1}(4), \operatorname{YYY}(4,4)$ DIMENSION WR (4), WI (4), TSAVE (4, 4),
$1 \operatorname{TLUD}(4,4)$, IVLUD (4), SVLUD (4)
DIMENSION ASAVE $(4,4), A 1(4,4), A 2(4,4)$
OPEN (10,FILE='HOPF.DAT',STATUS='OLD')
OPEN (20,FILE='HOPF.RES',STATUS='NEW')
WEIGHT=1556.2363
IY $=561.32$
$\mathrm{L}=13.9792$
RHO $=1.94$
$\mathrm{G}=32.2$
$X G=0.0$
$\mathrm{ZB}=0.0$
MASS =WEIGHT/G
BOY =WEIGHT
ZQDOT $=-6.3300 \mathrm{E}-04 * 0.5 *$ RHO*I**4
ZWDOT $=-1.4529 \mathrm{E}-02 * 0.5 *$ RHO*L**3
$\mathrm{ZQ}=7.5450 \mathrm{E}-03 * 0.5 * \mathrm{RHO} \mathrm{L} * * 3$
ZW $=-1.3910 \mathrm{E}-02 * 0.5 *$ RHO*L**2
ZDS $=0.5 *(-5.6030 \mathrm{E}-03 * 0.5 * \mathrm{RHO} * \mathrm{~L} * * 2)$
ZDB $=0.5 *\left(-5.6030 \mathrm{E}-03 * 0.5 *\right.$ RHO $\left.\mathrm{L}_{\mathrm{L}} * * 2\right)$
MQDOT $=-8.8000 \mathrm{E}-04 * 0.5 *$ RHO*L**5
MWDOT $=-5.6100 \mathrm{E}-04 * 0.5 *$ RHO*L**4
MQ $=-3.7020 \mathrm{E}-03 * 0.5 *$ RHO*L**4
MW $=1.0324 \mathrm{E}-02 * 0.5 *$ RHO*L**3
MDS $=0.5 *(-2.4090 E-03 * 0.5 * R H O * L * * 3)$

```
    MDB =0.5*(2.4090E-03*0.5*RHO*L**3)
C
C WRITE (*,1002)
C
    WRITE (*,1001)
    READ (*,*) ITOTAL
    READ (*,*) UO,ZG,DSAT
    WRITE (*,1004)
    READ (*,*) ZG
    WRITE (*,1003)
    READ (*,*) ALPHA
    ZGB=2G-ZB
    UO}=9.
    DSAT =0.4
C
    ZD=2DS+ALPHA*ZDB
    MD=MDS +ALPHA*MDB
C
C
    DETERMINE [A] AND
[B] COEFFICIENTS
C
    DV = (MASS - ZWDOT) * (IY -MQDOT)
& - (MASS*XG+ZQDOT) * (MASS*XG+MWDOT)
    A11DV= (IY-MQDOT) *ZW+(MASS*XG+ZQDOT) *MW
    A12DV= (IY-MQDOT) * (MASS+ZQ) + (MASS*XG+ZQDOT) * (MQ-MASS*XG)
    A13DV =-(MASS*XG+ZQDOT)*WEIGHT
    B1DV = (IY-MQDOT)*ZD+(MASS*XG+ZQDOT) *MD
    A21DV = (MASS - ZWDOT) *MW+ (MASS*XG+MWDOT) *ZW
    A22DV= (MASS - ZWDOT) * (MQ-MASS*XG)
&
    +(MASS*XG+MWDOT) * (MASS +ZQ)
    A2 3DV=- (MASS - ZWDOT) *WEIGHT
    B2DV = (MASS-ZWDOT)*MD+(MASS*XG+MWDOT)*ZD
C
    A11=A11DV/DV
    A12=A12DV/DV
    A13=A13DV/DV
    A21=A21DV/DV
    A22=A22DV/DV
    A23=A23DV/DV
    B1 =B1DV/DV
    B2 =B2DV/DV
C
C11DV=(IY-MQDOT) *MASS*ZG
C12DV=- (MASS*XG+ZQDOT)*MASS*ZG
C2IDV=- (MASS - ZWDOT) *MASS*ZG
C22DV=(MASS*XG+MWDOT) *MASS*ZG
C
C11=C11DV/DV
C12=C12DV/DV
C21=C21DV/DV
C22=C22DV/DV
```

C
DO 1 IT=1,ITOTAL
WRITE (*,3001) IT, ITOTAL
READ (10,*) U,TC
TC=TC*L/UO
$\mathrm{U}=\mathrm{U} * \mathrm{UO}$
CALCULATE THE GAINS
POLE=1.0/TC
ALPHA3 $=4.0 *$ POLE
ALPHA2 $=6.0 *$ POLE** 2
ALPHA1=4.0*POLE**3
ALPHAO = POLE**4
$\mathrm{K} 4=\mathrm{ALPHAO} /((\mathrm{B} 1 * \mathrm{~A} 21-\mathrm{B} 2 * \mathrm{~A} 11) * \mathrm{U} 0 * * 4$ $+(B 1 * A 23-B 2 * A 13) * Z G B * U 0 * * 2)$
A2M=B1*UO**2
$\mathrm{A} 3 \mathrm{M}=\mathrm{B} 2$ * $\mathrm{U} 0 * * 2$
AOM=-(A11+A22)*UO-A PPHA3
B1M=B2*UO**2
$\mathrm{B} 2 \mathrm{M}=(\mathrm{B} 2 * \mathrm{~A} 12-\mathrm{B} 1 * \mathrm{~A} 22) * \mathrm{U} 0 * * 3$
$\mathrm{B} 3 \mathrm{M}=(\mathrm{B} 1 * \mathrm{~A} 21-\mathrm{B} 2 * \mathrm{~A} 11) * \mathrm{U} 0 * * 3$
BOM= (A11*A22-A21*A12) *U0**2-A23*ZGB-ALPHA2-B1*U0*UO*K4
$\mathrm{C} 1 \mathrm{M}=(\mathrm{B} 2 * \mathrm{~A} 11-\mathrm{B} 1 * \mathrm{~A} 21) * \mathrm{UO} * * 3$
$\mathrm{C} 2 \mathrm{M}=(\mathrm{B} 1 * A 23-\mathrm{B} 2 * \mathrm{~A} 13)$ * ZGB*UO**2
COM=(A13*A21-A23*A11) *ZGB*U0 +ALPHA1
$-(\mathrm{B} 2+\mathrm{B} 1 * \mathrm{~A} 22-\mathrm{B} 2 * \mathrm{~A} 12) * \mathrm{~K} 4 * \mathrm{U} 0 * * 3$
$\mathrm{K} 2=\mathrm{C} 1 \mathrm{M} * B 0 \mathrm{M} * A 3 \mathrm{M}-\mathrm{B} 1 \mathrm{M} * \mathrm{COM} * A 3 \mathrm{M}-\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 3 \mathrm{M} * A 0 M$
$\mathrm{K} 2=\mathrm{K} 2 /(\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 2 \mathrm{M} * \mathrm{~A} 3 \mathrm{M}-\mathrm{B} 1 \mathrm{M} * \mathrm{C} 2 \mathrm{M} * \mathrm{~A} 3 \mathrm{M}-\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 3 \mathrm{M} * \mathrm{~A} 2 \mathrm{M})$
$\mathrm{K} 1=(\mathrm{COM}-\mathrm{C} 2 \mathrm{M} * \mathrm{~K} 2) / \mathrm{C} 1 \mathrm{M}$
$\mathrm{K} 3=(\mathrm{A} 0 \mathrm{M}-\mathrm{A} 2 \mathrm{M} \star \mathrm{K} 2) / \mathrm{A} 3 \mathrm{M}$

EVALUATE NONLINEAR RUDDER EXPANSION COEFFICIENTS

```
DTTW=-(1.DO/(3.D0*DSAT**2))*3.D0*K1*K1*K2
DTWW=-(1.DO/(3.DO*DSAT**2))*3.D0*K1*K2*K2
DTTQ=-(1.DO/(3.D0*DSAT**2))*3.D0*K1*K1*K3
DTQQ=-(1.D0/(3.D0*DSAT**2))*3.D0*K1*K3*K3
DTTZ=-(1.DO/(3.D0*DSAT**2))*3.D0*K1*K1*K4
DTZZ=-(1.DO/(3.DO*DSAT**2))*3.D0*K1*K4*K4
DWWQ =-(1.DO/(3.DO*DSAT**2))*3.DO*K2*K2*K3
DWQQ=-(1.DO/(3.DO*DSAT**2))*3.DO*R2*K3*K3
DWWZ=-(1.DO/(3.DO*DSAT**2))*3.D0*K4*K2*K2
DWZZ=-(1.DO/(3.D0*DSAT**2))*3.DO*K4*K4*K2
DQQZ=-(1.DO/(3.DO*DSAT**2))*3.DO*K4*K3*K3
DQZZ=-(1.DO/(3.DO*DSAT**2))*3.D0*K4*K4*K3
DTWQ=-(1.DO/(3.D0*DSAT**2))*6.D0*R1*K2*K3
DTWZ=-(1.DO/(3.DO*DSAT**2))*6.D0*K1*K4*K2
DTQZ=-(1.DO/(3.DO*DSAT**2))*6.DO*K1*K4*K3
DWQZ=-(1.DO/(3.DO*DSAT**2))*6.D0*K4*K2*K3
```

```
DTTT=-(1.DO/(3.DO*DSAT**2))*1.DO*KI*K1*K1
DWWW=-(1.DO/(3.DO*DSAT**2))*1.DO*K2*K2*K2
DQQQ =-(1.DO/(3.DO*DSAT**2))*1.D0*K3*K3*K3
DZZZ=-(1.DO/(3.DO*DSAT**2))*1.D0*K4*K4*K4
```

C C

```
        T(I,4)=YYY(I,4)
    16 CONTINUE
    17 CONTINUE
C
C CHECK THE EIGENVALUES
C
    WRITE(21,2003) TC*U0/L,WI(1),WI(2),WI(3),WI(4),
    &
                                WR (1),WR (2),WR(3),WR (4)
    INORM=1
    IF (INORM.NE.O) CALL NORMAL(T)
    INVERT TRANSFORMATION MATRIX
    DO 2 I=1,4
        DO 3 J=1,4
            TINV (I,J) =0.0
            TSAVE (I,J) =T(I,J)
        CONTINUE
    CONTINUE
    CALL DLUD(4,4,TSAVE,4,TLUD,IVLUD)
    DO 4 I=1,4
        IF (IVLUD(I).EQ.O) STOF 3003
    CONTINUE
    CALL DILU(4,4,TLUD,IVLUD,SVLUD)
    DO }8\mathrm{ I=1,4
        DO 9 J=1,4
            TINV (I,J) =TLUD (I,J)
        CONTINUE
            CONTINUE
C CHECK Inv(T)*A*T
IMULT=1
IF (IMULT.EQ.1) CALL MULT(TINV,ASAVE,T,A2)
IF (IMULT.EQ.0) STOP
P=A2 (3,3)
Q=A2 (4,4)
WRITE (21,*) P,Q
DEFINITION OF Nij
N11=TINV (1,1)
N21=TINV (2,1)
N31=TINV (3,1)
N41=TINV (4,1)
N12=TINV (1,2)
N22=TINV (2,2)
N32=TINV (3,2)
```

N42=TINV (4,2)
N13=TINV $(1,3)$
N23=TINV $(2,3)$
N33=TINV $(3,3)$
N43 $=\operatorname{TINV}(4,3)$
N14 =TINV (1,4)
N24=TINV $(2,4)$
N34 $=$ TINV $(3,4)$
N44=TINV (4,4)



DEFINITION OF Mij
$\mathrm{M} 11=\mathrm{T}(1,1)$
$\mathrm{M} 21=\mathrm{T}(2,1)$
M31 $=T(3,1)$
$\mathrm{M} 41=\mathrm{T}(4,1)$
$\mathrm{M} 12=\mathrm{T}(1,2)$
$\mathrm{M} 22=\mathrm{T}(2,2)$
M32 $=T(3,2)$
$\mathrm{M} 42=\mathrm{T}(4,2)$
M13 $=T(1,3)$
$\mathrm{M} 23=\mathrm{T}(2,3)$
M33 $=T(3,3)$
$\mathrm{M} 43=\mathrm{T}(4,3)$
$\mathrm{M} 14=\mathrm{T}(1,4)$
$\mathrm{M} 24=\mathrm{T}(2,4)$
M34 $=\mathrm{T}(3,4)$
$\mathrm{M} 44=\mathrm{T}(4,4)$
DEFINITION OF Lij
$\mathrm{L} 25=\mathrm{C} 11$ *M31*M31+C12*M21*M31
L26 $=2$ *C11*M31*M32 +C12* (M21*M32 +M22*M31)
$\mathrm{L} 27=\mathrm{C} 11 * \mathrm{M} 32$ *M32 + C12 *M22 *M32
L35 $=$ C 22 *M31*M31 +321 *M21*M31
L36 $=2$ * C 22 *M31*M32 +C21* (M21*M32+M22*M31)
$\mathrm{L} 37=\mathrm{C} 22$ *M32*M32 +C 21 *M22*M32

L21= DTTW*M11*M11*M21 + DTWW*M11*M21*M21 + DTTQ*M11*M11*M31 + DTQQ*M11*M31*M31 + DTTZ*M11*M11*M41 + DTZZ*M11*M41*M41 + DWWQ*M21*M21*M31 + DWQQ*M21*M31*M31 + DWWZ*M21*M21*M41 + DWZZ*M21*M41*M41 + DQQZ*M31*M31*M41 + DQZZ*M31*M41*M41 + DTWQ*M11*M21*M31 + DTWZ*M11*M21*M41 + DTQZ*M11*M41*M31 + DWQZ*M21*M41*M31

DTTT*M11*M11*M11 + DWWW*M21*M21*M21 + DQQQ*M31*M31*M31 + DZZZ*M41*M41*M41

TTW=M11*M11*M22+2.0*M11*M12*M21
TWW=M12*M21*M21+2.0*M11*M21*M22
TTQ $=$ M11*M11*M32+2.0*M11*M12*M31
TQQ $=\mathrm{M} 12 * \mathrm{M} 31 * \mathrm{M} 31+2.0 * \mathrm{M} 11 * \mathrm{M} 31 * \mathrm{M} 32$
TTZ=M11*M11*M42+2.0*M11*M12*M41
TZZ=M41*M41*M12+2.0*M11*M41*M42
WWQ=M21*M21*M32+2.0*M31*M21*M22
WQQ=M22*M31*M31+2.0*M31*M32*M21
WWZ=M21*M21*M42+2.0*M41*M21*M22
WZZ $=$ M22*M41*M41 $+2.0 * M 41 * M 42 * M 21$
$\mathrm{QQZ}=\mathrm{M} 31 * \mathrm{M} 31 * \mathrm{M} 42+2.0 * \mathrm{M} 41 * \mathrm{M} 31 * \mathrm{M} 32$
QZZ $=$ M32*M41*M41+2.0*M41*M42*M31
TWQ $=$ M11*M21*M32 +M31* (M11*M22 +M12*M21)
TWZ $=$ M11*M21*M42+M41*(M11*M22+M12*M21)
TQZ $=$ M11*M41*M32+M31* (M11*M42+M12*M41)
WQZ $=$ M21*M41*M32+M31* (M21*M42+M22*M41)
TTT=3.0*M11*M11*M12
WWW=3.0*M21*M21*M22
$\mathrm{QQQ}=3.0 * \mathrm{M} 31 * \mathrm{M} 31 * \mathrm{M} 32$
ZZZ=3.0*M41*M41*M42
$L 22=D T T W * T T W+D T W W * T W W+D T T Q * T T Q+D T Q Q * T Q Q+$ $D T T Z * T T Z+D T Z Z * T Z Z+D W W Q * W W Q+D W Q Q * W Q Q+$ $D W W Z * W W Z+D W Z Z * W Z Z+D Q Q Z * Q Q Z+D Q Z Z * Q Z Z+$ DTWQ*TWQ+DTWZ*TWZ+DTQZ*TQZ+DWQZ*WQZ + DTTTT*TTT+DWWW*WWW+DQQQ*QQQ+DZZZ*ZZZ

TTW $=$ M12*M12*M21 + 2.0*M11*M12*M22
TWW=M11*M22*M22+2.0*M12*M21*M22
TTQ $=$ M12*M12*M31 $+2.0 *$ M11 *M12*M32
TQQ=M11*M32*M32+2.0*M12*M31*M32
TTZ $=$ M12*M12*M41+2.0*M11*M12*M42
TZZ=M11*M42*M42+2.0*M12*M41*M42
WWQ $=\mathrm{M} 22$ *M22*M31+2.0*M21*M22*M32
$W Q Q=M 21 * M 32 * M 32+2.0 * M 22 * M 31 * M 32$
WWZ $=\mathrm{M} 22$ *M22*M41+2.0*M21*M22*M42
WZZ=M21*M42*M42+2.0*M22*M41*M42 QQZ=M32*M32*M41+2.0*M31*M32*M42
QZZ=M31*M42*M42+2.0*M32*M41*M42
TWQ=M12*M22*M31+M32* (M11*M22+M12*M21)
TWZ $=$ M12*M22*M41 + M42* (M11*M22 +M12 *M21)
TQZ $=$ M12*M42*M31 + M32* (M11*M42+M12*M41)
WQZ $=$ M 22 *M42*M31 + M32* (M21*M42 + M22*M41)
TTT $=3.0 * \mathrm{M11*M12*M12}$
WWW $=3.0 * \mathrm{M} 21 * \mathrm{M} 22 * \mathrm{M} 22$
QQQ $=3.0 *$ M31 $*$ M32*M32
ZZZ=3.0*M41*M42*M42

L23 =DTTW*TTW+DTWW*TWW+DTTQ*TTQ+DTQQ*TQQ+ $D T T Z * T T Z+D T Z Z * T Z Z+D W W Q * W W Q+D W Q Q * W Q Q+$ DWWZ*WWZ+DWZZ*WZZ+DQQZ*QQZ+DQZZ*QZZ + DTWQ*TWQ+DTWZ*TWZ+DTQZ*TQZ+DWQZ*WQZ + DTTTT*TTT+DWWW*WWW+DQQQ*QQQ+DZZZ*ZZZ

| L24 $=$ | DTTW*M12*M12*M22 | + DTWW*M12*M22*M22 |
| :---: | :---: | :---: |
|  | DTTQ*M12*M12*M32 | + DTQQ*M12*M32*M32 |
|  | DTTZ*M12*M12*M42 | + DTZZ*M12*M42*M42 |
|  | DWWQ*M22*M22*M32 | + DWQQ*M22*M32*M32 |
|  | DWWZ*M22*M22*M42 | + DWZZ*M22*M42*M42 |
|  | DQQZ*M32*M32*M42 | + DQZZ*M32*M42*M42 |
|  | DTWQ*M12*M22*M32 | + DTWZ*M12*M22*M42 |
|  | DTQZ*M12*M42*M32 | + DWQZ*M22*M42*M32 |
|  | DTTT*M12*M12*M12 | + DWWW*M22*M22*M22 |
|  | DQQQ*M32*M32*M32 | + DZZZ*M42*M42*M42 |

$\mathrm{L} 31=\mathrm{L} 21$
$\mathrm{L} 32=\mathrm{L} 22$
L33 $=\mathrm{L} 23$
L34 $=\mathrm{L} 24$
DEFINITION OF ALFA(I) AND BETA(I)
D1 $=$ N $32 * L 25+$ N33*L35
D2 $=$ N32*L26 + N33*L36
D3 $=$ N $32 * L 27+$ N33*L37

ALFA2 $=(\mathrm{D} 2-\mathrm{D} 21 * \mathrm{D} 1 / \mathrm{D} 11-\mathrm{D} 23 * D 3 / D 33) /$ (D22-D21*D12/D11-D23*D32/D33)
ALFA1 $=(\mathrm{D} 1-\mathrm{D} 12 * A L F A 2) / D 11$
ALFA3 = (D3-D32*ALFA2) /D33
D1 $=$ N42*L25 + N43*L35
D2 $=$ N42*L26 + N43*L36
D3 $=$ N42*L27 + N43*L37
D11 $=-$ Q
D12=0MEGAO
D21 $=-2$ *OMEGAO

```
D22=-0
D23=2*OMEGAO
D32 = - OMEGAO
D33 = - Q
```

```
BETA2 = (D2 -D21*D1/D11-D23*D3/D33)/
    (D22-D21*D12/D11-D23*D32/D33)
BETA1=(D1 -D12*BETA2)/D11
BETA3 = (D3 -D32*BETA2)/D33
```

```
L21A=2*C11*(ALFA1*M31*M33+BETA1*M31*M34) +
```

L21A=2*C11*(ALFA1*M31*M33+BETA1*M31*M34) +
C12* (ALFA1*(M21*M33 +M23*M31) +
C12* (ALFA1*(M21*M33 +M23*M31) +
BETA1*(M21*M34+M24*M31))
BETA1*(M21*M34+M24*M31))
L22A=2*C11*(ALFA1*M32*M33+BETA1*M32*M34)
+2*C11*(ALPA3*M31*M33 +BETA3*M31*M34)
+C12*(ALFA1*(M22*M3 3+M32*M23)
+BETA1*(M22*M34 +M24*M32))
+C12*(ALFA3*(M21*M33+M23*M31)
+BETA3*(M21*M34+M24*M31))
L23A=2*C11*(ALFA2*M31*M33+BETA2*M31*M34)
+2*C11*(ALFA3*M32*M33 + BETA3*M32*M34)
+C12*(ALFA2*(M21*M33+M23*M31)
+BETA2*(M21*M34+M24*M31))
+C12*(ALFA3*(M22*M33+M23*M32)
+BETA3* (M22*M34 +M24*M32))
L24A=2*C11* (ALFA2*M32*M33 +BETA2 *M32*M34)
+C12*(ALFA2*(M22*M33+M23*M32)
+BETA2* (M2 2 *M34 +M24*M32))
L31A=2*C22*(ALFA1*M31*M33 +BETA1*M31*M34)
+C21*(ALFA1* (M21*M33+M23*M31)
+BETA1*(M21*M34 +M24*M31))
L32A=2*C22* (ALFA1 *M32*M33 +BETA1 *M32*M34)
+2*C22*(ALFA3*M31*M33 +BETA3*M31*M34)
+C21*(ALFA1*(M22*M33+M32*M23)
+BETA1* (M22*M34 +M24*M32))
+C21*(ALFA3*(M21*M33+M23*M31)
+BETA3*(M21*M34 +M24*M31))

```
L33A=2*C22*(ALFA2*M31*M33 +BETA2*M31*M34)
    +2*C22*(ALFA3*M32*M33+BETA3*M32*M34)
    +C21*(ALFA2*(M21*M33+M23*M31)
    +BETA2*(M21*M34 +M24*M31))
    +C21*(ALFA3*(M22*M3 3+M23*M32)
    +BETA3*(M22*M34 +M24*M32))
```

L34A 2 2*C22* (ALFA2*M32*M33 + BETA2*M32*M34) +C21* (ALFA2* (M22*M33 +M23*M32) +BETA2* (M22*M34 +M24*M32))
$\mathrm{L} 21=\mathrm{L} 21 \mathrm{~A}+\mathrm{L} 21 * \mathrm{~B} 1 * \mathrm{U} * \mathrm{U}-\mathrm{Al} 3 * \mathrm{ZGB} * \mathrm{M} 11 * * 3 / 6 . \mathrm{DO}$ $L 22=L 22 A+L 22 * B 1 * U * U-A 13 * Z G B * M 12 * M 11 * * 2 / 2 . D 0$ L2 3 = L2 3A + L2 3*B1*U*U-A13*ZGB*M11*M12**2/2.D0
$\mathrm{L} 24=\mathrm{L} 24 \mathrm{~A}+\mathrm{L} 24 * \mathrm{~B} 1 * \mathrm{U} * \mathrm{U}-\mathrm{A} 13 * \mathrm{ZGB} * \mathrm{M} 12 * * 3 / 6 . \mathrm{D} 0$
$\mathrm{L} 31=\amalg 31 A+\amalg 31 * B 2 * U * U-A 23 * Z G B * M 11 * * 3 / 6 . D 0$
$\mathrm{L} 32=\mathrm{L} 32 \mathrm{~A}+\mathrm{L} 32 * \mathrm{~B} 2 * \mathrm{U} * \mathrm{U}-\mathrm{A} 23 * \mathrm{ZGB} * \mathrm{M} 12 * \mathrm{M} 11 * * 2 / 2 . D 0$
L3 $3=\mathrm{L} 33 A+\mathrm{L} 33 * B 2 * U * U-A 23 * Z G B * M 11 * M 12 * * 2 / 2 . D 0$
L34 $=\mathrm{L} 34 \mathrm{~A}+\mathrm{L} 34 * \mathrm{~B} 2 * \mathrm{U} * \mathrm{U}-\mathrm{A} 23 * \mathrm{ZGB} * \mathrm{M} 12 * * 3 / 6 . D 0$
L41 = - 0.5*M11*M11* (M21-U*M11/3.0)
L42 $=-$ M11* (M12*M21 + 0.5*M11*M22-0.5*U*M11*M12)
L43 = - M12* (M11*M22 + 0.5*M12*M21-0.5*U*M11*M12)
L4 $4=-0.5 * \mathrm{M} 12 * \mathrm{M} 12 *(\mathrm{M} 22-\mathrm{U} * \mathrm{M} 12 / 3.0)$
R11 $=\mathrm{N} 12$ * $\mathrm{L} 21+\mathrm{N} 13$ * L $31+\mathrm{N} 14$ * L 41
$\mathrm{R} 12=\mathrm{N} 12$ *L22 + N13 * L 32 +N14 * L 42
R13 $=$ N12*L23 +N13*L33 +N14*L43
R14 $=\mathrm{N} 12 * \mathrm{~L} 24+\mathrm{N} 13 * \mathrm{~L} 34+\mathrm{N} 14 * \mathrm{~L} 44$
R21 $=\mathrm{N} 22$ * L21 +N23*L31+N24*L41
R22 $=\mathrm{N} 22$ *L22 + N 23 *L $32+\mathrm{N} 24 *$ L 42
$\mathrm{R} 23=\mathrm{N} 22 * \mathrm{~L} 23+\mathrm{N} 23$ *L33 +N24*L43
R24 $=\mathrm{N} 22$ *L24 + N23*L34 +N24*L44
EVALUATE DALPHA AND DOMEGA
UINC $=0.001$
UR $=\mathrm{U}+$ UINC
UL $=\mathrm{U}$-UINC
$\mathrm{U}=\mathrm{UR}$
$A(1,1)=0.0 D 0$
$A(1,2)=0.0 D 0$
$A(1,3)=1.0 D 0$
$A(1,4)=0.0 D 0$
$A(2,1)=A 13 * Z G B+B 1 * U * U * K 1$
$A(2,2)=A 11 * U \quad+B 1 * U * U * K 2$
$A(2,3)=A 12 * U \quad+B 1 * U * U * K 3$
$A(2,4)=$
$A(3,1)=A 23 * Z G B+B 2 * U * U * K 1$
$A(3,2)=A 21 * U \quad+B 2 * U * U * K 2$

$$
A(3,3)=A 22 * U \quad+B 2 * U * U * K 3
$$

$A(3,4)=\quad B 2 * U * U * K 4$
$A(4,1)=-U$
$A(4,2)=1.0 D 0$
$A(4,3)=0.0 D 0$
$A(4,4)=0.0 D 0$

C

C

CALL RG (4, 4, A, WR, WI, 0, YYY, IV1, FV1, IERR)
CALL DSTABL (DEOS,WR,WI, FREQ)
ALPHR=DEOS
OMEGR=FREQ
U=UL
$A(1,1)=0.0 \mathrm{DO}$
$A(1,2)=0.0 D 0$
$A(1,3)=1.0 D 0$
$A(1,4)=0.0 \mathrm{DO}$
$A(2,1)=A 13 * Z G B+B 1 * U * U * K 1$
$A(2,2)=A 11 * U \quad+B 1 * U * U * K 2$
$A(2,3)=A 12 * U \quad+B 1 * U * U * K 3$
$A(2,4)=\quad B 1 * U * U * K 4$
$A(3,1)=A 23 * Z G B+B 2 * U * U * K 1$
$A(3,2)=A 21 * U \quad+B 2 * U * U * K 2$
$A(3,3)=A 22 * U \quad+B 2 * U * U * K 3$
$A(3,4)=\quad B 2 * U * U * K 4$
$A(4,1)=-U$
$A(4,2)=1$. ODO
$A(4,3)=0.0 \mathrm{DO}$
A $(4,4)=0.0 \mathrm{DO}$

CALL RG (4, 4, A, WR,WI, 0, YYY, IV1, FV1, IERR)
CALL DSTABL (DEOS,WR,WI, FREQ)
ALPHL=DEOS
OMEGL=FREQ
DALPHA = (ALPHR - ALPHL) / (UR-UL)
DOMEGA $=$ (OMEGR-OMEGL) / (UR-UL)
EVALUATION OF HOPF BIFURCATION COEFFICIENTS
COEF1 $=3.0 * R 11+\mathrm{R} 13+\mathrm{R} 22+3.0 * \mathrm{R} 24$
COEF2 $=3.0 * R 21+\mathrm{R} 23-\mathrm{R} 12-3.0 * \mathrm{R} 14$
AMU2 $=-$ COEF1 $/(8.0 *$ DALPHA $)$
BETA2 $=0.25$ *COEF1
TAU2 $=-($ COEF2-DOMEGA*COEF1/DALPHA) $/(8.0 * O M E G A 0)$
PER $=2.0 * 3.1415927 /$ OMEGAO
PER $=$ PER*U/L
WRITE $(20,2001)$ TC*U0/L, COEF1,DALPHA,
\&
1 CONTINUE

STOP

```
1001 FORMAT (' ENTER NUMBER OF DATA LINES')
1002 FORMAT (' ENTER UO, ZG, AND DSAT')
1003 FORMAT (' ENTER BOW PLANE TO STERN PLANE RATIO')
1004 FORMAT (' ENTER ZG')
2001 FORMAT (7E14.5)
2002 FORMAT (6E14.5)
2003 FORMAT (9F11.5)
3001 FORMAT (2I5)
    END
```


SUBROUTINE DSOMEG (IJK,WR,WI, OMEGA, CHECK)
IMPLICIT DOUBLE PRECISION (A-H,O-Z)
DIMENSION WR (4), WI (4)
CHECK=-1.0D+25
DO $1 I=1,4$
IF (WR (I).LT.CHECK) GO TO 1
CHECK=WR (I)
$I J=I$
1 CONTINUE
OMEGA=DABS (WI (IJ))
IF (WI (IJ).GT.O.DO) IJK=IJ
IF (WI (IJ).LT.O.DO) IJK=IJ-I
RETURN
END

SUBROUTINE DSTABL (DEOS,WR,WI, OMEGA)
IMPLICIT DOUBLE PRECISION (A-H,O-Z)
DIMENSION WR (4) ,WI (4)
DEOS $=-1.0 \mathrm{D}+20$
DO $1 \mathrm{I}=1,4$
IF (WR(I).LT.DEOS) GO TO 1
DEOS=WR (I)
$I J=I$
1 CONTINUE
OMEGA=WI (IJ)
OMEGA=DABS (OMEGA)
RETURN
END

SUBROUTINE NORMAL(T)
IMPLICIT DOUBLE PRECISION (A-H,O-Z)
DIMENSION T(4, 4), TNOR $(4,4)$
CFAC=T(1,1)**2+T(1,2)**2
IF (DABS (CFAC).LE. (1.D-10)) STOP 4001
$\operatorname{TNOR}(1,1)=1 . \operatorname{DO}$
$\operatorname{TNOR}(2,1)=(T(1,1) * T(2,1)+\mathrm{T}(2,2) * T(1,2)) / C F A C$
$\operatorname{TNOR}(3,1)=(T(1,1) * T(3,1)+T(3,2) * T(1,2)) / C F A C$
$\operatorname{TNOR}(4,1)=(T(1,1) * T(4,1)+T(4,2) * T(1,2)) / C F A C$
$\operatorname{TNOR}(1,2)=0 . D 0$
$\operatorname{TNOR}(2,2)=(T(2,2) * T(1,1)-T(2,1) * T(1,2)) / C F A C$

```
    TNOR (3,2)=(T(3,2) *T (1, 1)-T(3,1) *T(1, 2))/CFAC
    TNOR (4, 2)=(T(4,2)*T(1,1)-T(4,1)*T(1, 2))/CFAC
    DO 1 I=1,4
        DO 2 J=1,2
            T(I,J) =TNOR (I,J)
        CONTINUE
    1 CONTINUE
        RETURN
        END
```



```
    IMPLICIT DOUBLE PRECISION (A-H,O-Z)
    DIMENSION TINV (4, 4),A(4,4),T(4,4),A1 (4,4),A2 (4, 4)
    DO 1 I=1,4
        DO 2 J=1,4
                                A1(I,J)=0.DO
                A2 (I,J) =0.D0
    2 CONTINUE
    1 CONTINUE
            DO 3 I=1,4
            DO 4 J=1,4
                    DO 5 K=1,4
                    AI (I,J) =A (I,K)*T(K,J) +AI (I,J)
                    CONTINUE
        5 CONTIN
        3 CONTINUE
            DO 6 I=1,4
                DO }7\textrm{J}=1,
                    DO 8 K=1,4
                        A2(I,J)=TINV (I,K)*AI (K,J) +A2 (I,J)
                    CONTINUE
            8 CONTIN
    6 ~ C O N T I N U E ~
    DO 11 I=1,4
C WRITE (*,101) (A(I,J),J=1,4)
    11 CONTINUE
            DO 12 I=1,4
C WRITE (*,101) (T (I,J),J=1,4)
            12 CONTINUE
            DO 10 I=1,4
C WRITE (*,101) (A2(I,J),J*1,4)
            10 CONTINUE
C WRITE (*,101) A2 (1,1)
            RETURN
    101 FORMAT (4E15.5)
            END
```



```
    MDS =(-2.4090E-03*0.5*RHO*L**3)
    MDB =(2.4090E-03*0.5*RHO*L**3)
C
    ZG =0.4DO
    ZB =0.0DO
    XG =0.0DO
    XB =0.0DO
    ZGB =ZG-ZB
    ZD=ZDS +ALPHA* ZDB
    MD=MDS +ALPHA*MDB
C
C
C
C READ(30,*) THETA
    W =0.1DO
    Q =O.ODO
    Z =O.ODO
    Z =Z*L
C
C
    DETERMINE [A] AND [B] COEFFICIENTS
    DV = (MASS - ZWDOT) * (IY-MQDOT)
    & - (MASS*XG+ZQDOT)* (MASS*XG +MWDOT)
    AIIDV = (IY-MQDOT) * ZW+(MASS*XG+ZQDOT) *MW
    A12DV = (IY -MQDOT) * (MASS+ZQ) + (MASS*XG+ZQDOT) * (MQ-MASS*XG)
    A13DV= - (MASS*XG+ZQDOT) *WEIGHT
    B1DV = (IY-MQDOT)*ZD+(MASS*XG+ZQDOT)*MD
    A21DV=(MASS - ZWDOT) *MW+(MASS*XG+MWDOT) * ZW
    A22DV = (MASS - ZWDOT) * (MQ-MASS*XG)
    &
                +(MASS*XG+MWDOT) * (MASS+ZQ)
    A23DV = - (MASS - ZWDOT) *WEIGHT
    B2DV = (MASS - ZWDOT) *MD + (MASS*XG+MWDOT) * ZD
C
    A11=A11DV/DV
    A12=A12DV/DV
    A13=A13DV/DV
    A21=A21DV/DV
    A22=A22DV/DV
    A23 =A23DV/DV
    B1 =B1DV/DV
    B2 =B2DV/DV
c
    CALCULATE THE CONTROL GAINS
POLE=1.0/TC
ALPHA3 = 4.0*POLE
ALPHA.2=6.0*POLE**2
ALPHAI=4.0*POLE**3
ALPHAO= POLE**4
```

    \(\mathrm{K} 4=\mathrm{AL} \mathrm{PHAO} /((\mathrm{B} 1 * \mathrm{~A} 21-\mathrm{B} 2 * \mathrm{~A} 11) * \mathrm{U} 0 * * 4\)
    A2M=BI*UO**2
$\mathrm{A} 3 \mathrm{M}=\mathrm{B} 2$ *U0**2
$\mathrm{A} 0 \mathrm{M}=-(\mathrm{A} 11+\mathrm{A} 22) * \mathrm{U} 0-\mathrm{ALPHA} 3$
B1M=B2*U0**2
$\mathrm{B} 2 \mathrm{M}=(\mathrm{B} 2 * \mathrm{~A} 12-\mathrm{B} 1 * \mathrm{~A} 22) * \mathrm{U} 0 * * 3$
$B 3 M=(B 1 * A 21-B 2 * A 11) * U 0 * * 3$
$\mathrm{B} 0 \mathrm{M}=(\mathrm{A} 11 * \mathrm{~A} 22-\mathrm{A} 21 * \mathrm{~A} 12) * \mathrm{UO} * * 2-\mathrm{A} 23 * \mathrm{ZGB}-\mathrm{ALPHA} 2-\mathrm{BI} * \mathrm{U} 0 * \mathrm{U} 0 * \mathrm{~K} 4$
$C 1 M=(B 2 * A 11-B 1 * A 21) * U 0 * * 3$
$\mathrm{C} 2 \mathrm{M}=(\mathrm{B} 1 * \mathrm{~A} 23-\mathrm{B} 2 * \mathrm{~A} 13) * \mathrm{ZGB} * \mathrm{UO} * * 2$
$\mathrm{COM}=(\mathrm{A} 13 * A 21-\mathrm{A} 23 * \mathrm{~A} 11) * \mathrm{ZGB} * \mathrm{U} 0+\mathrm{ALPHA} 1$
\& $\quad-(\mathrm{B} 2+\mathrm{B} 1 * \mathrm{~A} 22-\mathrm{B} 2 * \mathrm{~A} 12) * \mathrm{~K} 4 * \mathrm{U} 0 * * 3$
$\mathrm{K} 2=\mathrm{C} 1 \mathrm{M} * \mathrm{BOM} * A 3 \mathrm{M}-\mathrm{B} 1 \mathrm{M} * C 0 M * A 3 \mathrm{M}-\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 3 \mathrm{M} * A 0 M$
$\mathrm{K} 2=\mathrm{K} 2 /(\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 2 \mathrm{M} * \mathrm{~A} 3 \mathrm{M}-\mathrm{B} 1 \mathrm{M} * \mathrm{C} 2 \mathrm{M} * \mathrm{~A} 3 \mathrm{M}-\mathrm{C} 1 \mathrm{M} * \mathrm{~B} 3 \mathrm{M} * \mathrm{~A} 2 \mathrm{M})$
$\mathrm{K} 1=(\mathrm{COM}-\mathrm{C} 2 \mathrm{M} * \mathrm{~K} 2) / \mathrm{C} 1 \mathrm{M}$
$\mathrm{K} 3=(\mathrm{A} 0 \mathrm{M}-\mathrm{A} 2 \mathrm{M} \star \mathrm{K} 2) / \mathrm{A} 3 \mathrm{M}$

```
    WRITE (* * *) K1, K2, K3, K4
```

DETERMINE THE EIGENVALUES
$A(1,1)=0.0 D 0$
$A(1,2)=0.0 D 0$
$A(1,3)=1.0 D 0$
$A(1,4)=0$. ODO
$A(2,1)=A 13 * Z G B+B 1 * U * U * K 1$
$A(2,2)=A 11 * U \quad+B 1 * U * U * K 2$
$A(2,3)=A 12 * U \quad+B 1 * U * U * K 3$
$A(2,4)=\quad B 1 * U * U * K 4$
$A(3,1)=A 23 * Z G B+B 2 * U * U * K 1$
$A(3,2)=A 21 * U \quad+B 2 * U * U * K 2$
$A(3,3)=A 22 * U \quad+B 2 * U * U * K 3$
$A(3,4)=\quad B 2 * U * U * K 4$
$A(4,1)=-U$
$A(4,2)=1.0 D 0$
$A(4,3)=0.0 D 0$
$A(4,4)=0.0 D 0$
C

C
CALL DEVLRG (NA, A, LDA, EVAL)
CALL DWRCRN ('EVAL', 1,NA, EVAL, 1,0)
DEFINE THE LENGTH X AND BREADTH B TERMS FOR THE INTEGRATION

XL( 1$)=0.0000 \mathrm{DO}$
$\mathrm{XL}(2)=0.1000 \mathrm{DO}$
$\mathrm{XL}(3)=0.2000 \mathrm{DO}$
$\mathrm{XL}(4)=0.3000 \mathrm{DO}$

```
    XL( 5) = 0.4000D0
    XL( 6)= 0.5000D0
    XL( 7) = 0.6000D0
    XL( 8) = 0.7000DO
    XL( 9)= 1.0000D0
    XL(10)= 2.00000D0
    XL(11)= 3.000000
    XL(12)=4.0000D0
    XL(13)=7.7143D0
    XL(14)= 10.0000D0
    XL(15)= 15.1429D0
    XL(16)= 16.0000D0
    XL(17)= 17.0000D0
    XL(18) = 18.0000D0
    XL(19)= 19.0000D0
    XL(20)= 20.0000D0
    XL(21)=20.1000D0
    XL(22) = 20.2000D0
    XL(23)=20.3000D0
    XL(24)=20.4000D0
    XL(25)=20.4167D0
    DO 102 N = 1,25
        XL(N)=(L/20)*XL(N) - L/2
102 CONTINUE
```

C
$\mathrm{BR}(1)=0.000 \mathrm{DO}$
$\operatorname{BR}(2)=0.485 D 0$
$\mathrm{BR}(3)=0.658 \mathrm{DO}$
$\mathrm{BR}(4)=0.778 \mathrm{DO}$
$\mathrm{BR}(5)=0.871 \mathrm{DO}$
$\operatorname{BR}(6)=0.945 D 0$
$\operatorname{BR}(7)=1.010 \mathrm{DO}$
$\mathrm{BR}(8)=1.060 \mathrm{DO}$
$\operatorname{BR}(9)=1.180 \mathrm{DO}$
$\operatorname{BR}(10)=1.410 \mathrm{DO}$
$\operatorname{BR}(11)=1.570 \mathrm{DO}$
$\operatorname{BR}(12)=1.660 \mathrm{DO}$
$\operatorname{BR}(13)=1.670 \mathrm{DO}$
$\operatorname{BR}(14)=1.670 \mathrm{DO}$
$\operatorname{BR}(15)=1.670 \mathrm{DO}$
$\operatorname{BR}(16)=1.630 \mathrm{DO}$
$\mathrm{BR}(17)=1.370 \mathrm{DO}$
$\operatorname{BR}(18)=0.919 \mathrm{DO}$
$\operatorname{BR}(19)=0.448 \mathrm{DO}$
$\operatorname{BR}(20)=0.195 \mathrm{DO}$
$\operatorname{BR}(21)=0.188 \mathrm{DO}$
$\operatorname{BR}(22)=0.168 \mathrm{DO}$
$\operatorname{BR}(23)=0.132 \mathrm{DO}$
$\mathrm{BR}(24)=0.053 \mathrm{DO}$

```
    BR(25)=0.000D0
```

$$
\begin{aligned}
& \text { PISIM }=\text { TSIM/DELT } \\
& \text { ISIM }=\text { PISIM }
\end{aligned}
$$

101 CONTINUE
CALL TRAP ( $25, \mathrm{VEC1}, \mathrm{XL}, \mathrm{HEAVE}$ )
CALL TRAP ( $25, \mathrm{VEC} 2, \mathrm{XL}, \mathrm{PITCH}$ )HEAVE=0.5*RHO*CDZ*HEAVEPITCH=0.5*RHO*CDZ*PITCH
COUPLING EQUATIONS
DWDV $=($ IY -MQDOT $) *$ HEAVE $+($ MASS*XG $+Z Q D O T)$ *PITCH DQDV $=($ MASS -ZWDOT$) * P I T C H+($ MASS *XG+MWDOT) *HEAVE

    C1DV \(=(\) IY - MQDOT \() ~ * M A S S * Z G * Q * * 2 ~\)
    
                - (MASS*XG+ZQDOT) *MASS*ZG*W*Q
    
    C2DV \(=-(\) MASS \(-Z W D O T) * M A S S * Z G * W * Q\)
    
                        \(+(\) MASS \(* X G+M W D O T) * M A S S * Z G * Q * * 2\)
    \(\&\)
    C
DW=DWDV/DV
$D Q=D Q D V / D V$
C1=C1DV/DV
$\mathrm{C} 2=\mathrm{C} 2 \mathrm{DV} / \mathrm{DV}$
LIMITING CONDITIONLIMITING CONDITION
IF (ABS (THETA) .GT. PI/4) THEN WRITE(*,*) 'BUBBLE EXCEEDED +/- 45, BOAT WAS LOST' STOP

    ENDIFCONTROL EQUATIONS
    DELTA $=(\mathrm{K} 1 * T H E T A+K 2 * W+K 3 * Q+K 4 * Z)$DELTA $=$ DSAT*DTANH (DELTA/DSAT)IF (DELTA .GT. 0.4) DELTA=0.4IF (DELTA .LT. -0.4) DELTA $=-0.4$
THEDOT $=$ Q

```
        WDOT =A11*U*W+A12*U*Q+A13*ZGB*DSIN(THETA)
    &
    QDOT =A21*U*W+A22*U*Q+A23*ZGB*DSIN(THETA)
    &
    ZDOT =-U*DSIN (THETA) +W*DCOS (THETA)
    IF (I.GT.3) GO TO 150
    INITIAL FIRST ORDER INTEGRATION
THETA = THETA + DELT*THEDOT
W = W + DELT*WDOT
Q = Q + DELT*QDOT
Z = Z + DELT*ZDOT
F1 (I) = THEDOT
F2 (I) =WDOT
F3 (I) \(=\) QDOT
F4 (I) \(=\) ZDOT
F1 (4) \(=0\)
F2 (4) \(=\mathrm{A} 11 * \mathrm{U} * \mathrm{~W}+\mathrm{A} 12 * \mathrm{U} * \mathrm{Q}+\mathrm{A} 13 * \mathrm{ZGB} * \mathrm{DSIN}(\mathrm{THETA})\)
\(+B 1 * U * U * D E L T A+D W+C 1\)
F3 (4) \(=\mathrm{A} 21 * \mathrm{U} * \mathrm{~W}+\mathrm{A} 22\) * \(\mathrm{U} * \mathrm{Q}+\mathrm{A} 23 * \mathrm{ZGB} * \mathrm{DSIN}\) (THETA) \(+B 2 * U * U * D E L T A+D Q+C 2\)
F4 (4) \(=-\mathrm{U} * \mathrm{DSIN}(\mathrm{THETA})+\mathrm{W} * D C O S(T H E T A)\)
THETA \(=\) THETA \(+(\) DELT \(/ 24.0) *(55.0 * F 1(4)\)
-59.0*F1 (3) +37.0*F1(2)-9.0*F1 (1))
\(\mathrm{W}=\mathrm{W}+(\mathrm{DELT} / 24.0) *(55.0 * \mathrm{~F} 2(4)\)
\(1 \quad-59.0 * \mathrm{~F} 2(3)+37.0 * \mathrm{~F} 2(2)-9.0 * \mathrm{~F} 2(1))\)
\(\mathrm{Q} \quad=\mathrm{Q}+(\mathrm{DELT} / 24.0) *(55.0 * F 3\) (4) -59.0*F3 (3) +37.0*F3 (2) -9.0*F3 (1))
\(Z \quad=Z+(D E L T / 24.0) *(55.0 * F 4(4)\)
1
ADAMS-BASHFORTH INTEGRATION \(-59.0 * F 4(3)+37.0 * F 4(2)-9.0 * F 4(1))\)
F1 (1) \(=\mathrm{F} 1\) (2)
F1 (2) \(=\mathrm{F} 1\) (3)
F1 (3) \(=\) F1 (4)
F2 (1) \(=\) F2 (2)
F2 (2) \(=F 2\) (3)
F2 (3) =F2 (4)
F3 (1) \(=\) F3 (2)
F3 (2) \(=\) F3 (3)
F3 (3) \(=\) F3 (4)
F4 (1) \(=\) F4 (2)
F4 (2) \(=\) F4 ( 3 )
F4 (3) \(=\) F4 (4)
C
TIME=I*DELT
```

```
            J=J+1
            IF (J.NE.IPRNT) GO TO }10
            ALFA=W/U
            ALFA=(-DATAN (ALFA))*180/PI
            WRITE (20,991) TIME*UO/L,Z/L,THETA*180/PI,
            &
                                    DELTA,W,Q,ALFA
            WRITE (*,991) TIME*U0/L,Z/L,THETA*180/PI,
        &
            J=0
C
    100 CONTINUE
        STOP
    991 FORMAT (2X,F8.2,6(2X,F8.4))
        END
C
    SUBROUTINE TRAP(N,A,B,OUT)
C NUMERICAL INTEGRATION ROUTINE USING THE TRAPEZOIDAL RULE
C
    IMPLICIT DOUBLE PRECISION (A-H,O-Z)
    DIMENSION A(1),B(1)
    N1=N-1
    OUT=0.0
    DO 1 I=1,N1
    OUT1=0.5*(A(I)+A(I+I))*(B(I+1)-B(I))
    OUT =OUT+OUT1
1 CONTINUE
    REIURN
    END
```


## APPEADIX D - ROOT LOCUS PROGRAM

```
% THIS MATLAB PROGRAM FINDS THE ROOT LOCUS FOR A GIVEN UO, Zg,
% Alpha, AND Tc.
```

$\mathrm{W}=1556.2363$; $\mathrm{B} 1=1556.2363$;
Iy=561.32;
$\mathrm{g}=32.2$;
$\mathrm{m}=\mathrm{W} / \mathrm{g}$;
rho=1.94;
$\mathrm{L}=13.9792$;
$\mathrm{xg}=0 ; 2 \mathrm{~g}=.4$;
$\mathrm{zgb}=.4$;
nd1=.5*rho*L^2;
nd2 $=.5$ *rho*L^3;
nd3 $=.5$ * rho ${ }^{\text {L }}{ }^{\wedge} 4$;
nd4 $=.5$ *rho*L^5;
$\mathrm{e}=\operatorname{zeros}(100,4)$;
$X=\operatorname{zeros}(4,1)$;
$Y=\operatorname{zeros}(4,1) ;$
$Z=z e r o s(4,1) ;$

Alpha=0;
Zqdnd $=-6.33 e-4 ;$ Zwdnd $=-1.4529 e-2 ;$ Zqnd $=7.545 e-3 ;$ Zwnd $=-1.391 e-2$;
Zds=.5* (-5.603e-3); Zdb=-.5*5.603e-3; Zdltnd=(Zds+Alpha*Zdb);
Mqdnd $=-8.8 e-4 ;$ Mwdnd $=-5.61 e-4 ;$ Mqnd $=-3.702 e-3 ;$ Mwnd=1.0324e-2;
Mds = .5* ( -0.002409 ) $; \mathrm{Mdb}=.5 * 0.002409$; Mdltnd= (Mds +Alpha*Mdb);
Zqd=nd3*Zqdnd; Zwd=nd2*Zwdnd; Zq=nd2*Zqnd; Zw=nd1*Zwnd;
Zdilt=nd1*Zdltnd;

Mqd $=n d 4$ *Mqdnd; $M w d=n d 3 * M w d n d ; M q=n d 3 * M q n d ; M w=n d 2 * M w n d ;$
Mdlt $=$ nd2*Mdltnd;

```
Dv=(m-Zwd) * (Iy-Mqd) - (m*xg+Zqd) *(m*xg+Mwd);
a11Dv= (Iy-Mqd)*Zw+(m*xg+Zqd) *Mw;
al2Dv=(Iy-Mqd) * (m+Zq) +(m*xg+Zqd) * (Mq-m*xg);
a13Dv=-(m*xg+Zqd)*W;
b1Dv=(Iy-Mqd)*Zdlt+(m*xg+Zqd) *Mdlt;
a21Dv= (m-Zwd)*Mw+ (m*xg+Mwd)* Zw;
a22Dv=(m-Zwd)* (Mq-m*xg) + (m*xg+Mwd})*(m+Zq)
a23Dv=-(m-Zwd)*W;
b2Dv= (m-Zwd) *Mdlt + (m*xg+Mwd) *Zdlt;
a11=a11Dv/Dv;a12=a12Dv/Dv;a13=a13Dv/Dv;
a21=a21Dv/Dv;a22=a22Dv/Dv;a23=a23Dv/Dv;
b1=b1Dv/Dv;b2=b2Dv/Dv;
```

```
    u = 9;
    for j = 1:240
    U = j/20;
    s1(j)= U;
    Fn=sqrt (UN2/(zgb*g));
    tconst=4.7512;
    pole=-1/(tconst*L/u);
    p1=[pole pole-0.01 pole-0.02 pole-0.03];
    a=[00 1 0;a13*zgb a11*u a12*u 0;a23*zgb a21*u a22*u 0; ...
    -u 1 0 0];
    aU=[0 0 1 0;a13*zgb all*U a12*U 0;a23*zgb a21*U a22*U 0;-U
```

```
b=[0;b1*u^2;b2*u^2;0];
```

b=[0;b1*u^2;b2*u^2;0];
bU=[0;b1*U^2;b2*U^2;0];
bU=[0;b1*U^2;b2*U^2;0];
K1=place (a,b,p1);
K1=place (a,b,p1);
Y = eig(aU-bU*K1);
Y = eig(aU-bU*K1);
e(j,:)= Y';
e(j,:)= Y';
end
end
for k = 1:200
for k = 1:200
U = 8.9 + k/1000;
U = 8.9 + k/1000;
s2(k) = U;
s2(k) = U;
Fn=sqrt (U^2/(zgb*g));
Fn=sqrt (U^2/(zgb*g));
tconst=4.7512;
tconst=4.7512;
pole=-1/(tconst*L/u);
pole=-1/(tconst*L/u);
p1=[pole pole-0.01 pole-0.02 pole-0.03];
p1=[pole pole-0.01 pole-0.02 pole-0.03];
a=[0 0 1 0;a13*zgb a11*u a12*u 0;a23*zgb a21*u a22*u 0;...
a=[0 0 1 0;a13*zgb a11*u a12*u 0;a23*zgb a21*u a22*u 0;...
-u 1 0 0];
-u 1 0 0];
aU=[0 0 1 0;a13*zgb al1*U a12*U 0;...
aU=[0 0 1 0;a13*zgb al1*U a12*U 0;...
a23*zgb a21*U a22*U 0;-U 1 0 0];
a23*zgb a21*U a22*U 0;-U 1 0 0];
b=[0;b1*u^2;b2*u^2;0];
b=[0;b1*u^2;b2*u^2;0];
bU=[0;b1*U^2;b2*U^2;0];
bU=[0;b1*U^2;b2*U^2;0];
Kl=place (a,b,pl);
Kl=place (a,b,pl);
Z = eig(aU-bU*KI);

```
Z = eig(aU-bU*KI);
```

1001 ;

$$
f(k,:)=Z^{\prime} ;
$$

end
axis([-. 4 . 1 - -8 . 8])
plot(real (e(1,1)),imag(e(1,1)),'x',... real (e(1,2)), imag (e(1,2)),'x',... $\operatorname{real}(e(1,3)), \operatorname{imag}(e(1,3)), ' x \prime, \ldots$ real (e(1,4)), imag(e(1,4)),'x')
hold
plot (real (e(:, 1)), imag (e(:,1)),'.',... real (e(:, 2)), imag (e(:, 2)),'.',... real (e(:, 3)), imag (e(:, 3)),'.',... real (e(:,4)), imag(e(:,4)),'.')
plot (real (f(:, 1)), imag(f(:,1)),'.',... real(f(:, 2)),imag(f(:, 2)),'.',... real (f(:, 3$)$ ), imag(f(:, 3)),'.',...
real (f(: 4 )), imag (f(:,4)),'.')
grid, title('EIGENVALUES FOR SPEEDS 1 TO 100 FPS') fritle('DETAILED LOW SPEED ROOT LOCUS PLOT') pause
hold

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